

COMMERCIAL CAR JOURNAL

with which is combined Operation & Maintenance

Reg. U. S. Pat. Off.
Entered as second-class matter at the Post Office at
Philadelphia, Pa., under Act of March 3, 1879



Vol. XLVII Philadelphia, April, 1934 No. 2

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Published Monthly by

CHILTON COMPANY

Chestnut and 56th Streets, Philadelphia, U. S. A.

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pect 9137
Portland, Ore.—72 Fifth St.

SUBSCRIPTION RATES: United States and United States Pos-
sessions and all countries in the Postal Union—\$1.00 per year. Canada and
Foreign—\$2.00 per year. Single copies 25 cents.

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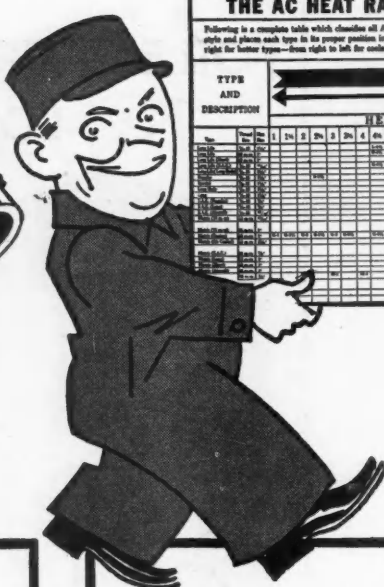
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	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16
AC 16																
AC 15																
AC 14																
AC 13																
AC 12																
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AC 10																
AC 9																
AC 8																
AC 7																
AC 6																
AC 5																
AC 4																
AC 3																
AC 2																
AC 1																



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Is the New Deal Being Messed Up?

SINCE about the beginning of this year it has been increasingly apparent that the general public was getting uncertain about restoring prosperity by promoting scarcity. Experienced and thoughtful business men had had misgivings about this paradoxical program from its beginning. But, afflicted by their own depression troubles, and in the face of the ballyhoo support of NRA, they had got little hearing even when occasionally they had the temerity to voice their misgivings. Unthinking people had been swept along on the tide of booster propaganda. They were delighted with the idea of a national leadership daringly doing things, even if they were in doubt just what things it was doing. So for a time small attention was given to the experimental character of measures or to the failure to set up any definite goal at which these measures were aimed.

More recently searching questions have been demanding answers. Have the recovery activities produced results proportionate to either their cost or to the hopefulness with which their initiation was greeted? Two inquiries as to the recovery program are lately being pressed:

First, will it work?

Second, do its authors conceive it as an end in itself, or merely as the beginning of a sweeping project of social reconstruction?

THE first question is being answered with less and less confidence even by those who most wish the answer to be affirmative. The second is asked with more and more misgiving by a constantly increasing number of people.

Not since 1896 has there been such interest in economic problems as there is today. People are reading more, thinking more, trying harder to put together the results of their reading, thinking and experience. And there is another possible parallel to 1896. A polling in August of that year would probably have swept the country for the program that in November was overwhelmingly beaten. Will the November of 1934 bring another testimony

to the moderating influence of second thought?

THERE has been much discussion of whether the blue prints of our recovery plan were borrowed more from Italy or from Russia; and the answer has commonly been hesitant, because of doubt what the plan aims at. Being pretty sure that neither Britain nor Canada has tried anything savoring of either Fascism or Communism, people are comparing their recovery accomplishments with our own—rather to the disparagement of our program. Britain and Canada have clung to the old ideals—tightening the belt, balancing the budget, insisting that economic law has not yet been repealed, keeping the Government out of business, and pay-as-you-go. On the face of returns to date, they have apparently done quite well. Canada seems to have rather bettered our record in overcoming unemployment, while the British exchequer the other day reported a comfortable surplus in contrast to our vast deficit.

SMALL wonder, then, that the American community is beginning to inquire a little about the efficacy of our more spectacular proceedings. We have poured out billions to prime the pump, and yet there are Thomases who doubt whether anything is coming out of the spout except a small portion of the priming water. Still more wonder whether the supply of priming water will hold out.

Out of these doubts some things seem to be crystallizing in the public thought. There is less ready acceptance of Government in the role of a beneficent Providence able to ladle out magically accumulated funds for the deficits of business.

THERE is increasing reluctance to assume that politicians and political appointees are more competent to direct business than are the men who have given their lives to building it.

Price-fixing is being examined in the light of a hundred generations of experience, with the resultant conclusion that it works no better now than under the Roman emperors or the early Chinese dynasties.

Question marks are being written opposite proposals to feed people with wheat that other people are paid not to raise, or to clothe them with cotton that the planters are hired to plow under.

It begins to be recognized that instead of restricting production our need is to encourage it and then get its fruits consumed through wider distribution of buying power in better wages and more employment.

There has been a healthy protest against spreading employment by reducing working hours. The argument that consuming power has never yet had a chance to prove what it can do for a truly prosperous community, is getting a serious hearing.

THE original idea of the industrial codes was to provide more employment and better wages, and do away with child labor. These things would have brought higher prices, and then profits would take care of themselves. Unfortunately some people got the idea that the N.I.R.A. had repealed anti-trust laws, and undertook to chisel out monopolies for themselves. Some of them seriously believed that was what the recovery program wanted them to do; more of them simply saw the opportunity, and reached for it. Both groups were wrong. The anti-trust laws have not been repealed, and, plainly, are not going to be. Monopolies, whether established through the machinations of private business or under grants of Government privilege, are altogether bad; and the worst of them are those erected under color of Governmental approval. It makes no difference whether that approval is given through a royal charter or an NRA code.

THE sooner we get back to accepting the simple fundamentals to which the codes were originally dedicated—more employment, better wages, better distribution of buying and consuming power—the sooner will we be back in the way of prosperity. And there are encouraging signs that with sober second thought the country is getting its feet into the safer paths.



Temporary National Code Authority for the Trucking Industry

Left to right: Front Row—James E. Murphy (Contract Carrier), St. Paul Park, Minn.; J. Rowland Bibbins (National Recovery Administration); Ted V. Rodgers (Contract Carrier), Scranton, Pa., Chairman; Roy B. Thompson (Manager, Federated Truck Associations of California), San Francisco, Vice-Chairman; W. A. Gordon (Furniture Warehouseman), Omaha, Neb. Back row—Edward F. Loomis (Staff Secretary); Edward S. Brashears (Staff General Council); Fred O. Nelson (City Drayman), New York City; Percy F. Arnold (Contract and Common Carrier and City Drayman), Providence, R. I.; William E. Humphreys (Staff Treasurer); J. H. McAlphin (Not-for-Hire), Eldorado, Ark.; H. D. Horton (Common and Contract Carrier), Charlotte, N. C.; Frank C. Schmidt (Common Carrier), Toledo, Ohio. Charles P. Clarke (National Recovery Administration), is not in the picture. Neither are staff men C. F. Jackson and J. V. Lawrence.

The Trucking Code is Rolling Down the Home-Stretch

THE Trucking Code—the trucking industry's experiment in self-regulation which will cost millions upon millions of dollars—is, at last, coming down the home-stretch. All that remains is for NRA to approve the industry's plans for registering the 1,600,000 motor vehicles which will come under the code.

Approval will depend upon the outcome of the hearing into the reasonableness of assessments proposed by the National Code Authority for the trucking industry. This hearing was ordered by General Johnson, after the National Code Authority had approved an assessment of \$3.00 per vehicle against all vehicles in the "for-hire" classification of the code, and 90 cents per vehicle against vehicles in the "not for hire" class. These assessments would furnish a budget somewhere around three million of dollars for administration of the code.

IT is difficult to see how the hearing into assessments can be anything but a formality. The writer has seen the stack of work-sheets containing an analysis of every conceivable item of expense which would be caused by the code. This assessment budget has been in preparation since last December. Budgets submitted by more than 20

states were taken into consideration in arriving at final results. Then different revisions were made of the budget to make certain that the work caused by the code would be done at the lowest possible cost.

IT isn't possible to list here all the items of expense figured into the 90-cent assessment. The cost of registration and of handling reports was analyzed from every angle. Some of the items covered by the assessment are:

Cost of registration insignia. (Insignia sample is reproduced in these pages.)

Freight and expressage on insignia to the distributing point in each state.

Printing and distributing the registration and report forms to the various states. (There is really only one large form which combines registration and operating data. And when the writer says large, he means a huge headache. He recalls that the tentative form he saw a few weeks ago consisted of six pages, each about half again as large as the page you are now reading, and much more closely printed. The original form was a simple one. Then came demands from NRA to procure operat-

ing data which would give the government its first accurate picture of the trucking industry. There is no doubt about the value and the necessity for these operating facts, but neither is there any doubt that a lot of operators will do considerable head-scratching, fuming, fussing and cussing before they come up with the answers.)

Cost of editing registration and report forms for tabulation.

Coding and tabulating of registration and report forms.

Filing and storage of registration and report forms.

Cost of maintaining registration offices located at points convenient to those who must register.

Postage on registration insignia from the state headquarters to the registrant.

Addressing registration insignia; their inventory, storage, insurance, etc.

Printing and shipping to state headquarters of registration certificates.

The filling in of registration certificates.

Addressing and mailing of registration certificates to each registrant.

THE assessment of \$3.00 covers all of the items above and additional costs of code administration which would be applicable only to "for-hire"

How the Trucking Code Stands

1. National Code Authority approved.
2. Effective date of code extended to March 30, 1934.
3. Break-down of nation into 12 regional areas approved.
4. Many State Code Authorities approved.
5. By-laws and regulations for State Code Authorities approved.
6. Hearing ordered on assessment fees of 90 cents per vehicle for not-for-hire vehicles and \$3 per vehicle for for-hire vehicles.
7. Registration insignia approved.
8. Registration and report form approved.

vehicles. Among these additional items are:

Securing compliance with the general administration of the Industrial Relations provisions of the code.

Providing facilities for the filing of rates and tariffs, and the administration of this article of the code.

Development of a cost formula on which rates and tariffs will be founded. Studies to determine the feasibility and desirability of a shorter working day in the industry.

Studies to develop standards of safety and health among employees of the industry.

Studies on which to base a report to the Administration as to the advisability of evidence of responsibility or insurance for public liability, property damage or cargo, which report is required in the code.

Development of trade agreements tending to liberalize labor conditions and to develop trade practices within natural or territorial groups of the industry.

Development of a system of uniform accounting and reporting to be recommended to the Administrator as required in the code.

IT can safely be said that the representatives of the trucking industry who worked up the assessments arrived at their figures only after the most painstaking effort. They had many conferences with various governmental departments, with other Code Authorities that have had experience in code administrative work, with private institutions and recognized experts who have had years of experience in gathering,



These are reproductions of the code registration plates that well-dressed trucks will wear

tabulating and handling data of the sort that will be required under the trucking code. Every effort was made to get the assessments down to the lowest possible point.

It is this very fact that makes it reasonable to suppose that the assessment hearing is not likely to develop objections which would cause the budget to be thrown overboard at this stage of code work. An objection to be valid would have to represent almost as much analytical and research work as has gone into the making of the budget.

SO code effectiveness actually is now in the home-stretch. Last month at this time it was still in the back-stretch but during the last 30 days Washington has seen a great deal of expert jockeying. As a result decisions have been made which were looked for months ago.

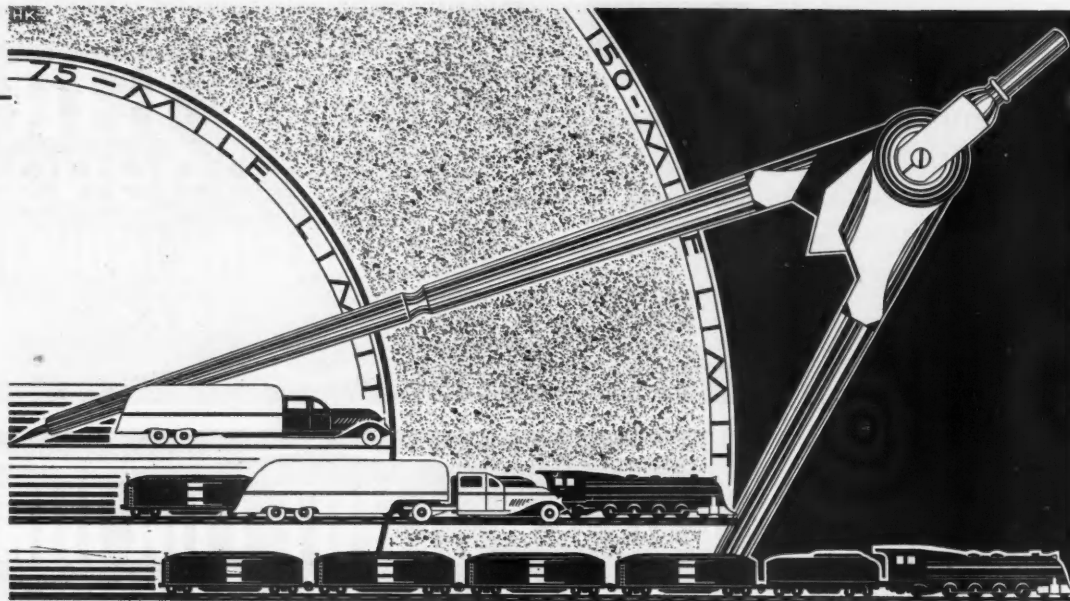
The signing of the trucking code by President Roosevelt last Feb. 10, was just the beginning of further troubles for those faithful representatives of the industry who have been making personal sacrifices to promote its welfare. Weeks went by and so did Feb. 25, the effective date of the code. But there was nothing that could be made effective because the machinery for setting the code in motion was dismantled, assembled, and dismantled time and again in undergoing the inspection of the various boards of NRA.

MEMBERS of the temporary National Code Authority were given official approval. These men shouldered the responsibility of trying to get the decisions by NRA which would give the industry what it had been looking for since December. Their patience was singularly rewarded when one of General Johnson's right-hand men raised the very original question of why the trucking code should include private carriers. Raising this question after the code had been approved by everybody from President Roosevelt down was a rather weighty straw but it didn't succeed in breaking the spirit of the National Code Authority.

Its members went to the mat and victory eventually was theirs. The NRA mogul who had raised the question had also (at least so the story goes) raised Cain with the various NRA "menials" who had worked on the trucking code. He wanted the code amended to stress the fact that if private carriers didn't want to register under the trucking code they could protest to NRA and it would be subject to hearing.

THE matter was settled just about the time the National Code Authority prevailed upon NRA to extend the effective date of the code 30 days—to March 25, that is. Both matters were covered in an executive order issued by General Johnson.

(TURN TO PAGE 15, PLEASE)



L.C.L. Traffic Report Aims To Put 150-Mile Limit on Trucks

FOR the first time in transportation history a searching analysis of the whole field of freight transported in lots of less-than-railroad-carload lots was minutely examined by a corps of transportation specialists under the guidance and direction of John R. Turney, an outstanding and forward-looking traffic executive, formerly vice-president of the St. Louis-Southwestern Railway, and, since the organization of the Federal Coordinator of Transportation Staff, the director of the Transportation Service Section.

The work of this organization has resulted in a proposed plan for the transportation of merchandise freight which is of primary importance to every motor truck operator because it proposes changes, which if adopted by the railroads and other carriers, would revolutionize the transportation of freight throughout the United States.

The merchandise report of the Section of Transportation Service has vital bearing upon the status and future development of the motor transportation industry. The nationwide study is based upon complete answers to detailed inquiries received from over 35,000 ship-

pers and consignees of merchandise freight; all principal railroads, express companies, freight forwarders and water carriers engaged in domestic freight transportation, all truck operators operating fleets of 10 or more vehicles in intercity transportation as common carriers, contract carriers or private vehicle operators.

The studies indicate that a startling amount of intercity merchandise freight is being transported by motor vehicles. The total tonnage of merchandise freight, that is, freight in less-than-railroad-carload lots, transported by all domestic land transportation carriers amounted, in 1932, the year studied, to about 52,000,000 tons, distributed among transport agencies as follows:

Railroad L.C.L. Service	15,234,000
Freight forwarders	1,902,000
Express companies	2,826,000
Highway common carriers ...	3,549,000
Highway contract carriers ...	2,459,000
Highway private haulers	26,252,000
Total (Tons)	52,222,000

These figures are, of necessity, to be considered as approximations, for returns were not made by all transportation agencies, but they are significant

in indicating the importance of highway transportation in the intercity movement of freight traffic. Highway vehicles transported over 32,000,000 tons of merchandise in 1932, as compared to less than 20,000,000 tons transported by railroad L.C.L., forwarder and railway express services.

The popularity of motor transportation is attested to by the fact that in 1932 the tonnage of merchandise freight transported by highway was more than twice the tonnage transported by rail L.C.L. service and that over half of the highway traffic moved for distances over 50 miles. This estimate, arrived at by dividing the total tonnage reported by shippers and receivers by two to allow for complete duplication, shows 14,763,000 tons transported by trucks for distances under 50 miles, 12,434,000 tons transported for distances of 50 to 250 miles, and 3,085,000 tons transported by highway over 250 miles.

The responses of 35,468 shippers, representing 112,142,038 tons of merchandise freight shipped or received, stated their reasons for using motor trucks as compared with other carriers'

service as shown below. In most cases several reasons were assigned.

	% of Shippers	% of Tonnage
Simpler classification or rates	16	25
Cheaper packing	21	27
Store door collection	51	54
Store door delivery	65	67
Cheaper total service	53	67
Faster service	65	73
More flexible or convenient service	43	61
Late acceptance of shipments	21	26
Less damage to or loss of freight	11	14
Personal friendship or interest	3	3

With respect to speed, studies indicated the average door-to-door speeds to be over 20 miles per hour, while present rail L.C.L. schedules were found rarely to exceed 20 miles per hour, with door-to-door services still slower owing to terminal detention. The motor vehicle was found to be "generally superior in speed to rail express service for distances under 150 miles, but generally inferior for distances in excess of 350 miles."

The almost universal demand of modern industry for complete merchandise service, including store-door collection and delivery service, was found to be responsible in large measure for the growth of highway transportation. As shown, about 65 per cent of the shippers representing 67 per cent of the freight shipped, use motor transportation because of delivery service, while 51 per cent of the shippers speaking for 54 per cent of the freight transported stated that they use motor freight service because of the availability of store-door collection service.

The cheaper total cost of motor freight transportation is a drawing card of great importance. Over 18,000 shippers, 53 per cent of all who responded, representing 67 per cent of the merchandise freight, reported that they used motor freight transportation partly because of the lower total cost of motor transportation. The unfavorable position of railroad merchandise rates as compared with motor freight transportation charges was found to be due partly to:

1. Complex classifications of railroad freight.
2. Complicated tariffs of rates and charges.
3. Rigorous packing requirements.
4. Incomplete transportation service.
5. Lack of rate parity.
6. The rigidity and structure of the railroad system.

IN this article are summarized those portions of the Federal Coordinator of Transportation's report on less-than-carload lot freight traffic which are of vital interest to the motor truck industry.

The analysis presented here is entirely impartial and, in this respect, resembles the voluminous report of the Coordinator's Section of Transportation Service.

Printed copies of the report will be available shortly. You can make arrangements to get a copy by communicating with the Federal Coordinator of Transportation, Washington, D. C.



JOHN R. TURNEY, Director of Section of Transportation Service

As a result of these factors, the level of railroad L.C.L. charges is higher than the highway transportation charges alternatively available to shippers and consignees of merchandise freight, with the result that the level of rail L.C.L. charges, as indicated by the third-class railroad freight rates, is higher than the highway freight rates for all hauls in the East under 280 miles, in the South for all hauls under 700 miles, in western trunk-line territory (the northwestern part of the United States) for all hauls under 500 miles, and in the Southwest for all hauls under 950 miles. Motor freight rates were found to be lower than the railway express rates for all distances.

Coupled with this fact is the further finding that "the present box car is inferior either to the motor vehicle or to rail express equipment in its ability to protect the lading against shocks, oscillation or vibration."

The effect of competition has been unfavorable financially to all forms of inland transportation, although not so damaging to motor carriers as a rule as to others, the studies revealed. The motor freight carrier has a simpler operating problem due to the smaller unit of equipment and the smaller number of routes covered.

Motor truck common carriers operating 10 or more vehicles were found to have earned in 1932 a return of slightly less than 2 per cent upon the capital investment after paying operating expenses and taxes, an operating ratio including taxes of about 99 per cent. Highway contract carriers were found to have an operating ratio including taxes of about 88 per cent. The average cost of operation of all high-

way freight transportation vehicles included in the survey, operated by common and contract and by private haulers, was found to be 21 cents per mile. The operating ratio including taxes of freight forwarders was found to be about 92 per cent. The average operation ratio of railroad less-than-carload merchandise freight transportation was found to be about 125 per cent, and the railway express ratio was found to be about 113 per cent.

In other words, railroad and express merchandise traffic failed to bear the full proportion of the total operating expenses and taxes in 1932 by \$6.04 per ton handled by express and \$4.13 per ton handled by rail L.C.L. service. The handling of rail merchandise traffic resulted in an out-of-pocket cost of about \$11.70 per ton, and the express service had an out-of-pocket cost of \$35.89 per ton. The services failed to bear their full share of total operating expenses by about \$80,000,000. This was found to be due largely to "the expense incurred in maintaining redundant rail organization, facilities and services, resulting in unnecessary duplications of station facilities, billing, platform handling, concentration and distribution, transfer enroute, and in a multiplicity of services and schedules."

The report recommends the integration of railroad express and forwarding merchandise traffic to eliminate preventable wastes in line and terminal costs in order to reduce the potential cost of rail merchandise transportation below the 1932 costs of motor truck transportation and to make the utilization of the proposed integrated rail L.C.L. service more economical than highway transportation for merchan-

dise traffic moving more than 100 miles.

"Assuming that the practices causing preventable wastes in the handling of merchandise transportation are eliminated, then highway transportation for distances over 150 miles would not be economically justified with motor vehicles operated at the average cost of their 1932 operations, and likewise concentration or distribution of merchandise in rail L.C.L. service for distances under 75 miles" would not be economically justified, even after the potential economies pointed out have been realized. Motor truck transportation for distances between 100 and 150 miles generally would be justified under these conditions only when the superiority in speed or flexibility of the vehicle is worth the additional cost of providing the service.

• Comparative Costs

The comparative costs of performing common carrier and private hauler motor truck service and potential railroad L.C.L. service costs per 100 pounds of freight for representative distances between 50 and 300 miles used in arriving at this conclusion are shown below:

Distance	Common Carrier Truck	Private Truck Hauler	Potential Rail L.C.L. Service
50 miles	21c.	11c.	27c.
75 miles	26c.	16c.	28c.
100 miles	30c.	21c.	30c.
200 miles	47c.	42c.	36c.
300 miles	63c.	63c.	42c.

The effect of this would be to establish three mileage or distance zones of "comparative utility," one within 75 miles, within which the motor carrier possesses a clear comparative advantage; another beyond 150 miles, in which the proposed rail L.C.L. service would have the comparative advantage, and a twilight zone between 75 and 150 miles, in which the type of service selected would depend upon the exigencies of particular cases, a sort of truck-rail twilight zone.

The changes in the organization and methods of the railroad, railway express and freight forwarding carriers proposed to bring about this economic balance and the elimination of wasteful and often destructive competition of rail and highway trucking services are radical, in that they go, as the word "radical" implies, to the root of the evil. The plan proposes:

1. The consolidation of the railroad L.C.L., the railway express and the freight forwarding merchandise services into two competing merchandise agencies or pools, each operating on a national scale throughout the United States, and each of comparable traffic and financial strength. These agencies are proposed to be owned by the rail-

roads over which they operate, and managed by independent managements in which the public is represented. The contracts under which the agencies operate should encourage direct and economical routing of the railroad lines, *but should protect* the revenues of each participating railroad carrier.

2. That these proposed merchandise agencies collect and deliver the freight at the patrons' doors, and transport it in "shock-proof" rail equipment at over-all or door-to-door speeds of at least 20 miles per hour. No recommendation is made in the report with respect to the ownership and operation of the cartage vehicles, except to the extent to be discussed later. The speeds suggested for merchandise rail service generally may be summarized as follows:

Service between points separated by:	Time of Service Delivery
350 miles or less.....	Over-night
900 miles or less.....	Second morning
1500 miles or less.....	Third morning
2100 miles or less.....	Fourth morning
2700 miles or less.....	Fifth morning
3300 miles or less.....	Sixth morning

Faster merchandise service at speeds of 35 miles per hour or over is recommended to be afforded on limited passenger trains between points between which these fast trains are operated so as to provide second-morning delivery within 1500 miles, third-morning delivery within 2500 miles, and fourth-morning delivery within 3400 miles.

3. That the present railroad L.C.L. merchandise classification, rate and packing regulations be completely abandoned and in their places substituted a simplified classification system, a simplified rate structure based upon the geographical location of the points of origin and destination, and simplified packing requirements. The report proposes that all articles of merchandise be placed in a first or standard class, with certain specifically named *heavy articles* grouped in a lower second class; and certain other enumerated *very heavy articles* grouped in a still-lower third class. This would replace with one standard and two exceptional classes, the present multitudinous classes and exceptions used in connection with railroad L.C.L. freight, the express classification and the forwarding companies' classification arrangements which are based upon the railroad classification system.

The rate structure proposed is a standard scale adjusted to meet the requirements of transportation conditions in different sections of the country, based upon full operating costs plus taxes plus a fair profit, *designed so as to make unprofitable the handling of merchandise by highway for dis-*

tances in excess of 150 miles, and the handling of merchandise by rail for distances under 75 miles. The distance between the points is recommended to be determined for rate-making purposes by measuring in degrees of latitude and longitude instead of in miles. The rates between two points are determined by locating the points of origin and destination and computing the difference of their latitudes and longitudes.

4. That rail and highway transportation be coordinated by contract, joint rate or lease arrangements or by ownership of motor trucks by the rail merchandise agencies, so that merchandise traffic may be concentrated at and delivered from a limited number of key concentration stations by motor truck and moved between these concentration stations by rail in car lots. This would tend to place the railroad and the motor truck in a supplemental rather than a competitive relationship.

• The Exchange Proposal

The Section of Transportation Service of the F.C.O.T. estimates that there are over 3,000,000 tons of merchandise moving by highway for distances over 250 miles, and over 12,000,000 tons for distances between 50 and 250 miles. Approximately 10,000,000 tons of less-than-carload freight are being transported by rail for distances under 50 miles, most of which could be hauled more efficiently by highway. "Coordination as a supplement to modernized service and tariffs (rates) would result in the exchange of this tonnage so that there should be returned to the rails at least 10,000,000 tons of long-haul traffic now moving by highway, and there should be diverted to the highway an equivalent amount of short-haul tonnage now moving by rail."

In short, the report proposes that the railroads, express carriers and forwarders on the one hand, and motor carriers on the other, arrange to swap approximately equivalent quantities of merchandise tonnage, so that each can operate with greater efficiency and profit in the distance zones in which the rail and the motor carriers have the greater comparative advantage in efficiency, so that the shipping public is given the advantage of a system of transportation which utilizes to the full extent the economies of each type of transportation—rail and road.

The report of the Section of Transportation Service has been forwarded by the Federal Coordinator of Transportation to the regional coordinating committees of the rail carriers and to the regional labor committees. The matter is now up to the railroads. They may adopt the plan in whole or in part, or they may offer a better plan, if one can be devised.

THE TRUCKING CODE IS ROLLING DOWN THE HOME-STRETCH

(CONTINUED FROM PAGE 11)

The extension part was logical because the code, as approved, provided for the registration of all members of the industry within 30 days after the effective date, and the election of state code authorities within 60 days after the effective date by members "who have registered as required."

In granting the application for extension of the time limit, Administrator Johnson stated that it appeared such modification should be granted "because of the complexity and size of the industry." So he ordered that "all periods specified in said code within which compliance shall be required and all periods within which elections shall be held and the period within which members of the industry shall register and report are hereby extended so that the commencement of the time with reference thereto shall be computed from the date of March 30, 1934."

And even so—assuming that the assessment hearing will be perfunctory and will be followed by NRA approval of the established fees—it is doubtful if the registrations can be handled in the few remaining days of April.

THE private carrier matter was handled in the form of an amendment to Article II, Section 1-B-(1) which specifies that the code shall apply to vehicles not for hire except to the extent that such transportation is subject to any other approved code of fair competition. The following amendment was ordered added to this:

"Such vehicles when also subject to any other code shall be registered under this code, by the person or other form of enterprise controlling the operation thereof, who shall furnish reports and pay equitable assessments under the code based on the code incident to registration and reports, all as may be approved by the administrator.

"The above amendment and the approval thereof shall take effect on April 5, 1934, unless good cause to the contrary is shown to the Administrator in Room 4217, Commerce Building, before that time and the Administrator issues a subsequent order to that effect."

In a sense this amendment threw the code wide open for further wrangling and dispute. It permitted private carriers to come forward and assert their rights to exemption. Fortunately for the code there was a decided limit to the time (score this point for the National Code Authority) when such exemptions could be claimed—April 5—and the matter, moreover, did not re-

No Action on Truck Retailing Code

UP to the time of going to press no action had yet been taken on the petition filed with the National Recovery Administration to include trucks above three-quarter-ton capacity under the marketing provisions of the Code of Fair Competition for the Motor Vehicle Retailing Trade.

It is understood that the representative character of the sponsors of the petition will be investigated before a hearing date is set to act on the request.

ceive widespread publicity. To the best of the writer's knowledge no exemptions were claimed.

WITH the coming of April, code affairs began dancing to a faster tempo. Maestro Johnson started beating the time with an order simplifying the course of matters requiring decisions down the red-taped corridors of NRA. The solo adagio dances turned into waltzes and finally lively fox-trots.

Many temporary state code authorities were recommended and approved. The list is too lengthy for this space.

Then NRA approved regional breakdown of the United States made by the National Code Authority for the purpose of setting up Regional Code Authorities in the administration of the code. The regions and the state areas included in their boundaries are:

Region 1. Maine, New Hampshire, Massachusetts, Connecticut and Rhode Island.

Region 2. New York (Except New York City), New York City, New Jersey and Vermont.

Region 3. Pennsylvania, Delaware, District of Columbia, Maryland and West Virginia.

Region 4. Virginia, North Carolina, South Carolina, Georgia and Florida.

Region 5. Alabama, Kentucky, Tennessee and Mississippi.

Region 6. Ohio, Indiana, Michigan and Illinois.

Region 7. Oklahoma, Texas, Arkansas and Louisiana.

Region 8. Iowa, Nebraska, Missouri and Kansas.

Region 9. North Dakota, South Dakota, Minnesota and Wisconsin.

Region 10. Colorado, Wyoming and New Mexico.

Region 11. Montana, Idaho, Oregon, Washington and Utah.

Region 12. Arizona, Nevada and California.

AFTER this came a reassuring announcement from NRA headquarters stating that "plans for registration of 1,600,000 vehicles under the code are nearing completion," with the General quoted as saying that "The NRA code for this industry is an important step toward stabilization of the trucking industry" and that "this Administration will exert every effort to effect this code and to enforce compliance."

The next progressive step was the calling of the hearing on assessments which was discussed earlier in this article.

The latest is the approval given to the by-laws and administrative regulations for State Code Authorities.

THE by-laws define the State Code Authorities as agencies of the National Code Authority and distinctly states that the Code Authority and the American Trucking Associations, Inc., and affiliated constituents are separate entities, but that the Code Authorities may employ associations as agencies. Where the Code Authority makes use of any trade association facilities in carrying out its administrative function, it is specifically cautioned to avoid discrimination against non-members of such association.

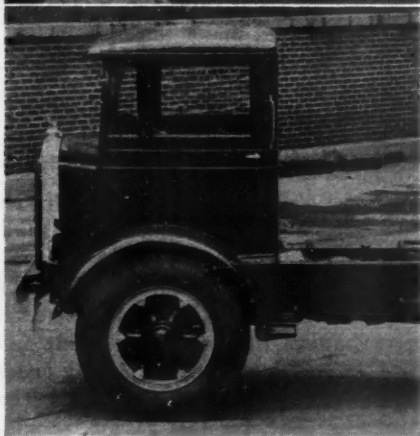
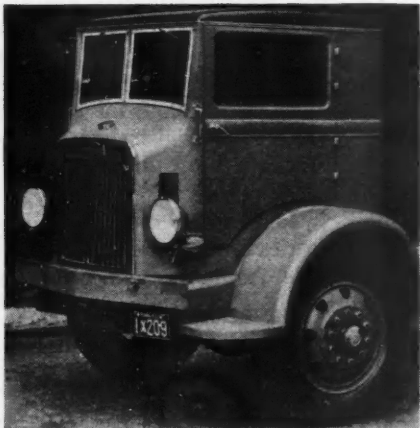
State Code Authorities, according to the regulations, have no power to interpret the code, grant exceptions under it or exceptions from it. All interpretations of the code must be made by the National Code Authority with the approval of the Administrator.

It will be the duty of the State Code Authority to collect from members of the industry the information and statistics required under the code.

The records and accounts of the State Code Authorities shall be open to inspection by the National Code Authority or its agents.

AS the code and all its administrative trappings now stand, it is probable that they represent the nearest approach to the ideal that NRA officials have evolved from their experience. It is a pattern they will very likely follow in revising many approved codes to eliminate their bad features. The trucking code, for this reason, should, once it becomes operative, rapidly realize the goal of the industry; self-regulation, employment of 300,000 additional wage earners and stabilization of a mighty industry which is certain to contribute even more to the future economic welfare of the nation than it has in the past.

What's Back of the Come-Back of the Camel-Back?



At top—The Autocar cab is entered by means of one step in the front

Center—Two front steps lead to the cab of the camel-back

Above—Mack has two steps located on the back of the front fender

BORN of seemingly unrestrained and certainly unstandardized legislative restrictions on heavy duty road transport the country over, a new type of vehicle has made its appearance in trucking circles. I use the term "new" advisedly since to those who are of the trucking industry this vehicle resurrects the shade of a construction which was rather common many years ago, save for profound improvements in mechanical design and incomparable refinement in appearance.

From now on the truck buyer will have to select his heavy duty equipment from a group of vehicles described as "cab-over-engine," "engine-under-seat," "camel-back," "traffic," and other terms yet to be coined. The thing to bear in mind is that, essentially, all these terms are intended to describe a vehicle of a specific design, hereinafter called "close-coupled," featuring a weight distribution of one-third of the gross weight on the front axle and two-thirds on the rear.

It seems to be a commonly accepted fact that we have been rapidly approaching the point where the highly developed conventional type of motor truck was becoming hampered by legislative restrictions in many localities to the point where it was in imminent danger of being outlawed. The new "close-coupled" vehicle meets the legis-

lative requirements, at least as they are written at the present, so admirably as to make that fact sufficient economic justification for its wide acceptance.

So far as can be learned, the following three legal requirements wrote the specifications for the new construction:

1. Limitation on rear axle loading.
2. Limitation of gross weight.
3. Limitation on the overall length of single units and combinations.

From a purely engineering point of view, the matter of getting a weight distribution of 1/3-2/3 can be handled in a number of ways, each one involving some serious compromise, as will be evident from the following analysis. The various arrangements that have been investigated are:

1. Moving the rear axle back. This is objectionable because it leads to excessive wheelbase length.

2. Moving the front axle back, commonly termed the "set-back front axle construction." This has been done in existing chassis as for example on General Motors and Mack trucks with a distribution of 30-70. However, this does not reduce the wheelbase to any great extent since the rear axle must be moved back at the same time. And, to obtain a 1/3-2/3 distribution with this arrangement would entail a still further sacrifice in wheelbase and maneuverability.



The Sterling cab is entered by means of a single step in front



Mechanics may stand upright in handling Autocar major overhauls

By JOSEPH GESCHELIN

THIS article analyzes the reasons which have guided certain truck makers in bringing out the so-called camel-back design. The reasons, briefly, are to place trucks within reach of users which will enable them to overcome legal restrictions.

The article is based on a paper written by the author, who is a member of the Chilton editorial staff, and read at the April meeting of the Metropolitan Section of the S.A.E.

3. By combining a partly set-back front axle and a close-coupled cab; also by moving the cab forward so that a portion of the engine protrudes into the cab. Either construction meets the requirements of the desired weight distribution but does not produce the maximum economy in loading space and overall length.

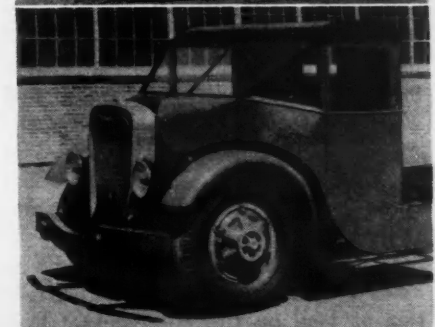
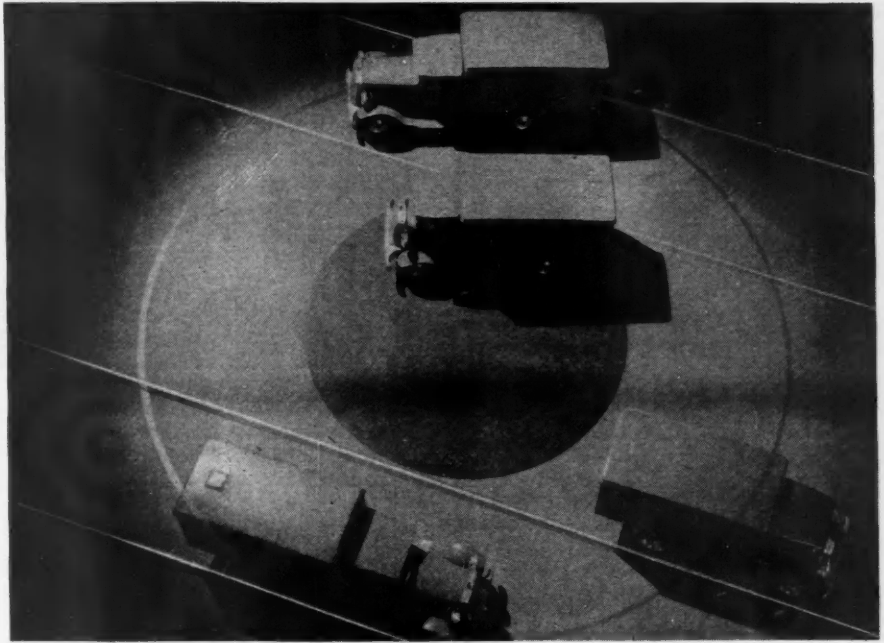
4. Moving the cab completely forward over the front axle and at the same time moving the rear axle forward, thus reducing materially the wheelbase for a given body length. This construction is the one that has been widely adopted.

To visualize how these alternate constructions actually compare in practice, I am using the example given by B. B. Bachman in a paper read before the Philadelphia Section, SAE, in December, 1933, and published in *COMMERCIAL CAR JOURNAL* for January, 1934. Referring to Fig. 1, the comparisons are made with reference to a conventional vehicle on 184 in. wheelbase with $\frac{1}{4}$ - $\frac{3}{4}$

distribution. Please refer to sketch A.

"If the attempt is made to attain 1/3-2/3 on the same construction it will be necessary to increase the wheelbase 23 inches, making a total of 207 inches, as shown in sketch B. In the second case, sketch C, by moving the front axle back 13 inches and moving the rear axle back 17 inches, a 1/3-2/3 distribution can be obtained with a wheelbase of 188 inches. In the under-the-seat model, sketch D, due to the great shortening of the overall length and the relocation of weights resulting in a change in the center of gravity, the wheelbase is reduced to 128 inches, an actual reduction of over 30 per cent.

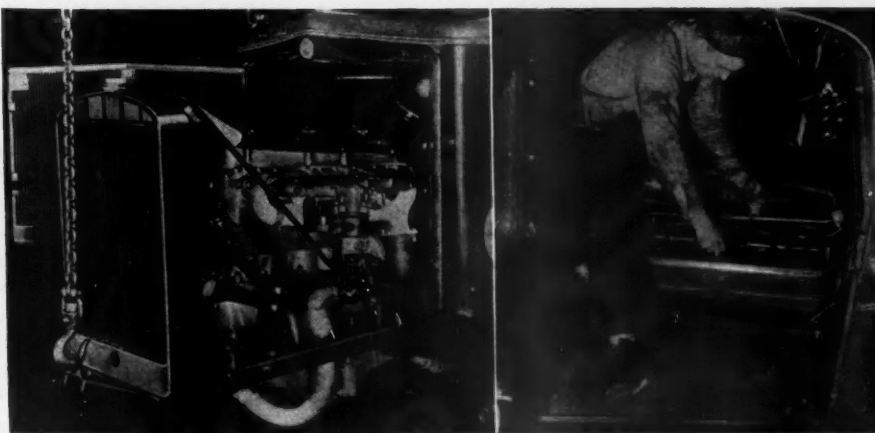
"It is also to be noted that although moving the rear axle back does not change the overall length of the vehicle, it does result in a definite increase in the turning circle. In Case 2, there is a saving of about 10½ inches in overall length and a very slight increase in turning circle, while with the under-seat



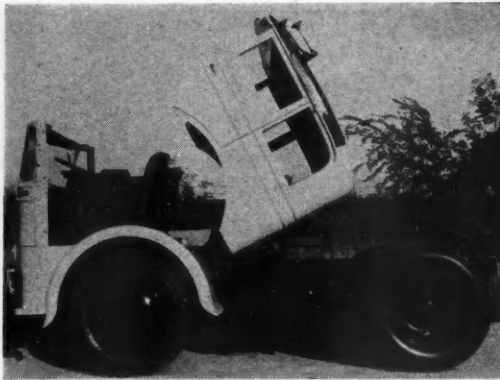
At top—The Hendrickson camel-back cab is entered by one rear step

Center—The low-hung Curtis Bill has cab moved forward over engine

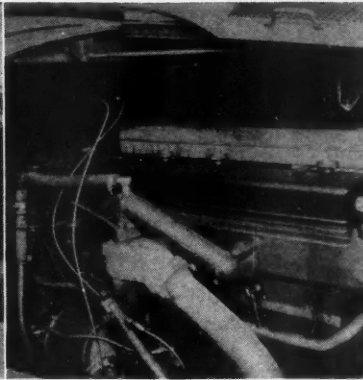
Above—The White K series has the cab moved forward over the engine



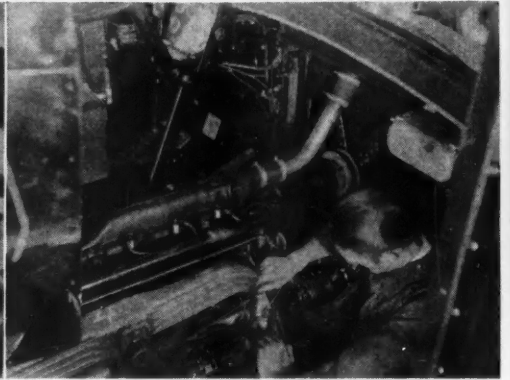
The GMC powerplant may be slid out for major overhauls. The mechanic is shown working on a minor job inside the cab compartment



The cab of the Sterling tilts to permit major overhaul accessibility



Accessibility in the White K model is conventional



In the Mack the mechanic may stand in space between engine and frame

model there is a reduction of nearly 51 inches in overall length and a reduction of 25 feet in the turning circle diameter."

Mr. Bachman did not cover the construction which is typified by the White K Series. The White construction permits the use of a longer body with the same wheelbase or the same length bodies with shorter wheelbase with a somewhat shorter overall length for the same body lengths, although the economy in overall length and turning radius does not approach the gain in "camel-back" construction.

Because the limitations are imposed on both length and weight, the "camel-back" construction has been widely adopted because it is the only one that answers both requirements.

• Unique Advantages

Taking the current "close-coupled" designs at their face value and assuming that the vehicles have been designed according to prevailing high standards we find that the new construction offers a great number of advantages apart from meeting the legislative requirements.

1. Ideal distribution of weight on rear axles resulting in a uniform distribution of load on tire equipment. This should result in considerable economy since the rear tires no longer will be overloaded.
2. Liberal gain in gross weight with the same axle loading.
3. Better maneuverability due to the short wheelbase.
4. Better ride for the driver due to location over the springs.
5. Ideal visibility unapproached by any other type of construction.
6. Increased loading space of 3 to 6 ft. depending upon make.
7. Increased loading space permits the operator to use his present body and trailer equipment and yet, in most cases, to come within the length limitations.
8. On tractor semi-trailer applications the king pin can be so located as to

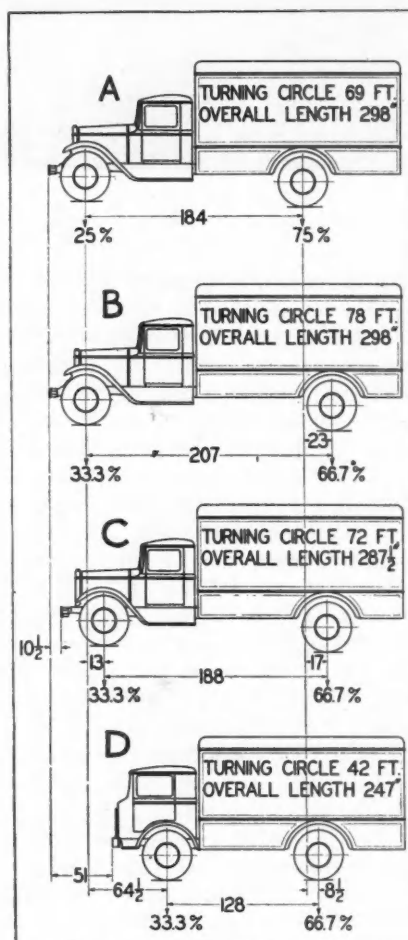


Fig. 1. Showing how different axle arrangements affect chassis design

throw a greater proportion of the load on the front axle of the tractor. This is particularly marked in inter-state vehicles involving the use of sleeper-cabs which add about two feet to the length of the tractor. With a short wheelbase job it is perfectly possible to accommodate a long cab and yet permit the mounting of the fifth wheel well in front of the rear axle.

9. Special vocational applications, as for example, milk delivery, dump trucks, coal trucks, store-door delivery involving a combination unit, inter-city hauling, etc.

It seems obvious from an analysis of designs now in production that the various truck organizations have been able to produce a vehicle which is safe, comfortable, and as easy to operate as the conventional job; also one which is not particularly difficult to service. Of course the degree in which these elements are present varies with makes.

• Analysis of Specific Designs

The element which seems to be getting the greatest emphasis is that of engine accessibility. This doubtless because of the fallacy accepted by many operators that if the engine isn't placed out in front of the cab under a hood of its own, its inaccessibility is a thing to be reckoned with.

Nevertheless, manufacturers, accepting this emphasis as evidence of buyer resistance have recognized its importance by devising a variety of arrangements, each claiming that his affords the greatest degree of accessibility. Since each designer has approached the problem conscientiously the choice of design, so far as the individual operator is concerned, is a matter of personal preference.

Thus in the Autocar design, for minor repairs and adjustments the engine is readily accessible from within the cab simply by removing the seat cushions and folding back the hinged seat back. For major overhauls requiring removal of the cylinder head, by removing the right fender splash guard, which is attached by means of five wing nuts, there is provided sufficient space to enable the mechanic to get directly at the engine without jacking up the wheels or removing the engine from the chassis.

In the General Motors Truck job the engine is exposed by the removal of a hood within the cab which is secured by latches. For major overhauls the engine is arranged to slide out either partly or completely. For this purpose the engine is mounted on a sub-frame rolling easily on rollers mounted inside of the side rails.

(TURN TO PAGE 20, PLEASE)



Mr. George T. Hook, Editor
Commercial Car Journal

Dear George: I am sending herewith an outline of a request for repair work which was made by one of our dealer service operators. It appears to be a little too good to be thrown in the waste basket. If you can make any use of it, you are welcome.
F. L. Faulkner, Automotive Department, Armour & Co.

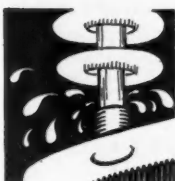
Truck No. One-O-One-Six-Eight

I. BODY

The dents are few—in fact just one—it's in the right front fender, Please ding it out to look like new, and I'll applaud the mender.

And if you have some time to spare,
A dab of paint just here and there
Will make the butchers glad to see
This ear from Armour's family,
And all the grocers near and far
Will hail it as a brand new car.

For Wilson has a new Ford Eight and Swift a Chevy Six,
So Armour needs a snappy car to show those boys some tricks.



II. RADIATOR

The radiator leaks a bit, please fix it up—and say,
Don't spill the liquid cause it's filled with Armour's G.P.A.
And don't leave grease or dirt or flux within the cooling cells,
For if you do the G.P.A. will boil and foam and smell,
And when it hits some spot that's hot
You'll swear, by gad, it's what it's not!

III. BATTERY & GENERATOR

The battery's only two months old—it cost nine and a quarter—
But check the specific gravity, and feed it distilled water.
The generator is quite new so don't disturb the stator,
Just take a piece of emery cloth and clean the commutator.

IV. CARBON & VALVES

The carbon was scraped at 23 "thou," the valves were ground then, too;
It's needed bad after 23 more, so see what you can do.

V. IGNITION

Ignition's bad, it makes me mad.
Under the hood and 'neath the dash
The sparks jump to and fro;
I look for them—just miss a crash!
What's wrong I do not know.
Why it makes the wagon jerk and buck
Till you're sure as hell of getting stuck.



VI. TIRES

The tires are new on all four wheels,
It's only a good spare we're needing;
But the rims are so pitted
When the new tires are fitted
They fail very soon at the beading.

VII. CARBURETOR

The carburetor's full of muck; it needs a good de-griming
To get more miles from gasoline and do away with priming.



VIII. PISTON RINGS

The piston rings are all worn out; the cylinder walls need honing
Before you snap new rings in place. Then Armour will do less groaning
For oil this engine eats in gobs—
Enough to run two diesel jobs.
And that will make less trouble—see?
Cause big oil bills are blamed on me!

IX. BRAKES

From the container 'neath the hood (it's in the braking system)
The oil leaks out—please make it tight, and fix the master piston.
Emergency needs tightening (though only used for parking),
A turn or two will do the trick to keep the truck from "larking."

X. TRANSMISSION & REAR END

The transmission grease is just like soap—that goes for rear end, too.
Please fill them both with "winter grease," so the gears will slide right through.

XI. REMARKS

It really is not as bad as it sounds,
(The truck, I mean, not the verse);
But those are the things that are wrong with my truck—
It could be a whole lot worse.
And whether the notes are on knicks or on knacks,
I hope they're not too galling.
What's really needed—to get to brass tacks—
IS THOROUGH OVERHAULING

Signed
W. H. Kissler



WHAT'S BACK OF THE COME-BACK OF THE CAMEL-BACK?

(CONTINUED FROM PAGE 18)

Hendrickson has a hood inside the cab large enough to permit working on valves and ignition. For further accessibility, there are hinged hood sides between the top of the frame and the bottom of the cab which in combination with the wide track axles permit adjustments on tappets and the carburetor. For major overhauls, the engine can be reached by removing the integral bumper and front cross member. The radiator guard, radiator and cross member are designed for ready removal.

In the Mack the engine enclosure is in two parts. The main housing may be raised by removing the seat and back cushions of the middle seat for access to spark plugs, distributor, generator, oil filter and valve tappets. The two side seats fold up out of the way and the floor boards lift out affording access to the rest of the engine. For major overhauls the entire power plant may be removed through the front after removing the radiator.

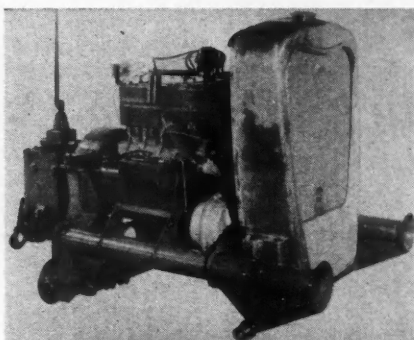
In the Sterling for more repairs and adjustments the engine is made accessible simply by the removal of the inside aluminum hood which is located directly between the two bucket seats. For major overhauls which require removal of the engine, the cab has been so arranged as to tilt back on hinge pins, as in the case of a tractor unit; or the cab can be entirely removed by taking out the hinge pins in the case of a truck unit with a body in place.

In the White K series, engine accessibility is entirely conventional, the unique feature being the fact that even the overhead valve engines have all accessories located on the right hand side where they are readily accessible in a group.

The Curtis-Bill powerplant, while not as accessible for minor repairs and adjustments as the other "close-coupled" jobs, for major overhaul accessibility has been worked out in excellent fashion. The entire front-drive unit is mounted in a sub-frame whose backbone consists of two tubular members which telescope over the tubular main frame and are fastened in place by means of large nuts over the ends. This permits the removal of the entire powerplant in a very simple operation.

• Control Functions

From the point of view of the operator, as well as that of the designing engineer, questions have been raised at various times concerning the simplest way of working out the remote control of the functions of steering, gearshifting and clutch operation. Behind these questions is the fact that with the new



The entire powerplant of the Curtis Bill slides out. Tubular side-members telescope over tubular frame

construction the driver and consequently his controls are quite unconventionally located and the linkage between the controls and the corresponding units are inclined to become quite complicated.

While this complication undoubtedly exists, none of the organizations participating in this movement seems to have had any particular difficulty in finding a practical solution. So far as I can learn, all of the jobs now in production have continued the use of manually operated steering gear, clutch, and gear shift. Sterling seems to be the only one using an air-controlled clutch among the production jobs.

So far as gear shifting is concerned, on the Mack job this has been accomplished by building the complete shifter mechanism into the transmission in the conventional manner on the unit power plant transmission but extending it to the left instead of up through the top; then by means of a simple sliding and rocking rod connection to the conventional ball-jointed gearshift lever, it is possible to secure a very effective layout.

Camel-back Conclusions

IN concluding this study, I should like to make the following points:

1. The new design is definitely born of legislative restrictions particularly (a) limitation on axle loading; (b) limitation on gross weight; (c) limitation on the overall length of single vehicles and combinations.

2. The special advantages of the new construction are (a) ideal weight distribution for tires; (b) shorter turning circle and, consequently, better maneuverability; (c) an increase in gross load with the same body length; (d) increased loading space within prescribed length limitations.

On the General Motors job, the shifting operation requires an up and down motion instead of the usual forward and back motion. They effect a direct gear-shift control without rods or intermediate connections by using a long lever and placing the fulcrum of the gearshift lever in a higher position over the transmission gear case. It is claimed that the function of shifting is as simple as on the conventional job.

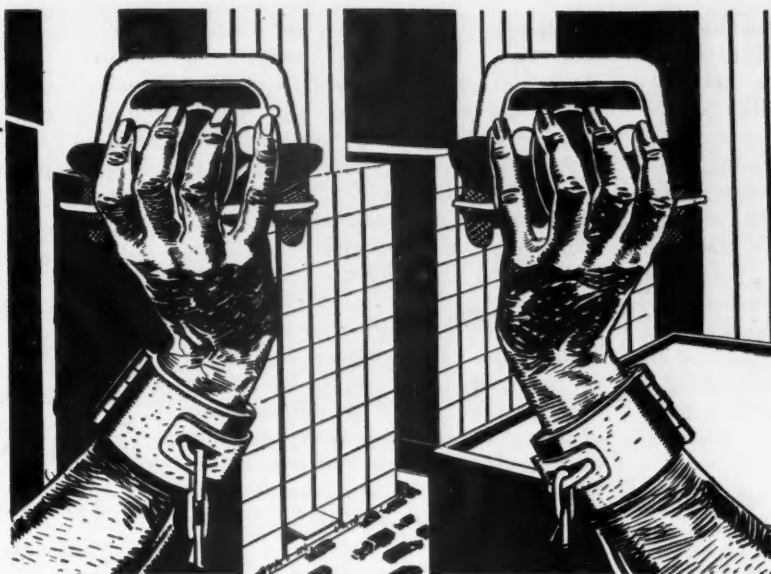
At the present writing, none of the organizations with which I have been in contact contemplates power steering. A number of prominent engineers tell me that with 1/3-2/3 gross weight distribution, manual steering seems to present no problems and, in fact, is comparable with the steering of modern buses due to the refinements incorporated in the steering gears now available. I am told further that the experimental work of several organizations indicates that manual steering, at least on motor trucks, is perfectly feasible up to a limit of about 9000 lb. weight on the front axle, and inasmuch as this represents the maximum gross weight permitted by law, these organizations feel that it is unnecessary for them to consider power steering.

However, the steering problem is of such vital importance that engineers cannot overlook the possibilities offered by some practical form of servo mechanism. In recent correspondence, S. Johnson, Jr., of the Bendix-Westinghouse Automotive Air Brake Co., says that he does not question the ability to steer manually with 9000 lb. load on the front wheels if the steering gear ratios are stepped up high enough. However, when about 10,000 lb. load on the front axle is exceeded he believes that power steering is positively required. He also feels that the time is not far off when vehicles with loads around 8000 lb. on the front wheels will use power steering because of the obvious advantage of lower steering gear ratios. Another feature of the power steering mechanism which is stressed by Mr. Johnson, is the reduction of load on the steering gear mechanism. With the Bendix-Westinghouse design it is claimed that this can be reduced to approximately one-fourth of what it normally is with manual operation.

• Unconventional Engines

It seems quite pertinent to make at least passing reference to the possibility of applying some hitherto unconventional types of engines to the new construction in order to promote greater accessibility. One of the possibilities of

(TURN TO PAGE 28, PLEASE)



Can Business Groups Jointly Own Trucks and Avoid For-Hire Laws?

By FRED. A. ELDEAN
Counsellor-at-Law

SOME merchants in a Montana city desiring to economize in their transport costs have adopted a novel truck purchase and share-expense operation system, the legality of which was recently sustained by the highest court in that State.

The essentials of the plan are as follows: Through a Merchants' Mutual Service organization the participating merchants agreed to purchase a truck upon installments. The truck (other trucks are to be added as needed) is to be used exclusively for transporting merchandise sold or purchased by the members for delivery to each other and to their respective customers. The purchase price and the expense of upkeep, repair, replacement, operation, including wages for necessary help employed in the operation, management and use of the truck, are to be paid for monthly upon the basis and ratio of the weight of merchandise carried for each mile for each member.

The seller of the truck was made transportation manager upon a salary, plus a bonus basis when a majority of the members vote such for good service.

Neither the Merchants' Mutual Service nor the seller (the transportation-manager-operator) of the truck were held to be within the motor carrier regulatory law which includes those "motor carriers operating for hire."

IF one man can own and operate his own truck, why can't two or more join together in truck ownership and operation, may be asked, without being subject to any greater regulation than that of the singly-owned trucks?

Should regulation of for-hire motor transport cause the rate-level to reach a point where private industry could more economically and with a fair degree of convenience operate private vehicles, it undoubtedly would do so. This might easily be done by a large industry. But what about the small fellow?

Suppose he follows the lead of the Montana case and together with another or others purchases a motor truck to be used by them jointly. Is "joint ownership and operation" a possible solution?

This unanswered question is raised without any attempt to pass on its practical feasibility or desirability but to stimulate thinking on its possibilities and its legal implications.

While the court decision holds the plan does not subject the parties to the control of the motor carrier act, it is to

be noted that this is an exceptional case. Many cases in which somewhat similar arrangements have been put into effect have resulted in totally different decisions.

But it is perhaps the modified features of this particular arrangement which distinguish this case from the others.

Consequently it will be well to examine the other arrangements to see wherein they failed to come within the pale of legal sanction.

In a Florida case where some merchants organized a cooperative association not for profit, known as the Merchants' Mutual Association, for the express purpose of conducting the business of hauling by motor trucks for compensation the goods, wares and merchandise of its stockholders only, the court held that while it was not a common carrier, as "by limiting its patronage to its stockholders, it successfully eliminates itself from that classification," it nevertheless is by virtue of the contracts of carriage between the corporation and the members subject to the law governing contract carriers.

The contract carrier provision embraces those not operating as common carriers who haul for compensation under contract where the service consists of continuous or recurring carriage under the same contracts.

The Florida arrangement comes the

nearest of any perhaps to the Montana case. Admittedly it is difficult to distinguish the two. In so far as there appears to be a difference at all in the arrangement, it relates to the express incorporation to carry (even though not for a profit) still for a *compensation*. In the Montana case, there is no carriage for compensation but a direct contribution to expenses.

If one man can own and operate his own truck, why can't two or more join together in truck ownership and operation, may be asked, without being subject to any greater regulation than that of the singly-owned truck?

• What About Small Fellows?

Should regulation of for-hire motor transport cause the rate-level to reach a point where private industry could more economically and with a fair degree of convenience operate private vehicles, it undoubtedly would do so. This might easily be done by a large industry. But what about the small fellow?

Suppose he follows the lead of the Montana case and together with another or others purchases a motor truck to be used by them jointly. Is "joint ownership and operation" a possible solution?

This unanswered question is raised without any attempt to pass on its practical feasibility or desirability but to stimulate thinking on its possibilities and its legal implications.

A Texas case which, while it does not involve a business association, is of interest as indicating how an apparently simple business arrangement may be sufficient to throw one into a "for-hire" class.

Armour & Co., to supplement its own trucks, engaged a truck under the following conditions: The owner of the truck was to supply it for Armour & Co.'s exclusive use for a certain period upon a stated weekly compensation. The truck owner was to keep the truck in good condition and to substitute another if the first became unserviceable. He was to pay all expenses of operation including the driver's salary. The driver, however, was to be under Armour & Co.'s exclusive control. He was also to hold Armour & Co. harmless against any claim of the driver under the Workmen's Compensation law. He was also responsible for the prompt remittance of any collections made by the driver.

The truck owner was held to be a contract carrier upon the view that he was operating the truck upon the highway for compensation.

This result seems subject to considerable question. However, it indicates the necessity of not injecting elements into the arrangement which might result in its being construed to be a "for-hire" carrier operation.

Suppose, instead of paying a stated sum as a "cover-all expense," the driver's wages and compensation insurance (and other necessary insurance—see *COMMERCIAL CAR JOURNAL*, February, 1934, "Who is Liable—Shipper or Trucker" article page 17) had been provided by Armour & Co. and he had in fact become their employee; and further that the truck owner was to be paid on a mileage basis for the use of his truck. The truck owner in that situation, getting a compensation for the use of his truck, is after all in effect in the same position as the conditional seller of the truck. Surely it would require a considerable strain on the imagination to conclude such an arrangement would make the truck-owner a "for-hire" operator.

Recently the Pennsylvania Public Service Commission had before it the perplexing problem of the status of the truckmen who hauled for a dairymen farmers' cooperative organization.

This association had approximately 17,000 members. Its stated purposes were the improvement of marketing conditions by securing advantages in the sale of milk and the provision of trucking facilities for transportation of milk. Any person could become a member by buying one share of stock for \$2.50 and one share at the same price for each ten cows in excess of the first ten.

Three-party agreements were executed between the farmer, the association and the trucker. The trucker agrees to collect the milk daily and each farmer agrees to consign his milk by the named trucker. The milk receiving depot remits its check for the milk to the farmers less transportation charges. The check to cover transportation is sent to the association office. After deducting six-tenths of one per cent for its services the association remits the balance to the truckers.

• Heed Common Carriers

The Commission held that the truckers were common carriers. As reasons for this conclusion the Commission points to these arguments: There was evidence that one trucker actually solicited business of some non-members; that the association itself solicited new business; that the truckers had acquired many new shippers' business and "if they had not actively solicited new business, have at least held themselves out as willing to accept hauling for new producers."

Further, it pointed out that the transportation of milk was a substantial business of itself and not incidental to some other line of activity engaged in by them and this makes rendering of this service a public calling.

The Commission complains that if service to such a large group were ex-

empted, its regulation might be impaired. The courts have frequently stated they cannot legislate. Assuming the operations were not within the regulatory law and if the predicted unhappy condition resulted, the remedy would be with the legislature. However, the Commission may have anticipated that the farmers' political strength would prevent any legislative enactment. It will be interesting to see what happens upon appeal in this case or what may result in the legislature should the farm group present a bill taking their trucking operations of this character out of the common carrier category entirely.

• Farmers Tried This

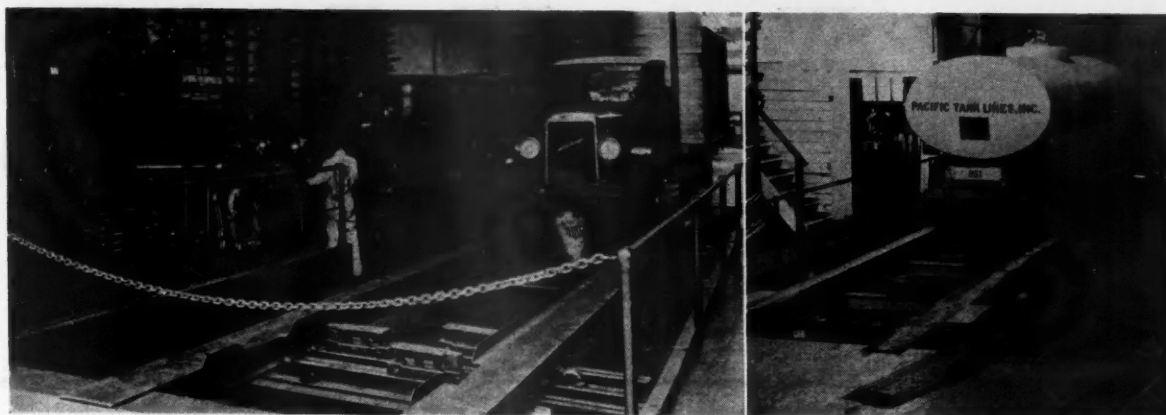
A plan whereby a Farmers' Exchange received a bill of sale for livestock at the time the truckman employed by the Exchange loaded the stock on the truck and under which the price to be paid the farmer was computed upon the basis of the price received at the stockyards less 60 cents per 100 pounds for transportation which was divided 50 cents to the truckman and 10 cents to the association was held by the Ohio appellate court a mere device to evade the law and the trucker was a common carrier.

An early Colorado case held a trucker hauling for an association which included 90 per cent of all shippers in the territory a common carrier on the ground that the public served is so large as to be the public.

In a farmers' cooperative case in Maryland, the court reached the result that the cooperative organization, while not a common carrier, was nevertheless subject to its statutory provisions governing the operation of public carriers.

There have been at least two cases in which associations of theatre operators have been organized to transport their films, one an Ohio case and the other an Iowa case, in each of which membership in the association was open to any film exhibitor who would purchase stock. It was pointed out in the Iowa case that there were no offices, no meetings, no dues; in fact, there were lacking, in that case at least, any of the elements which might possibly bring an association out of the category of being a mere device for the evasion of common carrier regulations.

From the foregoing it will be seen that it is exceedingly difficult for a business association to conduct trucking operations without becoming subject to "for-hire" regulations. Undoubtedly, there will arise cases involving associations having a somewhat different setup from that involved in the cases here described. It will be interesting to note these developments and also the possible expansion of the "joint truck ownership and operation" idea.



This wheel alignment set-up in a West Coast service establishment is typical of the installations which Bear Mfg. Co. is promoting to take care of heavy-duty trucks and buses

• THE EAR-TO-THE-GROUND DEPARTMENT •

To Spare Tire Changers

SOMETHING decidedly new in the way of spare tire carriers for trucks will soon make its appearance. This carrier moves rearward on rollers, tilts downward and the spare tire slides to the ground after its anchorage is removed. The sliding carrier is locked in place under normal conditions. Every idea connected with it aims to make the changing of flat tires an almost effortless matter so far as the spare is concerned. If you are interested we'll tell the maker.

A New 1½-Ton FWD

FWD is coming out with a new 1½-ton, four-wheel-drive model with an 84-hp. engine. It will be capable of 50 m.p.h. fully loaded. The list price of \$2,400 will be the lowest in the company's history. In appearance it will be a distinctive addition to the FWD line.

A Revolutionary Jack

The Vickers Mfg. Co. promises an early announcement of a new hydraulic jack which, Mr. Vickers informs us, represents a departure from all the old traditions of jack manufacture. It may be termed "revolutionary," he says. It is of the hand or tool-box type and will come in 3, 5, 8, 12, 16 and 20-ton capacities.

Names May Be Mentioned

Back in October we told you a large oil company was planning to market a new break-in oil within 30 days. Marketing plans, however, were not completed until last month. It's called Essolube Break-in Oil, and the large oil company is the Standard Oil Co. of N. J. It will be marketed exclusively through motor car dealers and repair shops.

Corbitt Plans New Types

The Corbitt Co. expects to bring out shortly some new types of vehicles. Chief Engineer Rawlins has promised to keep us posted.

You Can Save on Oil—

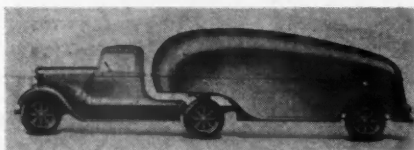
An oil reclaimer, designed for the smaller operator of trucks, has been made available by The Hilliard Corp. It will refine 360 gal. of dirty oil per month and, if run to capacity, will save more than \$1,000 per year. The total cost of reclaiming ranges from 6 cents to 12 cents per gallon. If you'll tell us the number of vehicles you operate, the number of gallons drained per month, the number of gallons of new oil bought, the cost of new oil per gallon, and the cost of power per kilowatt hour in your city, the company will tell you just how much you can save.

Front-Drive Job Coming

A new front-drive truck is on its way. The manufacturer recently obtained a patent on a front-drive axle, and has promised us full technical details.

Reclaim as You Ride

Are you interested in an engine accessory which, the manufacturer claims, is not a mere oil filter, but "a real oil reclaimer, embodying the same principles" as a large commercial reclaimer put out by the same maker? Oil pressure is the driving force, and the exhaust heat is used to evaporate water and gasoline dilution. The dilution is fed right back into the manifold. The device is designed to do away with oil changing. If you want details, we'll get them for you.



What the well-dressed tank trailer of the future may look like is suggested in this design developed by Fruehauf. Major tank companies have approved it

Pancakes and Camel-Backs

Wherever fleet operators congregate the conversation eventually gets around to camel-back truck models. And then someone always begins speculating about the advantages of the pancake-type of engine in such a design. We expect to have a look at a truck chassis equipped with a pancake engine before the May issue goes to press. We hope to have something to report.

Bu-Gas vs. Butane

E. J. McClanahan, manager of the sales development department of the Standard Oil Co. of California, wishes us to make it clear that the word Bu-Gas, used in the January articles on butane, is a trademark adopted and registered by his company to distinguish its liquefied petroleum gases from those of the same nature marketed by others.

Still Just a Plan

Last month we intimated that a large passenger-car maker was planning to bring out a truck line. Our latest feed-box dope is that the plans apparently have been shelved. There remains, however, a possibility that the plans will be revived and the low-priced truck line marketed through an existing truck set-up.

Can 2 Frenchmen Be Wrong?

How long will it be before American manufacturers take a leaf out of the French book and make available to truck operators warning signals with distinct safety advantages? One of these French devices has a photoelectric cell on the rear of the truck body. When the lights of the car behind are turned on they operate the cell and sound a warning signal in the driver's cab. Another device, brought out by a radio maker, works by means of a microphone system. In both cases the truck driver can sit snugly in a modern enclosed, insulated cab and yet know instantly when asking for the "courtesy of the road."—G. T. H.

"We Cut Idle Truck Time 75% and Saved \$2,000 a Year Per Truck"

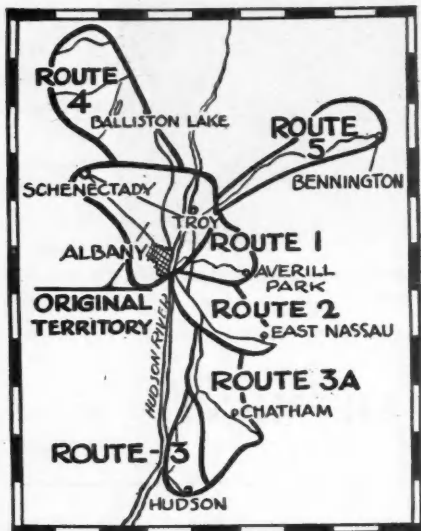


FIG. 1
Map of Truck Routes

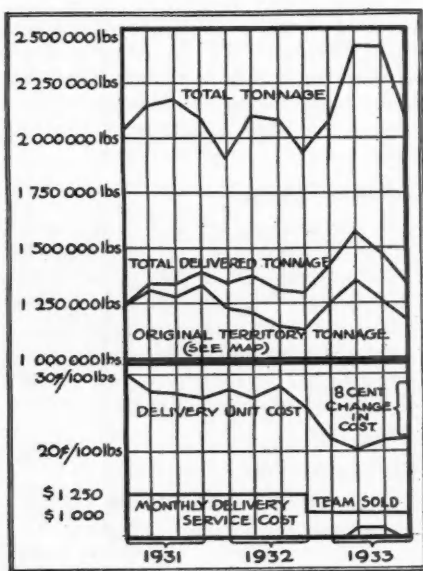


FIG. 2
Delivery Tonnage and Cost Record

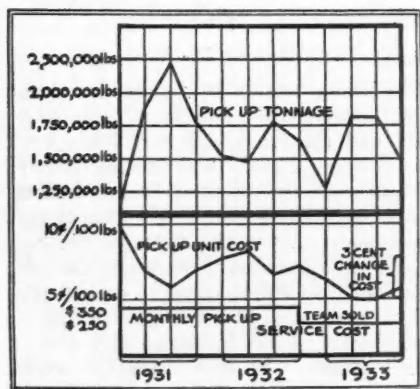


FIG. 3
Pick-up Tonnage and Cost Record

APRIL, 1934

By WALTER S. McEWAN, JR., FLEET SUPERVISOR
Oppenheim & McEwan Co., Inc., Wholesale Grocer, Albany, N. Y.

Who figures that if his trucks lose an hour a day
his company loses the profit on \$500 business

I AM ashamed to admit it, but our three trucks are now saving us \$6000 a year. By reducing our fleet and at the same time extending our delivery territory, we have actually cut our unit costs in half. In the course of saving \$2000 a truck, we reduced our idle time 75 per cent, cutting it from 29 per cent to 8 per cent.

Oppenheim & McEwan Co., Inc., wholesale grocer of Albany, N. Y., serves a territory about 150 miles in diameter, in which it "normally" does about a million dollars' business. Twelve salesmen cover this territory on schedules which call for coverage each week or every other week. The firm employs about 30 more people in the administration and operation of the business. Besides the general wholesale business, the firm operates a separate coffee roasting plant and also holds the local franchise for the Arrowhead Stores, a voluntary chain organization.

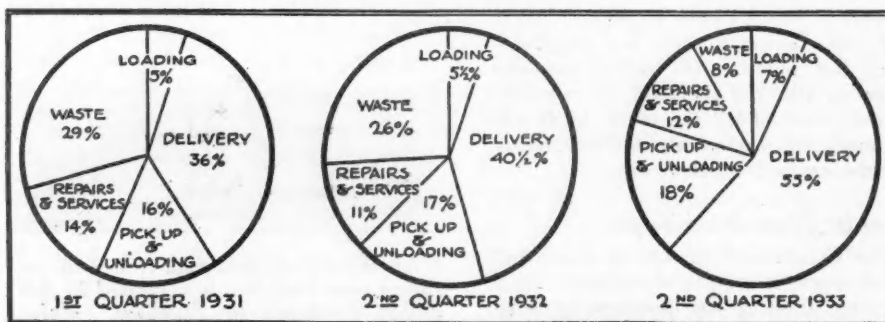
In 1931 we were operating four truck units, one team and three practically identical automotive units of about 6 tons' capacity. Besides doing all our incoming hauling, these trucks handled

all our deliveries in the original territory, as shown on the Map, Fig. 1. These deliveries accounted for 60 per cent of our outgoing tonnage. Outside truckers carried 35 per cent more, while customer calls and rail shipments took the remaining 5 per cent.

In late 1930 we moved from very inadequate quarters to our present plant and then worked out a record system to cover our entire operation. Our records were compiled from drivers' cards, shippers' reports, etc., to give us a complete picture of our entire operation.

The basis of our truck records is the hour. As originally 75 per cent of our truck overhead was fixed, we favored time rather than mileage. Furthermore, with wide differences in mileage per hour on our different types of work, and our need of exact costs for each individual type, a system based upon distance would have been very complicated. We determined the exact amount it cost us to operate a truck one hour. This is known as the Hourly Truck Cost and varies with the service and type of equipment, oats or gas (team or auto). This hourly cost is not constant

FIG. 4
Showing the Improvement in Profitable Use of Truck Time



THE COMMERCIAL CAR JOURNAL



ALTHOUGH Mr. McEwan operates a small fleet of trucks, the principles which caused him to increase the value of his truck equipment apply alike to all fleets. And since he expounds them very well, and proves their soundness with practical results, we think his article merits careful study.

If it stimulates thought and action in the direction of increasing truck working hours and thereby reducing idle truck time, it will have served its very constructive purpose.

but is revised as needed. Knowing the hourly truck cost, the number of hours taken by each tonnage movement, the tonnage, and the ratio of the productive and unproductive time, it is relatively simple to determine the unit cost for each type of work.

July, 1931, found us with six months' data on hand and improvements under way. We were working under a fixed program as we had found conditions so bad that very definite and major changes would have to be made in our operations if we were to accomplish anything. This program called for the reduction of our idle time by the cutting of our fleet and the extension of our delivery territory. The team was indicated as the unit which could best be spared; but sentiment stood in the way, and it was not until 1932 that we finally eliminated this unit. Our excess truck capacity was carried over from our old building where we had been forced to overcome our plant shortcomings by rolling equipment. As a result we were over-equipped and found ourselves with three full truck days a week for which we had no work. Our records showed us that our trucks were actually working only 50 per cent of the time.

Our only real solution was to extend our delivery territory, thus putting some of the idle time to work. Accordingly we worked out a program of delivery



route extension, securing our tonnage from the 35 per cent carried by outside truckers. This program was based upon the fact that we could handle the extra tonnage without adding to our equipment. We planned to spread our fixed charges over a greater tonnage and thereby reduce the fixed costs per unit handled. As we already owned and maintained our equipment, whatever we secured over and above our bare operating expenses would apply against the fixed charges. Of course we were using the existing truck rates as a measuring stick.

The estimating of one of our first extensions will be outlined to show how we actually worked this out. This is the run, east of Troy, that is designated as Rural Route No. 1. Our Volume Record showed that we were sending about 1000 pounds into the towns in this section on alternate weeks, at a rate of 30 cents a hundred. We estimated that it would take a truck three hours to cover this territory and we used the current hourly truck cost of \$2 to determine that the route cost would be \$6 or 60 cents a hundred. However, with a 25 per cent operating cost ratio our actual added cost would be 15 cents a unit, which would leave the other 15 cents of the 30-cent rate to apply against the fixed overhead. We fully realized that our operating percentage was low for this work since the mileage per hour would be higher than it was in the city work upon which all our costs were based at that time. However, we had the assurance of the sales department that the tonnage would increase under our own delivery and we permitted this expected increase to balance the higher operating cost item. That we were safe in doing so proved to be the case. In fact, we began immediately to handle heavier orders and eventually carried enough tonnage to bring our own costs below the outside rates.

This increased tonnage came from several sources and accompanied all changes from outside truckers. For one thing, we frequently ran into towns to which we had been unable to secure

satisfactory service. Also, with our trucks running into a town, we could carry items formerly prohibited by freight charges. We ourselves could carry these for the extra gas, etc. Then, naturally, our salesmen would work to sell a full load because they were paying for the trip whether the truck carried 5 tons or 500 pounds. The tonnage increase ran from 50 per cent to 150 per cent in the different territories under our own deliveries. Eventually we carried practically everything on these routes.

By July, 1931, we were operating on three rural routes, R. 1, R. 2, and R. 3A, as shown on the Map, Fig. 1. The first two were extension runs, the truck proceeding from the last city stop into the country; but Route 3A was entirely country work. It was on this latter run that we got our first really reliable costs on the new work and found that these figures were checked by those on the other two. With the introduction of these three runs we had taken up all the useable work and were then forced to wait for the sales department to make more available.

Some very definite progress had been made. Reference to our records showed that in the second half of the year our trucks had actually worked 140 hours on these runs. When this actual time was increased to include the platform and other unproductive time, it became 228 hours switched from waste to earning time. We were justified in charging these routes with their fair burden of unproductive time as our unit costs were approaching our former rates. There was no question but that this new work would eventually pay its way and was even now in a position to stand its full share of our fixed overhead. These 228 hours at the current hourly cost represented a service or route cost of \$570. As the delivered tonnage was 70 tons, we were carrying these goods for about 40 cents a hundred. This compared very favorably with our former rates, especially since we knew our costs would drop with our rising efficiency.

Although we had started only three

small runs, we felt that very satisfactory results had been obtained. A type of work that had been undertaken to lower costs by spreading our fixed charges and which we expected to do little more than pay its bare added cost was paying its full cost, fixed as well as variable. As soon as we saw that this was the case, we changed our original plans and proceeded to actually compete with the outside truckers.

The year 1932 was the cost eventful of the three since we started work on our high costs. During 1931 we had worked out and proved our plans and we were now ready to go ahead on solid ground. The first event in 1932 was the starting of the Hudson run, Rural Route No. 3. This territory was covered by the Chatham (Route 3A) truck, which delivered that town on the return run. This Hudson route was especially important in that it used enough idle time to drop our city costs more than one cent a hundred.

The Hudson run filled our week with the exception of Wednesday, some of the formerly idle time having already been used in other changes to make the extension schedule possible. In April a day's work was made available on alternate Wednesdays, and we started Rural Route No. 4 as shown on the map. Because of unsatisfactory delivery we had not been obtaining profitable business in this territory and, at the request of the sales department, we had studied our covering of this section. In this particular case we reversed the procedure. We told the office what tonnage was needed to produce a given delivery cost. The tonnage necessary to give a reasonable figure was about twice normal. Our experience told us that we could expect this increase, and we started the run. The tonnage rapidly rose to a satisfactory figure, and, although higher than on any other route, the costs have always been within reason. As we are doing a very profitable business in this section, this run has always paid its way.

This was not all for the year 1932. In October we eliminated the team, with the drop in truck overhead shown in Fig. 2. Considering the amount of effort, this was the biggest move in the whole program. In one step we reduced our overhead by \$3000 a year.

We were covering Route No. 4 every other Wednesday, but we were still short a day's work on the alternate week. This last gap was filled by our extension to Bennington, Vermont, in March, 1933.

With the institution of our Bennington run we completely filled out our schedule, finishing the extension program started two years before. While this particular work has been under way, we have been working to improve our trucking as a whole. While some

progress has been made in this line, we have only recently begun serious work, since up to now our equipment has been equal to demands. Due to business conditions we have been giving the best possible service to our trade, with the result that considerable work is being done without regard to the finer points of truck operation.

An indication of the improvement that we have made is found in Fig. 4 which shows the percentage division of truck time for three representative quarters. It will be noted that our idle time has shrunk from 29 per cent to 8 per cent. A further sign of improvement occurs in the ratio of loading time and delivery time. When it is realized that considerable of this delivery work is done in the country where the tonnage handled per hour is one-third that in the city, and that we have actually kept the average tonnage handled per hour constant, it will be seen that we have already made definite progress.

The actual figures of the extension routes and the resulting costs follow:

ROUTE NO.	1931		1932		1933		TOTAL		Route Cost In Cents	Former Rate 100 lb.
	Tons	Cost	Tons	Cost	Tons	Cost	Tons	Cost		
1	8	\$50	35	\$220	39	\$200	82	\$470	29	30
2	36	\$300	44	\$390	55	\$420	135	\$1,110	41	35
3	26	\$220	215	\$1,615	203	\$1,230	444	\$3,065	35	40
4			36	\$450	59	\$572	95	\$1,022	54	40
5					75	\$533	75	\$533	35	40
TOTAL	70	\$570	330	\$2,675	431	\$1,955	840	\$6,200	37	
COST in cents 100 lb.		40½		40½		34		37		

The behavior of our delivery costs is shown in Fig. 2. On this chart the Delivery Service charge has been averaged in two periods, the first ending with 1932 when the team was sold. This has been done to eliminate the effect of the pick up work and to show the true trend of our delivery costs for the three years.

As previously mentioned, our trucks were doing all our incoming hauling. The only room for improvement here was the general raising of efficiency. Furthermore, we had known that the costs would drop as we improved our operation through our extension work. The pick up figures are shown in Fig. 3. Incidentally, our city delivery costs curve has paralleled this pick up costs curve, while the average delivery costs curve has not fallen so sharply because of the rural work.

In carrying out our extension program and cutting our costs we have necessarily spent more money than if we had not undertaken this work. However, we have secured a very handsome return on the investment. The \$6200 that we spent on our extension routes in the three years represents 2510 truck hours. This sum is made up of 2510 hours of fixed overhead at 50 cents per hour, or \$1,255; 2510 two-man truck

crew hours at \$1.00 per hour, or \$2,510; plus the operating costs which are the remainder, or \$2,435. In other words, by spending this \$2,435, we saved \$3,765 (fixed charges plus labor) which we were already spending and which would otherwise be lost. Each dollar we spent brought back \$1.55. When we deduct the dollar we actually spend, we have a net saving of 55 cents. Fifty-five per cent interest on our investment is high enough to be worth working for. On top of this we are getting dividends in the form of doing our own work for less than we formerly paid others to do it for us.

Idle equipment is costly. When a truck stands still it does not require gas, oil, repairs, et cetera, but that old fixed overhead goes right along. We figure that if our equipment loses an hour a day, we lose the profit on \$500 business. Yet, if we had merely kept our equipment moving over our original territory, we should have been just as badly off. In fact, we should have been actually wasting money, as we should

have been "making work" really to keep our trucks on the streets. Our only solution was to increase the tonnage, which we did by extending our delivery territory. When we come to figure our savings, we find that they are more than we realized was possible with only three pieces of equipment with which to work.

Because of the inherent nature of truck costs with their many variables, it is difficult to figure accurately our savings of the past period. However, we can readily estimate our savings for the year 1934. We shall assume that in 1934 we shall handle a tonnage equal to the average of 1931, 1932, and 1933. Reference to the charts, Fig. 2 and Fig. 4, shows a pick up cost drop of at least three cents a hundred and a delivery cost drop of eight cents. Our Volume Records give an incoming tonnage average of six and one-half million pounds and an outgoing tonnage average of five million pounds. With our pick up cost drop of three cents on that tonnage, we shall save \$1,950 there; and with our eight cent delivery cost drop on the outgoing tonnage, we shall save \$4,000. Now that we have learned the value of keeping our trucks moving, we will certainly pick up that extra \$50 to make our yearly savings an even \$6,000.

Viscosity Affects Oil Use But Engine Speed is the Real Hog

COLD weather starting and winter lubrication, those devilish twins that annually plague fleet operators, are having their tails cut off and their pitchforks dulled by scientific developments. The man who is responsible for abating this related pair of nuisances and thereby earning the gratitude of operators, is H. C. Mougey (pronounced Moo-zhay, just in case you should accost him sometime to pay your respects), chief chemist of General Motors Research Laboratories.

An inkling of his accomplishments in the field of winter lubrication was revealed by Mr. Mougey (get it right now) at a recent meeting of the Philadelphia Section of the S.A.E. His talk centered chiefly around the new standard engine oils for winter use—10 W and 20 W, for whose adoption he was largely responsible.

Mr. Mougey stated that a fleet of cars had been sent up to Regina, Saskatchewan, in order to study problems of lubrication under severe winter conditions. During the period of the tests the temperature averaged zero and varied roughly between plus 45 and minus 45 deg. At one time there was a drop of 66 deg. in 36 hours and at another, 60 deg. in 14 hours.

Graphic records of cranking speed vs. time under cold weather conditions showed that if the engine is cranked for a considerable period without firing, the cranking speed is likely to increase, which is due to the oil on the cylinder walls becoming diluted with unvaporized fuel. This increase in cranking speed usually is of little benefit, however, because during the cranking period the manifold becomes loaded, and the mixture then is so rich that it will not fire.

Mr. Mougey cited a number of authorities as to the minimum cranking capacity required to assure starting in cold weather. P. J. Kent, chief electrical engineer of the Chrysler Corporation, had said the starter must be capable of turning the engine over at 40 r.p.m. with crankcase oil showing a viscosity of 6000 seconds at 0 deg. F. According to C. M. Larson, supervising



THE following conclusions developed by Mr. Mougey deserve emphasis:

1. The increase in starting troubles in recent years is chiefly due to the fact that we did not use oils that were light enough.

2. Whereas in former years we could depend upon viscosity of the crankcase oil being rapidly reduced by unvaporized fuel, this is no longer the case, at least not to the same degree.

3. Oil consumption is influenced more by engine speed than viscosity, this applying to both light and heavy oils.

And, as a tip to operators, addition of lard oil to crankcase oil is beneficial in running in new engines.



H. C. MOUGEY

engineer of the Sinclair Refining Co., the starter should crank the engine at 35 r.p.m. with oil of 18,000 viscosity in the crankcase, which corresponds to a viscosity of 30,000 for new oil. According to another authority the cranking speeds should be between 20 and 40 r.p.m. with oil of between 12,000 and 26,000 viscosity. The main reason we have had increased starting trouble in recent years, Mr. Mougey said, is that we have not used as light oils as we should have. It was to remedy this situation that the new engine oils 10 W and 20 W were adopted.

A chart was thrown on the screen showing the variation in fuel characteristics during the period 1920-1934. The volatility (as represented by the 10 per cent point) gradually increased until a few years ago, since which time it has remained substantially constant. On the other hand, the potentiality for crankcase-oil dilution, as represented by the 90 per cent point, remained substantially constant until a few years ago, when it began to decrease. From this the conclusion was drawn that whereas in former years we could depend upon the viscosity of the crankcase oil being rapidly reduced by unvaporized fuel, this is no longer the case, at least not to the same degree.

The 10 W and 20 W oils replace the former S.A.E. 10 and S.A.E. 20 oils which had viscosity ranges of 90 to 120 and 120 to 185 Saybolt-Universal seconds at 130 deg. F. Mr. Mougey said there was an inconsistency in specifying the viscosity limits for these oils, which are to be used at low temperatures, at 130 deg. F., because, owing to the difference in temperature coefficient of oils of various provenience, an oil which is within the viscosity range of S.A.E. 10 at 130 deg. may be more viscous at zero degrees than another oil which comes within the S.A.E. 20 range at 130 deg. For this reason the viscosity limits for 10 W and 20 W oils, which are intended primarily for cold weather use, are specified as at zero deg. F.

Some anxiety had been expressed as to the ability of the new winter lubricants to provide efficient lubrication

under hard-driving conditions. Mr. Mougey set at rest any doubts on this score by mentioning that Ab. Jenkins, when making his record-breaking drive of 25 hours on the Salt Lakes of Utah, used S.A.E. 20 W oil in the engine of his car, the use of this oil having been decided upon after it had been found that it was possible to get 3 m.p.h. higher speed with it than with S.A.E. 30 oil. If the oil served satisfactorily in this engine, which was driven "all out" for long periods, it certainly should give adequate lubrication in engines that are driven under ordinary conditions.

The safe temperature and viscosity limits, Mr. Mougey pointed out, depend on the composition of the bearing metal and the oil, on design, and on operating conditions. Babbitt begins to crumble at 400 deg. Fahr. and the temperature of the bearing therefore must be kept below this point in order to prevent failure. Ordinarily there would be a difference of 100 deg. between the temperature of the bearing and the temperature of the crankcase oil, which would reduce the safe limit for the latter to 300 deg. A slight margin should be allowed even on this limit, and Mr. Mougey exhibited a chart on which the minimum safe viscosity of the crankcase oil was plotted as a function of the maximum oil temperature, according to which the minimum safe viscosity is about 35 seconds and the maximum permissible oil temperature, 280 deg. This would indicate that it is perfectly safe to use 10 W and 20 W oils in winter time. Confirmation of this result was furnished, moreover, by the findings of a Committee on Cylinder Wear of the Institution of Automobile Engineers which diluted crankcase oil with 90 per cent of kerosene without finding any appreciable increase in the rate of wear.

Numerous charts were exhibited showing the relation between oil viscosity, engine speed and oil consumption. In practically all cases the consumption increased very rapidly with the speed, this applying to both light and heavy oils. Also, the consumption was always greater with the lighter oils, but the difference in the rates of consumption of light and heavy oils respectively varied remarkably with different engines. In cars in poor repair the light oils were found to give very nearly the same mileage as the heavier oils, though the expectation of a greater mileage is usually the reason for preferring the heavier oils. Engine speed has a much greater effect on oil consumption than oil viscosity. In one series of tests, when the car speed was increased from 30 to 55 m.p.h. the oil mileage decreased from 4,000 to 580 miles per gallon. This ratio of 6.9 to 1 was the average from a considerable number of cars, the lowest being 2.3 and highest 19.8.

Another advantage of the lighter oils is that with them bearing lubrication starts sooner after the cold engine has been started. Charts of engine characteristics thrown on the screen, taken while using 10 W and S.A.E. 30 oils respectively, showed that the horsepower and torque were greater with the former, while the specific fuel consumption and the friction horsepower were lower.

An addition of 10 per cent of lard oil did not change this relationship between the characteristics obtained with 10 W and S.A.E. 30 oils respectively.

In the discussion following the talk Mr. Mougey explained that the initials S.A.E. were not applied to the new oils because it was desired to subject them to a period of trial first, and that the letter "W" stood for Winter.

In reply to another question, the author stated that additions of lard oil were beneficial in running in new engines. Such engines, if a little stiff, can be worn in more quickly and with less danger if a little lard oil is added to the mineral lubricant. Later on it is difficult to show any advantage in lard oil additions in the average engine.

What's Back of the Come-Back of the Camel Back

(CONTINUED FROM PAGE 20)

course is the horizontal-opposed piston type engine, either gasoline or diesel, which might be typified by the flat 12-cylinder gasoline engine introduced a year ago by The White Co.

Another rather novel design is the crankless diesel introduced at the last motor boat show by the Sterling Engine Co. At present this engine is built in the form of a marine power plant, but very soon it is expected to be built for installation in trucks.

One rather interesting construction is suggested by the Hall-Scott engine-on-side design which was first used on a 40-passenger A.C.F. bus last year. This engine, like the White horizontal opposed piston type, is mounted under the frame with all accessories on the under side where they can be readily accessible. It can be serviced completely over an inspection pit. The engine must be installed back of the front springs because of its width. This simplifies the cab construction as the cab can now be mounted directly on the rails in a normal low position requiring little, if any, insulation against engine heat.

As a practical matter the selection of the chassis, wheelbase, etc., should be viewed in the right light by the purchaser and engineer. All engineering is a compromise and the new construction is no exception. On the basis of preliminary calculations you may find that

a certain wheelbase, for example, is needed to produce the desired distribution of 1/3-2/3.

The reasonable thing to do is to select the nearest standard wheelbase that is available in production. This is not only reasonable and economical but amply justified by engineering considerations.

• Future Possibilities

What the future holds is justly a matter for conjecture being hedged about with many variables, not the least of which is a guess as to the probable trend of State legislation. But there is an interesting possibility. We find that all the activity pictured above is confined to a class of vehicles representing only 1.36 per cent of total truck production. Wouldn't the "close-coupled" construction be of benefit in some of the lighter truck classifications?

The White Co. seems to have made a big stride in that direction with the 701 and 702 series of 1¼ to 1½-ton and 1½ to 2-ton capacity respectively, introduced a few months ago. These jobs embody a modified construction with the cab moved forward so that the engine projects into the cab about 9 in. The C-A dimension, back of cab to center of rear axle, runs about 9 to 14 in. more than for conventional vehicles of the same wheelbase.

When this dimension is utilized correctly, you get a load distribution of approximately 1/3-2/3. This makes a more pleasing looking vehicle and results in better loading of the tires. In fact in some cases it makes possible a reduction in tire size for the same rating due to the reduction in load on the rear tires.

Look to the Carburetor

(CONTINUED FROM NEXT PAGE)

up to date, enabling them to be operated at a profit, by utilizing this new equipment. Vehicles operating under conditions which may have changed greatly from those for which the trucks were originally designed, can in many cases, be materially improved in performance by making use of the new model carburetor units.

In certain kinds of service, a large amount of dirt and foreign material is drawn into the carburetor, necessitating frequent carburetor overhauls and cleaning. In such cases, the sealed and balanced carburetors, ready for quick installation, are of obviously great advantage. The moderate added cost which this type of replacement entails, is soon more than made up by the lengthening of the periods between cleanings.

Serious consideration before deciding which of these two courses to follow, often results in a considerable saving to the fleet operator.



A FEW thoughts on a subject that is a mystery to many fleet operators and many in the trade, and which will help to determine when to rebuild and when to replace a carburetor.



BY A. H. WINKLER, CHIEF SERVICE TECHNICIAN, BENDIX STROMBERG

Look to the Carburetor If You Are Looking for Economy

THE subject of carburetion is still somewhat of a mystery to a great many commercial motor fleet operators. The lack of accurate information on this subject has often resulted in the cost of vehicle operation per ton-mile increasing after a carburetor unit has become worn and out of adjustment, despite the fact that there had been no change whatever in operating conditions.

There are two means of keeping operating costs down through routine attention to the carburetor unit. The first is, cleaning and rebuilding the original unit; the second is, replacing it with a carburetor of later and more efficient design. Which one the fleet operator should choose depends largely upon the type of service in which the vehicle is operated. The following is a brief outline of just what the two terms *Rebuilding* and *Replacing* mean.

When a carburetor has been in service over a period of months or years an accumulation of dirt, sand, grit and corrosion necessarily collects around the various metering holes and valves, caus-

ing an irregular and altered flow through the carburetor. This can result in only one thing—increased fuel cost per ton-mile.

The thought naturally arises in the fleet owner's mind that if the fuel is retarded by partly plugged holes, less fuel is passed through the carburetor and more economy should result. This is not true, however, because after a certain point is reached in "leaning-down" a carburetor, the throttle must be opened further to allow more fuel to enter the cylinders in order to produce the same amount of power. This fuel is not burned efficiently because of the great excess of air present.

Mechanical wear of carburetor parts also takes place. Throttle shafts, and valve faces and seats, are subject to severe punishment, because it is through the carburetor throttle that the truck driver changes the speed of his truck. The effects of this wear are shown by decreased mileage and difficulty in satisfactorily idling and starting the motor.

It should be the policy of the carburetor manufacturer to factory-rebuild

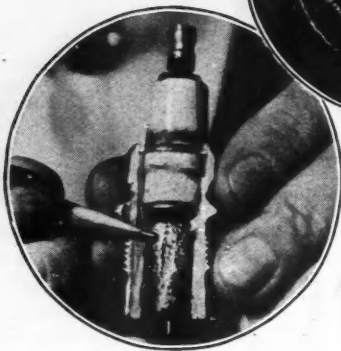
worn carburetor units in such a manner that they are practically as good as when they originally left the factory. During the rebuilding process the carburetor must be completely taken apart, even to the smallest plug, and each part inspected separately before being put through a cleaning process. After the parts are cleaned, the various valves, bushing and metering holes must be checked for wear or defect, and replaced with new parts where necessary. When the carburetor is reassembled it should be pressure-tested on a special apparatus, so that any chance leaks may be found, and final adjustments made, before the instrument is returned to the truck operator and again put into service.

As engineering skill has progressed from year to year, new designs have been developed to improve the performance and economy and lengthen the time interval between these cleaning and rebuilding periods on any carburetor installation. Old and apparently obsolete carburetors may sometimes be brought

(CONTINUED ON PAGE OPPOSITE)



Oxides deposited on the insulator are good conductors of electricity at high temperatures and, in effect, short circuit the plug



Maintenance Men: Know Thy Spark Plugs!

A Commandment for Those Who Want to Get the Most Out of Engines

SOME day we hope you fellows in the maintenance field will understand that a spark plug has something else to do besides give you a lot of trouble, and it will give a lot less trouble if you try to understand what it is supposed to do."

We are quoting here from a talk by Alex Taub, chief of the engine section, Chevrolet engineering department, before a recent session of the Metropolitan Section Society of Automotive Engineers.

Mr. Taub did not elaborate on the statement at the time, but there is plenty of truth in it. A spark plug can and will give trouble if it isn't the right one, or if it isn't looked after. On the other hand the spark plug isn't always to blame for some of the troubles credited to it.

Among the things which sometimes may be traced to spark plugs, are the following:

1. Hard starting.
2. Loss of power at high speeds.
3. High fuel consumption.
4. Missing at low or idling speeds.

Obviously there can be many possible causes for any of these, but it does seem that since spark plugs are so easily ac-

cessible, are so simple to clean, adjust or replace that at least that one possible source of trouble could be guarded against. There are really only a few things that can be wrong with a spark plug.

One group of troubles arises from using the wrong type or length of plug, or a plug which is either too hot or too cold for the particular engine, and the other group comes from incorrect spark plug care.

Of the first group, the engine manufacturers, working together with the spark plug manufacturers, have scientifically determined everything except the matter of sensitivity to temperature for varying operating conditions. It is quite important in some engines just how far the gap is from the shoulder of the plug. Given two plugs of the same general type, a "long" plug may work better in one make of engines, a "short" plug in another make. By working better we mean the development of maximum power or torque, together with good idling characteristics and best fuel economy.

In many engines the position of the gap with respect to the inner surface of the combustion chamber is highly important. When fuel enters the chamber, particularly at low speeds, the density of the mixture is not even throughout the chamber—you will have layers of thin and heavy mixtures, called "stratification" by engineers. Engineers explore the chamber near the spark plug to find if, or how far, the plug should extend into the chamber so that the spark will occur in a layer of dense mixture—insuring more complete combustion by proper ignition of the charge.

To illustrate the point, Fig. 1 shows a chart of fuel consumption in miles per gallon for the same engine, used in a low-priced truck, at various road speeds from 20 to 55 miles per hour, the only difference between the operating conditions being in the matter of spark plug length. The solid line is for plugs which extended $\frac{3}{8}$ in. farther into the combustion chamber than the plugs which gave the dotted line. In another make of engine the condition might be reversed.

By ATHEL F. DENHAM

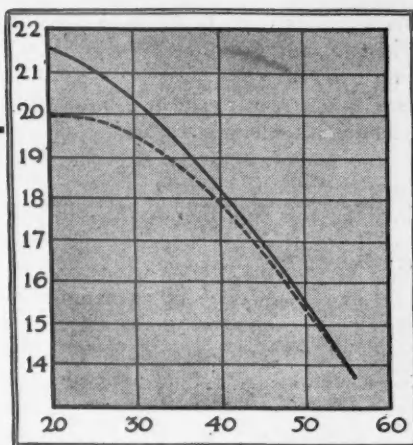
The reason for this difference in fuel economy of course is incomplete combustion—caused by either incomplete combustion (partial missing) or complete misfiring in some of the cylinders occasionally. This missing will not be noticeable necessarily to the driver; at the worst it might come to his notice only in the form of a very slight increase in engine roughness. If soft rubber mountings are used, even this symptom might not show up. At any rate the best way to avoid fuel and power losses from this source is to be sure the spark plugs used are those specified by the factory, where engineers have taken weeks to determine the best position.

Next there is the matter of whether the plug is too hot or too cold. Perhaps a word of explanation is in order. A spark plug at best is a compromise between best idling and maximum torque operation. As the throttle on an engine is opened the spark plug temperature increases. If the spark plug electrode does not cool rapidly enough the high temperature of the spark plug will cause pre-ignition.

On the other hand if the plug is designed for rapid cooling, it may cool too fast when the engine is idling or running under light load, causing fouling. Fouling due to carbon deposits not being burned off is due to a plug which runs too cold.

The difference between a hot and a cold plug is generally only a matter of the distance from the tip of the insulator to the seat of the insulator in the metal shell—the longer the distance other things being equal, the "hotter" the plug.

Spark plugs that come with new engines generally lean toward the "hot" side. All new engines pump oil to a



These curves show the variation in fuel consumption in an engine resulting merely from substituting a plug $\frac{3}{8}$ in. longer from shoulder to gap than standard. In another engine the same effect might be obtained by substituting a shorter plug. The graph reads m.p.g. along the side and m.p.h. along the bottom

greater or lesser degree before the rings are properly seated, pistons and cylinder walls worn in. Moreover many people use oil in gasoline for breaking-in purposes. To prevent fouling of the plugs during the break-in period, when engines are generally run at comparatively low speeds also, engineers provide a slightly hotter plug than would be desirable when the vehicle is later operated under considerable loads and at relatively high speeds.

For door-to-door delivery service on the other hand—and other services where the engine is seldom run at high speeds, or is idling a good portion of the time—an even hotter plug than is recommended by the factory may occasionally be necessary. Factory standard plugs are designed, as we have said, as an effective compromise for "average" service. Operating conditions change the requirements frequently. However, in changing from a hot to a cold plug or vice-versa, care should be taken that the plugs are of the same length—so that the position of the gap in the combustion chamber is not changed.

So much for plug selection. Far more

C. C. J. DETROIT EDITOR

Tips on Spark Plugs

● Re-gapping should be done with a feeler gage because it is a precision operation.

● Avoid breaking the insulator by making all re-gapping adjustments by bending the side electrode only.

● If plugs are fouled, don't be content with cleaning them—look for the source of the trouble.

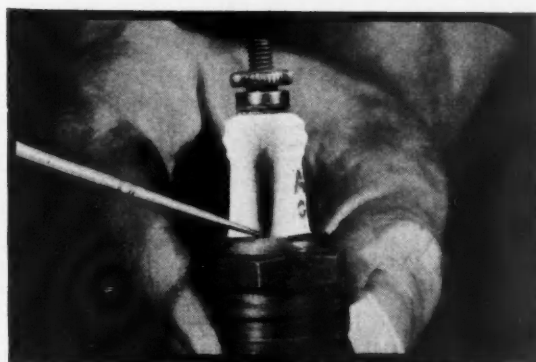
● Spark plugs should be re-gapped every 3000 to 5000 miles. The gap gradually widens after several thousand miles normal service. If it widens quickly and unnaturally the wrong type of plug is being used.

● Gap position in combustion chamber should be according to manufacturer's recommendation for best performance. Use correct length of spark plug.

● The recommended gap setting is best for average service. In case of a bad idle, or hard starting, it might be necessary to use a wider gap—not more than ten thousandths wider. A wider gap requires more frequent cleaning for consistent service. Gaps should never be adjusted narrower than specified.

● Select hot or cold type of spark plug to give best operating temperature. The type should be such as not to cause pre-ignition or too rapid deposits of oxides when the engine and accessories are in good mechanical condition. For low speed service the plug should be hot enough to prevent accumulation of carbon and soot.

● Keep the plug in condition by periodic cleaning and re-gapping, then if troubles develop, you know you can look elsewhere for the cause.



Left — Pre-ignition may cause blowby between the insulator and shell. It can be eliminated by using a cooler plug

Right—Re-gapping is a precision operation. Don't guess—use feeler gage



important is the question of proper care of spark plugs. Here two factors mainly are important: the gap width, and the amount of coating on the insulator, forgetting such obvious items as cracked insulators, etc.

In one factory a group of engineers takes about three weeks to determine the correct gap width for its standard spark plugs every time a new model is brought out. This may seem rather ridiculous in view of the fact that spark plug gaps in the field are more often adjusted, if at all, entirely without the use of feeler gages, or reference to factory recommendations. Yet the width of that gap plays an important function in the operation of the engine.

Some engineers claim that a tenthousandth of an inch error in gap width might in some cases produce a 5 to 8 per cent loss in fuel economy. Whether or not this is correct, both too wide or too narrow a gap will give trouble eventually in some form or other.

Ignition systems are merely a means of transferring energy from the battery into a spark at the correct instant. A given ignition system can supply just so much energy at the spark plug and no more. Remembering that, and considering that the wider the gap the more energy it requires to make a spark jump that gap, and the faster the engine operates the less effective the ignition system, it can be seen that if the gap is wide enough the spark won't jump at all or at least will be very weak—causing mis-firing at high speeds.

A better illustration perhaps is the fact that if instead of twenty-thousandths of an inch, the gap is forty-

thousandths wide, that would be equivalent to difference between one battery and two batteries in series. The wider gap would require the two batteries to give just as effective a spark.

Too narrow a gap on the other hand may cause bad idling and hard starting. In this case the spark itself may not be sufficiently hot or sufficiently long to produce proper ignition of the fuel and air mixture in the combustion chamber. Too narrow a gap might also cause mis-firing at high engine speeds in some engines by not properly igniting the mixture.

It is not likely that spark plug gaps will be found too narrow in service unless they have been set that way in adjusting the gap. They may frequently be too wide, however. There is a normal "wear" of electrodes at the gap, and in addition there may be burning of the electrodes due to high temperatures. It is highly recommended that for maximum economy and consistency of operation, spark plugs should be re-gapped whenever they are cleaned.

That brings us to the last but perhaps most important factor contributing to spark plug troubles—deposits on the center insulator. These deposits may be of various types: there may be oil from oil pumping, etc.; there may be carbon deposits which have not been burned off. The latter, if present, demonstrate that the plug used is too cool in operation.

Then there are the oxides which deposit from the fuel burned, and in some cases special fuels leave additional forms of chemical deposits on the plug. All of these are conductors of electricity and have the habit of increasing in

conductivity as their temperature is increased. Being conductors, they to some extent "short-circuit" the plug and rob it of some of the energy derived from the ignition system. At high engine speeds, when spark plug temperatures are high, this may cause mis-firing—again of a nature which cannot be readily detected by the operator of the vehicle. This type of mis-firing, which may cause a loss in fuel economy of 10 per cent or even more at times, is particularly hard to notice. What happens is the spark plug mis-fires a couple of times—during this period the temperature in the combustion chamber, and that of the plug, drops—the coating on the insulator cools off, loses its conductivity, and again there is sufficient energy for the spark to jump across the gap. The cycle then starts over again.

It will be seen incidentally that the wider the gap the smaller the amount of coating on the plug necessary to cause short-circuiting and mis-firing.

The best answer to this problem is frequent cleaning of the insulator. An insulator normally is white. If it has a reddish brown color, or if it is glassy, or blistered, it has a coating of oxide even though that coating isn't apparent in any other way. It is a good policy to clean spark plugs about every 3000 to 5000 miles of operation. If the rapid appearance of these coatings is a chronic condition of the engine, it is possible that the plug may be too hot, for the higher the temperature the more readily these coatings form. If the substitution of cooler plugs doesn't solve the problem, the trouble is probably in the ignition or intake system of the engine, including the carburetor.

Federal's New 2-Ton Lists at \$845

FEDERAL Motor Truck Co. has announced a new 2-ton truck at \$845. Known as Model 18X, it has a maximum gross rating of 11,000 lbs. In appearance, in the number of available wheelbases and in the matter of power-plant, it is similar to the 1½-ton 15X described on page 36 of the March issue.

Standard tires are 6.00/20 six-ply single front and dual rear. The standard 137-in. wheelbase chassis weighs 3800 lbs.

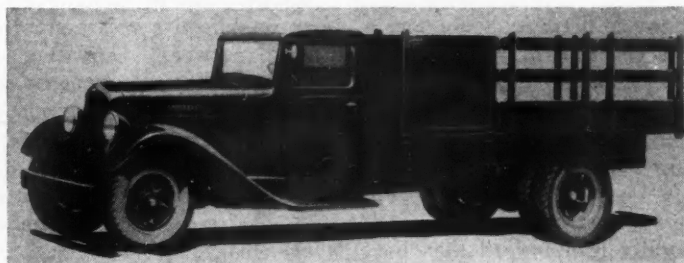
While the Hercules JXA is standard, a larger engine (JXB) of 263 cu. in. displacement is optional at extra cost.

The fish belly frame has a maximum depth of 8½ in. Flanges are 2⅞ in. wide and the thickness of the steel is 7/32 in. Front springs are 38 in. long by 2½ in. wide. Rear springs are 50 in. long by 2½ in. wide. Five leaf auxiliary

springs 41 in. long by 2½ in. wide are standard equipment. All spring eyes are rubber bushed and require no lubrication.

A 4-speed transmission is mounted in unit with the engine. There is an opening on the right side of the transmission providing for a standard power take-off. The transmission ratios are: First speed 6.40 to 1; second speed 3.09 to 1; third speed 1.69 to 1; fourth speed 1.00 to 1; and reverse 7.82 to 1.

The rear axle is full floating bevel drive type with a straddle mounted pin-



ion gear. Standard gear ratio of the rear axle is 6⅞ to 1 with 6⅜ to 1 optional. Universal joints are of the roller bearing type which require lubrication only at very infrequent intervals. A single propeller shaft is used on the standard 137 in. wheelbase chassis, while two shafts with a self-aligning center bearing are provided on the longer wheelbases. Large, powerful hydraulic brakes of the internal type have a braking area of 260 sq. in., 15 in. x 2¼ in. F., 16 in. x 2¼ in. R. fully weather proof. All brake drums are of cast alloy iron.



This shows IHC Model C-1 with roomy, attractive cab and all-steel pick-up body

IHC Builds New Half-Ton in Own Plants

A NEW half-ton truck, improved in appearance and completely built in its own plants is announced by the International Harvester Co. of America. It is a 113-in. wheelbase job which will be known as the C-1. The chassis will list at \$400 f.o.b. factory. With shock eliminators and bumper the list will be \$425. Body prices are not yet available.

In appearance the new model is distinctive and decidedly pleasing. A stream-lined hood and cowl, V-type radiator and valanced fenders all contribute to its attractiveness. The radiator shell is finished in the same color as the hood and cowl and trimmed with a polished stainless steel moulding and satin-finished aluminum grille.

The cowl is furnished as standard equipment, less the windshield. The latter is sloping and hinged at the top. The cab is roomy, comfortable and insulated against heat and sound. It is of composite wood and metal construction, with a one-piece, deeply-crowned steel roof.

The six-cylinder L-head engine, with 3 5/16-in. bore and 4 1/8-in. stroke, has a displacement of 213 cu. in. Brakehorsepower is 78 at 3600 r.p.m., and 148 lb. ft., of torque are developed at 800 to 1000 r.p.m. The precision-type main and connecting rod bearings are remov-

Chassis of 113-in. wheelbase Model C-1 lists at \$400 with shock eliminators and bumper \$25 extra. Front-end changes greatly improve appearances.

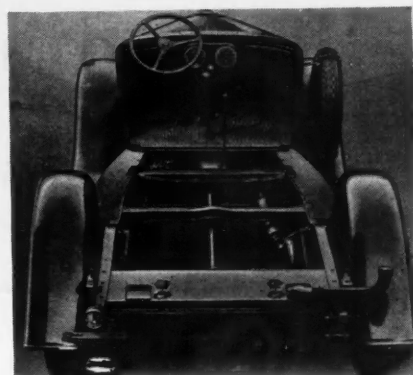
able and replaceable. Hardened exhaust-valve seat inserts are employed and the downdraft carburetor is fitted with an air cleaner.

The frame is sturdy and well reinforced by gusseted cross members. Frame dimensions are 53 3/4 x 21 1/4 x 9/64.

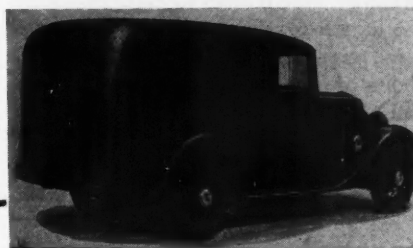
Other chassis features include cam-and-lever steering, roller bearing anti-friction type universal joints and internal-expanding brakes of the equal action cam type.

The all-steel pick-up body is 47 1/8 in. wide and 66 in. long inside. Side panels 13 in. high are provided with 6-in. flare boards. Stake pockets are provided so that a special canopy top may be mounted.

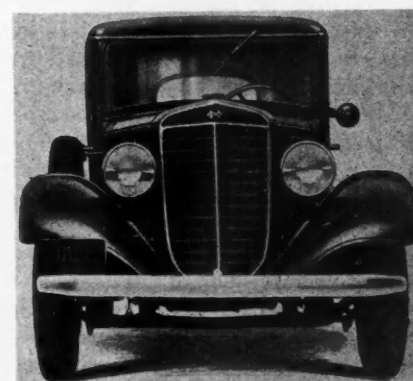
The panel body is 6 ft. long, 52 in. high inside and 55 in. wide inside at the belt.



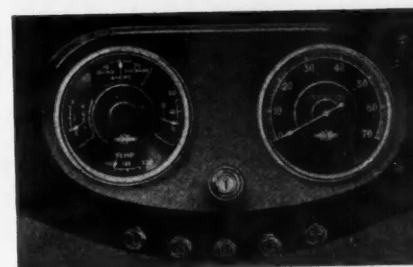
Plan view of the C-1 chassis



The C-1 with a 6-ft. panel body mounted on the 113-in. wheelbase



The V-type radiator has a satin-finished aluminum grille



Airplane type instruments and conveniently located controls feature the C-1 instrument panel

Buda Breaks Out With Full Diesel in Two Sizes

80 and 90 hp. jobs differ only in bore. Use Bosch fuel injection

THE Buda Co., of Harvey, Ill., has a line of heavy-duty, six-cylinder automotive diesel engines, built under license from the German M.A.N. company. Both sizes of engines have a 5½-in. stroke, with bores of 3 13/16 and 4 in., respectively, for piston displacements of 377 and 415 cu. in. Both engines are governed at 2000 r.p.m. at which speed the smaller develops better than 80, and the larger above 90 hp.

The engines are of the four-cycle, full-diesel, solid-injection type, using Bosch fuel injection systems. An air storage chamber is incorporated in the cylinder design. These chambers are closed off by a rotating plunger type valve for easy starting when the engine is cold. With the air chambers thus closed off—by means of a single lever operating all six valves—compression ratio is 17 to one.

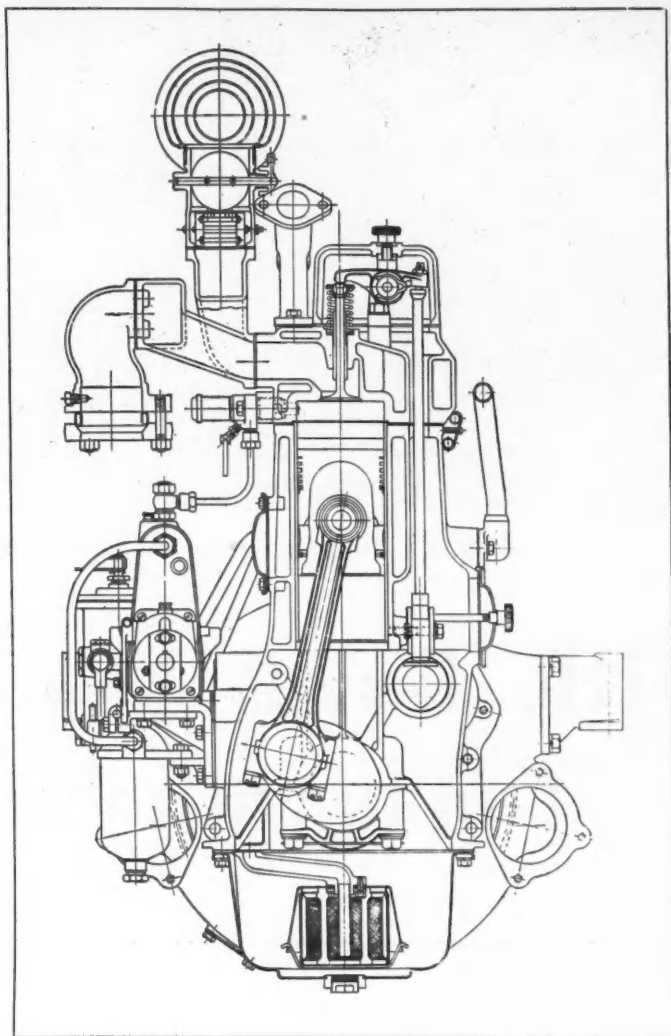
The valves are opened again when the engine has reached its normal operating temperature—that is, a temperature of 130 deg. Fahr. or better—under which conditions it will then be running on a ratio of roughly 13 to one.

These engines show an unusually low maximum rate of pressure rise, which accounts for their relative smoothness of operation and the fact that fairly light weight could be achieved without resort to light-metal engine castings. The only major aluminum alloy part is the cylinder head cover enclosing the overhead air intake and exhaust valves.

The object of operating on a relatively low compression ratio, according to Buda engineers, is to obtain the maximum air supply in the combustion chamber, achieving an increase in turbulence for more effective combustion, thereby increasing the horsepower output and combustion efficiency, as shown by a reduction in the amount of smoke in the exhaust gas.

Best fuel consumption is obtained apparently in these engines in the 1200 to 1600 r.p.m. range. The maximum

This cross-section of the Buda M.A.N. diesel engine shows the location of the injection pump and nozzles



B.M.E.P. values are obtained in the same engine speed range, although the maximum torque curve is relatively flat from 600 to 1800 r.p.m. The engines do not carry air compressors for supercharging.

Aside from the fuel mechanism the powerplants do not depart widely from accepted good heavy-duty automotive design practice. Blocks are fitted with replaceable dry type steel sleeves clamped in place by the cylinder head. There are seven main bearings of the steel-backed precision, interchangeable type replaceable individually without disturbing the crankshaft.

Connecting rods are rifle drilled for pressure lubrication to the full-floating piston pins. Pistons are of aluminum alloy with piston head design incorporating a transverse specially shaped trough to increase turbulence. There are six rings per piston.

Both intake and exhaust valves are of silchrome steel. Tappet adjustment can be made with the engine running. Valve seat inserts are provided for the exhaust valves. Valve springs can be replaced without removing cylinder heads. Pistons and connecting rods are removed through the top of the block.

Incorporated in the cooling system is

a Harrison oil temperature regulator. Lubrication is of the full pressure type throughout including the rocker arm shaft, and governor.

Impeller type pumps for water circulation are mounted on the timing gear cover, and are driven by the camshaft gear. From the pump water goes through the oil temperature regulator, where it divides, part going to a distribution gallery on the cylinder head, the remainder going into the block. Cooling fans are mounted on anti-friction bearings.

The Bosch fuel system is too well known to require description here. Fuel injection nozzles are of the closed pintle type and normally should not require adjustment. Control of the amount of Fuel injected is through a variable cut-off on the injection plungers, the stroke remaining constant. The Pierce governor regulates the amount of fuel directly, the hand "throttle" being connected to the governor.

A variable timing device which regulates the start of injection is provided, but adjustment of this mechanism is not recommended subsequent to shipment from the factory. There is in the system also a Bosch fuel filter, while the air intake is provided with an intake silencer and air cleaner—combined.

\$ALVAGE—From a Shop Man's Mail

A Gassing Eliminator

HOW to eliminate obnoxious exhaust fumes has been a subject for discussion among fleet operators for a long time. To help solve this problem—generally referred to as “gassing”—there is the Doering Gassing Eliminator. It is simple in design and construction. The body is of cast iron and the cam and rollers of hardened steel. It works this way:

When the forced momentum of the engine causes a greater velocity or r.p.m. than idling speed when deceleration takes place, the value of the Eliminator is automatically snapped open by the vacuum in the intake manifold, thus cutting off the fuel supply from the carburetor and permitting free air to be drawn into the engine. When the momentum or the r.p.m. of the engine diminishes to idling speed or until the throttle is again opened, the valve of the Eliminator automatically closes and the engine continues to operate normally.

This device will be sold nationally. We'll put you in touch with the distributor.

A Locking Expander

An expander for curing piston slap in both cast-iron and aluminum-alloy pistons has been announced by Liberty Accessories Corp. It comes in three sizes to equip all cars and many trucks. Installation is made simply by means of a special pair of pliers. A soft steel locking pin prevents the expander from working out of position.

Hydraulic Cylinder Hone

Wagner Electric Corp. is marketing a home set adaptable to drill press operation for honing hydraulic brake cylinders. The set is contained in a steel box with complete instructions. It takes care of brake cylinders from 1 in. to 2 in. inclusive, and lists at \$15.50.

Rust-Proofed Clamp

Wittek Mfg. Co. is putting up its Star hose clamp in packages of 10 as well as 100. The clamp is a universal type and fits all hose from 1 in. to 3 in. The clamp is rust-proofed and the bolt and nut cadmium-plated. Catalog sheet and other literature will be supplied.

A Paint-Spray Help

A new, patented fluid cut-off valve, for use with its paint spray guns, has been developed by The DeVilbiss Co. The valve permits the operator to drain the gun of fluid or to blow out and obstruction without disconnecting the gun from the hose, thus increasing production speed.

Brake Lining Cleaner

Armite Laboratories has a liquid cleaner which, applied to brake lining, causes it

Sent out by manufacturers of automotive products. The editor will gladly put readers in touch with makers mentioned

to lose its glaze. The maker claims this will cause brakes to equalize themselves and give better service. The cleaner comes in a squirt can and is sold on a guarantee basis.

For Hardened Valve Seats

With hardened valve seats coming into extensive use, Albertson & Co. has brought out a Sioux dual action grinder which will grind the hardest valve seats made to a perfect mirror finish. Accuracy is within .001 of an inch without any need for delicate adjustments. The complete grinder consists of a high-speed driver, a grinding wheel holder and a dressing tool.

Heat-Resisting Expander

American Hammered Piston Ring Co. has a piston expander made of Crovanite, which the maker refers to as a “new super-heat-resisting metal.” Six sizes take care of all requirements. It can be installed with a screw driver. An exclusive feature is full length expansion of the piston skirt on the thrust side.

Water Pump Packing

Aluminum Industries, Inc., has a new Permite packing for all types of water pumps. It is made of long asbestos fibre filling, with reinforcing lead strands forming a core, which is covered with a tight-fitting case of whip braid. This is pressed to shape and impregnated with a water-resisting lubricant and natural graphite. It is claimed that it will not harden in service and is not affected by any anti-freeze.

Fuel Pump Analyzer

A fuel pump analyzer, developed by the AC Spark Plug Co., combines a flow and pressure test which gives an accurate analysis of a fuel pump under actual operating conditions without removing the pump from the engine. List price is \$7.

Atmosphere-Proof Hoist

With no change in prices, an improved spur-gear chain hoist has been brought out by the Wright Mfg. Division of American Chain Co., Inc. All exposed parts are now zinc-coated. Precision ball bearings with integral grease seals support all moving parts and increase efficiency 10 per cent.

Gauges for the Front End

Bear Mfg. Co. has a device for gaging turning radius correctly. It weighs about 10 lb. and is portable. A pair is necessary, one for each wheel.

Bear also has a combination gage which checks both toe-in and camber and which is adjustable to all wheels and fits any tire.

Dual-Type Wrenches

Wrenches which combine two types of openings—box and open end—have been introduced by The Herbrand Co. They are drop-forged from chrome vanadium steel. Both ends have the same size openings. The box end has a 12-point opening.

Oil-Spray Air Cleaner

Industrial Air Cleaner Company has an oil-spray type of air cleaner. The dirt settling compartment is furnished in various sizes to meet requirements.

For Sticky Valve Stems

Kapolene is a lubricating chemical compound furnished by Klemm Automotive Products Co. Added in the proportion of one to three parts to the crankcase oil it prevents congealing at low temperatures, dissolves gummy accumulations on valve stems and piston rings.

We Repeat an Offer

Last month a great many readers wrote in for the valuable publications listed below. We are repeating this free offer. Check any or all of the books and mail the order blank to the Salvage Department, Commercial Car Journal, Philadelphia, Pa.

Please send me the following

Free Books

- ☐ 13. Operators' Tire Handbook
- ☐ 14. Motor Mechanics' Handbook
- ☐ 15. Piston Ring Manual
- ☐ 16. Spark Plug Chart
- ☐ 17. Executive Thinking
- ☐ 18. Demountable Body Equipment
- ☐ 19. Counter & Recorder Catalog
- ☐ 20. Silent Gear Catalog
- ☐ 21. South Bend Lathe Catalog

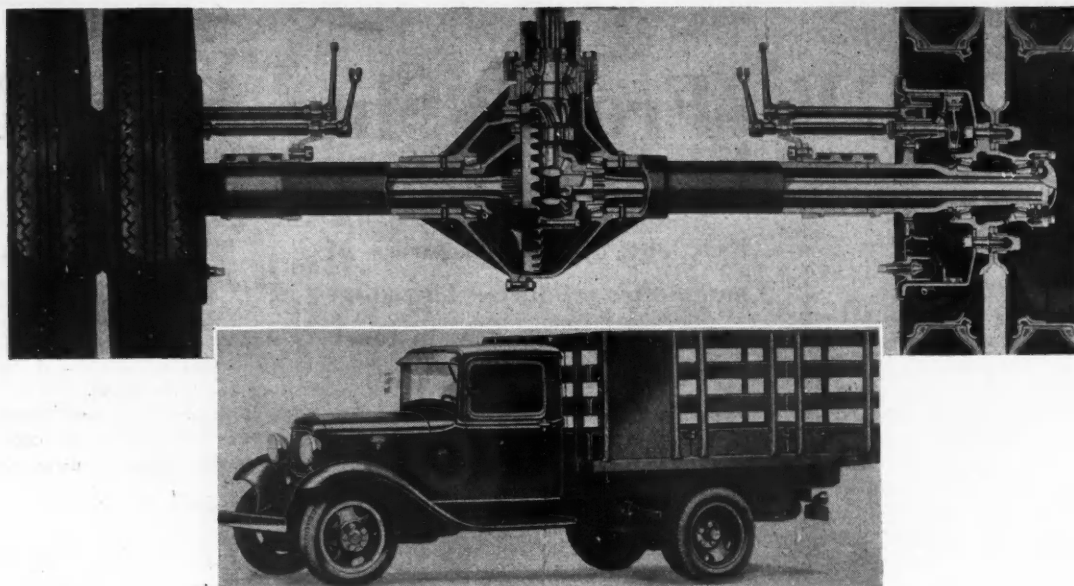
Name

Title

Firm Name

Address

City and State



Views of the new Ford truck and full-floating axle

Ford Bolsters Its Bid for Truck Volume With Full-Floating Rear

THE principal features of the new Ford trucks for 1934 include a new full-floating rear axle and a number of changes made to make the V-8 engine especially suitable for truck work. The forward end of the truck has been re-styled with a new chrome-plated moulding around the radiator grille, chrome-plated hood hinge rod and ornament for the radiator filler cap.

The new rear axle is more heavily constructed throughout than formerly with ample torque capacity to handle the full output of the V-8 engine. The pinion shaft is larger with larger bearings and the teeth on pinion and ring gear are wider than in the former $\frac{3}{4}$ -floating axle. The straddle-mounted driving pinion and ring gear thrust plate, two Ford axle features for some years, are continued, the latter with an enlarged surface. The straight type rollers in the straddle-mounting for the driving pinion are now solid, giving them a higher

New axle is more heavily constructed with ample torque capacity to handle V-8 output

load-carrying capacity than the former spiral flat wire type.

Provision has been made for positive lubrication to the pinion bearings. The centrifugal action of the ring gear pushes lubricant through a passage leading between the two taper pinion bearings. A similar duct acts as a return canal for the circulating lubricant.

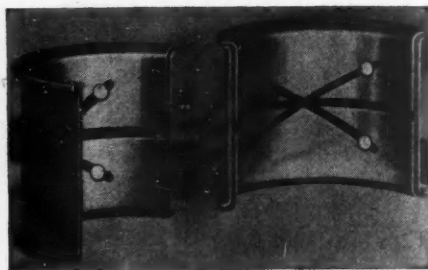
A new design feature is embodied in the differential bearing mounting. The taper roller bearings now are located equidistant from the centerline of the ring gear, thus providing equal distribution of stresses between the two bear-

ings and equalizing bearing wear. The new design also embodies a more rugged conically shaped differential case heavily ribbed inside to decrease deflection of the ring gear and a heavier housing for the differential assembly. To accommodate the full-floating axle shaft, the differential bearings are larger in diameter. The axle shaft tubes are of thicker material and the strength of the spring seat mountings has been increased.

The axle is especially designed for use with dual tire equipment. The taper roller bearings in the wheels are located directly under the load center when this equipment is mounted.

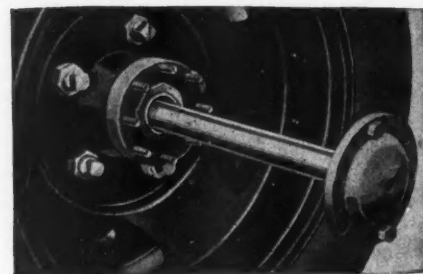
The driving flanges on the axle shafts are forged integral. Where the shafts slip into the differential side-gears 16 spline fittings are used with the shafts upset to eliminate localized stresses.

The V-8 truck engine is equipped with the new dual downdraft carburetion system. (TURN TO PAGE 68, PLEASE)

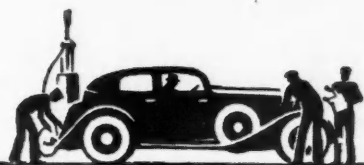


Left—The Ford "floating" connecting rod bearings are now surfaced with a high lead-bronze material

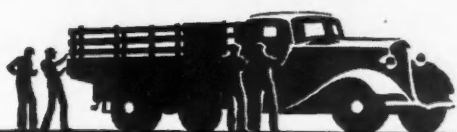
Right—Axle shafts may be readily removed without jacking up the wheels



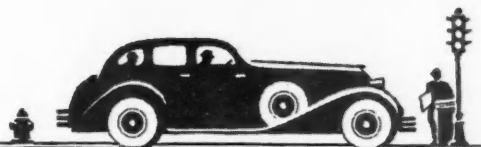
CHOSEN



by 35.7% of all car builders



by 74.4% of all truck builders



by more than a million
American motorists!

Lockheed Hydraulic brakes enter their eleventh year of service more firmly entrenched in the respect and approval of American car and truck owners than ever before.

Their principle of permanently, perfectly equalized pressure at all four wheels, their simplicity, their ease of adjustment and the infrequency of any need for it . . .

these things have won more than a million American motorists to Lockheed Hydraulic brakes.

Their easy factory installation, their excellent service record and their wide public approval, have gained for these fine brakes the endorsement and adoption by 35.7% of all car builders and 74.4% of all truck builders.

HYDRAULIC BRAKE COMPANY
DETROIT, MICHIGAN

LOCKHEED HYDRAULIC *Four* BRAKES *Wheel*

OFFICIALLY SERVICED THROUGHOUT THE NATION BY WAGNER ELECTRIC CORPORATION

THE COMMERCIAL CAR JOURNAL

APRIL, 1934

COMMERCIAL CAR JOURNAL NEWS

Truck Output Gains 140%

Truck production figures just released by the U. S. Bureau of Census show a gain of 140 per cent in the first two months of 1934 over 1933. U. S. and Canada output was 93,257 against 37,867. February production was 46,566 units, a gain of nearly 200 per cent over the 15,669 produced in February of last year.

The figures show a substantial improvement in production of trucks of more than two tons, which indicates that heavy-duty types are sharing in the recovery.

Mack May Sell Fords

At the annual meeting of stockholders of Mack Trucks, Inc., A. J. Brosseau, president, said considerable improvement had developed in the heavy-duty truck business. He added that negotiations were proceeding with the Ford Motor Company on an arrangement whereby Ford trucks would be sold and serviced through Mack Trucks branches, but no immediate decision was in prospect. E. R. Hewitt and W. D. Sargent were elected directors.

Groves Gets Autocar Control

Control of the Autocar Co., Ardmore, Pa., has passed into the hands of Wallace Groves of New York City, who, it is understood, acquired 60 per cent of the 200,000 outstanding shares of Autocar stock. Mr. Groves is described as an independent operator with no bank or brokerage connections. At the recent annual stockholders' meeting Mr. Groves spoke optimistically of the opportunities in the truck industry, of the future of the Autocar Co. and endorsed the management and policies of President Robert P. Page, Jr. He was elected to the board of directors.

Senate Gets Truck Bill

Senator Clarence C. Dill (D., Wash.) has introduced in the Senate the bills prepared by Joseph B. Eastman, Federal Coordinator of Transportation, proposing regulation of bus and motor freight transportation (S. 3171) and regulation of water transportation (S. 3172). They have been referred to the Committee on Interstate Commerce.

Dodge Sales Jump 705%

Dealers' retail deliveries of Dodge trucks during the first 11 weeks of 1934 totaled 7400 units, as against 919 units delivered during the corresponding eleven weeks of 1933—an increase of 705.2 per cent. Deliveries for the week ending March 17 alone were 952 Dodge commercial cars and trucks, according to J. D. Burke, director of truck sales.

March White's Biggest Since 1930

Orders received by The White Co. for the first three months show an increase of 150 per cent over the corresponding months of 1933. March was the biggest

month for the company since March, 1930, more than 800 orders being received from 60 distinct types of business.

Operators Add Diesels

As an addition to the growing list of fleet operators who have adopted its diesel engine, the Cummins Engine Co. announces that the U. S. Truck Co. of Detroit, has joined the family.

This company recently installed its first Cummins model HA-4 cyl. diesel in a heavy-duty Mack and has placed an order for 12 more.

Pacific Freight Lines of Los Angeles, which has been operating 21 units equipped with Cummins diesels has ordered 50 more. Transamerican Freight Lines of Detroit, with 11 Cummins diesel-powered vehicles, has ordered 20 more.

Reo Shipments Up 335%

Reo Motor Car Co. shipments of speed wagons, trucks and buses during the first quarter of 1934 have been three and one-third times as great as during the first quarter last year, according to Elijah G. Poxson, general sales manager.

Herrington Beters 1933

In the first three months of this year, the Marmon-Herrington company booked more actual business than in the entire 12 months of 1933, reports Vice-President Bert Dingley. All of the orders are for all-wheel-drive equipment. In 1933 the company sold more than six times as many units as in 1932.

Hercules Back on Coast

Hercules Motors Corp. has re-established direct factory representation on the Pacific Coast with the moving of Oliver S. Kelly to headquarters in San Francisco.

Kleiber Handles Studebaker

W. H. Edwards, general manager of the Studebaker Truck Division, announces that Kleiber Motor Co. of San Francisco and Oakland, will distribute Studebaker trucks in nine counties of California. Kleiber will continue to manufacture and sell its heavy-duty diesel-engine trucks.

Ford Passes Underwriters

Sharp reduction in the cost of fire fighting equipment for cities up to 25,000 population was indicated with the announcement that the new Ford V-8 truck had passed all tests prescribed by the National

Board of Fire Underwriters and is listed among accepted driving units for a pump capable of delivering 500 gal. of water a minute for an extended period. Ford is the first in its class to pass the tests.

Goodrich Has New Tire

A newly developed truck and bus tire is announced by The B. F. Goodrich Co. Known as the Goodrich Triple Protected Silvertown, it has three new protection features, which the maker designates as Plyflex, Ply-lock and 100 per cent Full-Floating Cords.

AC To Make Bearings

The AC Spark Plug Co. is installing equipment for the production of bronze-lined, steel-backed engine bearings, preparatory to entering the bearing business. Production is expected to start June 1.

Dodge Adds to Truck Staff

Dodge has made the following truck representative appointments: E. F. Drew in the North Jersey district, and W. C. Price in the Scranton district of the New York region; W. C. Pollard in the Evansville district, and J. E. Conley in the Huntington and Columbus districts of the Cincinnati region; M. J. Herold in the Baltimore district of the Philadelphia region; L. F. Turney in the Abilene, San Antonio and Amarillo districts of the Dallas region; E. A. Granroth in the Erie district of the Pittsburgh region.

Ed Lowe on S.A.E. Staff

Edward F. Lowe has been appointed assistant general manager and C. B. Whittelsey, Jr., assistant secretary of the Society of Automotive Engineers.

Ed Lowe, well known to fleet men and truck factory executives, was one of the organizers of the Monarch Governor Co. and pioneered development of the automatic type of governor in the truck field. Later he became interested in K. P. Products Co. as general manager. After K. P. Products merged with Handy Governor Corp., he became vice-president in charge of sales of Handy.

Balser and Jamison Named

Appointment of F. W. Balser and John T. Jamison to the field sales force of the Federal Motor Truck Co., is announced by J. F. Bowman, vice-president in charge of sales.

Herrick Resigns; Reed On

W. C. Herrick has resigned as Mack branch manager in the Cleveland district. Leonard F. B. Reed has been made his successor, with F. V. Fullem, assistant branch manager.

Gilbert Goes to Thornton

Frederick G. Gilbert, former Timken De- (TURN TO PAGE 68, PLEASE)

COMMERCIAL CAR JOURNAL'S

TRUCK SPECIFICATIONS TABLE

The Commercial Car Journal's Truck Specifications Table is brought up to date in each issue from data supplied monthly by truck manufacturers

KEY TO ABBREVIATIONS AND REFERENCE MARKS

GENERAL

Chassis Price—Chassis price quoted applies to the standard wheelbase and specifications listed. All prices are F.O.B. factory.

***—List price not yet established. Ready next issue.

Tonnage Rating—Where a spread of ratings is given the maximum ratings are for ideal operating conditions and the minimum for extremely difficult conditions; the ranges between are for varying operating conditions.

Gross Vehicle Weight—Is chassis weight, plus body and cab, plus payload. Gross vehicle weight given for a model is based on maximum recommended tire size and not on tires listed as standard equipment.

Chassis Weight Stripped—Includes gas, oil and water and all things included in chassis price. Does not include the weight of cab.

Maximum Brake H. P. at Given R.P.M.—Is actual dynamometer reading without accessories.

Tractors—Unless given the designation N (meaning not available as tractor), all standard models may be assumed to be available as tractor.

(A) All Torque and Brake Horsepower values listed are based on engine outputs with all Standard Equipment Accessories running and are the same values obtained with the truck on the road in actual operation.

(N) Not available as tractor.

(T) This designation accompanying a model number indicates vehicle is specifically designed for tractor use only.

c. o. e.—Cab-over-engine design.

(3) Corbitt—Larger engines and corresponding auxiliary units provided on all models at extra cost.

(4) Day Elder—Model 75—1½ ton—same specifications except price—\$945, and larger tire size—B6.00/20 front and DB6.00/20 rear.

(5) Dodge—F-61 available as special tractor truck with 146-inch wheelbase with model designation of F-60, at \$2645. K-61 available as special tractor truck with 146-inch wheelbase with model designation of K-60, at ***.

(5a) Dodge—Model H20.¼-1 ton, gross vehicle weight 6,000 lb., price \$502, has same specifications as H30 except tires which are 7.50/17 and lighter rear springs.

(6) General Motors—Models T-18 to T-61 inclusive are also available for export only as coach chassis. Double reduction axles optional in Models T-43, T-43T, T-51, T-61, T-83 and T-95 at extra cost. Trailing type axles available on Model T-95 at price deduction. Optional size engines available on Models T-85, T-85H, T-95, T-110, and T-130 at varying cost.

Gramm—Larger engines and corresponding auxiliary units provided on all models at extra cost when type of service demands. Wheelbases and body mounting dimensions may change to suit special requirements. Double reduction axles available on all models except AX and BX.

Gross weight indicated for each model in the table is the straight rating.

Series CXH is supplied with Hercules JXB engine in Model CXHB and Hercules JXC in Model CXHC.

(7) Grass Premier—Eight cylinder engines available on following models: 835 with Lye. GU at \$1515 list; 865 with Lye. HF at \$4230; 875 with Lye. AE at \$5400.

(8) International Harvester—A-1, ¾ ton, same as A-2 except less spring leaves and smaller tires.

(9) Le Moon—Model 600 available with Lye. AEC at same cost. Models 701 and 801 available with Waukesha 68RL at same cost.

(10) Sterling—Rocker arm used in place of springs.

(*) Sterling—These models also available equipped with Cummins Model H Diesel engine.

†Reo—Model 1D is the longer wheelbase edition of Model 1B. The frame dimension is 7x2¼x4. It is furnished at extra cost.

††Reo—2J, 2K same as 2H except 166 in wheelbase and price of \$1695

††Reo—3J same as 3H except wheelbase of 170 in. and price of \$2085; 3K same as 3H except 185 in. wheelbase and price of \$2155. 3M same as 3H except 205 in. wheelbase.

(11) Studebaker—S-2 in 141 in. and 165 in. wheelbases has 6¼ in. frame depth.

(12) White—Each model shown is furnished with different specifications for different tonnage ratings.

•—Factory governed speed 2400 r.p.m.

(13) Marmon-Herrington—Available with Hercules Diesel engine. Price on application.

(14) Ford—Rear axle ratios 5.14 and 6.6 optional on 1½-ton trucks.

(15) Mack—Chassis price and weight include cab.

MAKES—ALL

AB—American Bosch.
A LaF—American La France.
AL—Auto Lite.

B—Bendix.
BB—Borg & Beck.
BL—Brown-Lipe.
BO—Bendix front, Own rear.

Bio—Blood.
Bu or Bud—Buda.
BW—Borg Warner
BWS—Bendix front, Westinghouse rear.

Ca—Carter.
Ch—Chicago.
CI—Ignition by compression.

Cl or Cla—Clark.
Cle—Cleveland.
Co—Covert (transmission).

Co—Covert (clutch).
Con—Continental.
Cot—Cotta Gear.

Cum—Cummins-Diesel.
Det—Detroit Lubricator.
DG—Detroit Gear and Machine.

DR—DeLoe Remy.
Eat—Eaton.
Ei—Elsemann.

En—Governor built in engine.
EV—Electro-Vac (gov.) Pierce.
Fe—Fedders.

Fu—Fuller.
Ge—Gemmer.
GO—G. & O.

Ha—Handy (governor).
Ha—Hannum (steering gear).
HaS—American Car & Fdry.

Her—Hercules.
Hr—Harrison.
HS—Merchant & Evans (clutch).

HS—American Car & Fdry. (governor).
Jac—Saginaw.
Jo—Jones.

KP—Handy.
L—Lockheed.
Li—Lipe, W. C.

LN—Leece Neville.
Lo—Long.
LO—Lockheed front, Own rear.

LW—Lockheed front, Wisconsin rear.
Lyc—Lycoming.
Mc—McCord.

Ma—Marvel.
ME—Merchant & Evans.
MM—Mechanics Mach.

Mo—Modine (radiator).
Mo—Monarch (governor).
My—Mallory.

NE—North East.
No—Not supplied.
ns—No Standard.

O or Ow—Own.
Op or Opt—Optional.
Pe—Pierce (governor).

Pe—Perfex (radiator).
PS—Peters & Sneed.
RB—Robt. Bosch.

Ro—Rossford.
Ros—Ross.
Sc—Scintilla.

Sch—Wheeler-Schebler.
Shu—Shuler.
SpB—Spicer and Blood.

SpI—Spicer.
St or St—Sterling.
St—Stromberg.

Til—Tillotson.
T or Tim—Timken.
TWH—Timken Wisconsin Herrington.

WG—Warner Gear.
Wa—Waukesha (governor).
Wau—Waukesha.

W or Wis—Wisconsin.
Ws—Westinghouse.
Yo—Young.

Zen—Zenith.

BRAKES—SERVICE

Location

2—Two Wheels, rear only.
2/4—Two-wheel brakes effective on all four wheels through driveshaft.
4/6—Brakes on four rear wheels effective on all wheels through driveshaft.
T/4—Brake on transmission effective on all four wheels through driveshaft.
4—Four Wheels, front and rear.
4r—Four Wheels, rear only.
6—Six Wheels, front and rear.
J—Jackshaft.
P—Propeller shaft.

Type

I—Internal.
X—External.

Operation

A—Air.
D—Hydraulic and mechanical.
H—Hydraulic.
M—Mechanical.
V—Vacuum.

BRAKES—HAND

Location

C—Center of double propeller shaft.
2—Rear wheels.
4—Four wheels.
R—Worm or bevel gearshaft.
T—Transmission.
F—Driveshaft.

Type

D—Tru-Stop disk.
I—Internal.
X—External.

BRAKE DRUMS

Material

s—Cast alloy iron.
A—American Car Fdry.
C—Centrifuge
D—Dayton.
E—Ermalite.
G—Gunite.
H—Hunt Spiller.
c—Cast iron.
p—Pressed steel.
P—Pressed steel.
s—Cast steel.
(Where a combination of any of the above is used, the first reference mark applies to the front and the second to the rear drums.)

CLUTCH

Type

D—Multiple disk.
dp—Double plate.
O—Plate in oil.
P—Single plate.

ENGINE

Valve Arrangement

F—Inlet valve in head; exhaust valve at side.
H—In head.
L—"L" head, valves at side.
T—Inlet and exhaust on opposite sides.

Camshaft Drive

C—Chain.
G—Gear.

Piston Material

A—Aluminum alloy.
B—Semi-steel.
C—Cast iron.
N—Nickel iron.
S—Aluminum alloy with strut.

Main Bearings

r—Rear main bearing.

Oiling System

CC—Pressure to main, connecting rod and camshaft bearings.

FP—Pressure to main, connecting rod camshaft bearings and piston pins.
PC—Pressure to mains and connecting rod bearings.
PQ—Pump, gravity and splash.
PS—Pressure with splash.

FRAME

Type

I—"I" Beam.
C—Channel.
T—Channel tapered front and rear.
L—Channel reinforced with liner.
B—Channel reinforced with both liner and fishplate.
P—Channel reinforced with plate.
TL—Channel tapered front and rear reinforced with liner.
D—Drop Center.
TF—Tapered front.
X—X-Braced.

FUEL SYSTEM

Fuel Feed

E—Electric pump.
G—Gravity.
M—Mechanical pump.
P—Pressure.
V—Vacuum.
B—Bosch.
C—Cummins.

REAR AXLE

Final Drive and Type

B—Bevel.
C—Chain.
D—Dead.
F—Full-floating.
2—Double Reduction.
S—Spiral bevel.
W—Worm.
w/2—Worm or Double Reduction Optional.
¼—Semi-floating.
¾—Three-quarter floating.

Drive and Torque

A—Radius Rods and Torque Arm.
H—Hotchkiss (springs).
R—Radius Rods.
T—Torque Arm.
U—Torque Tube.

SPRINGS

Auxiliary Type

¼—Semi-elliptic above or below main springs.
¼—Quarter elliptic.
C—Coil spring.
N—No.
O—Optional.

TIRES

B—Balkon.
DB—Dual Balloons.
P—High Pressure Pneumatics.
DP—Dual High Pressure Pneumatics.
S—Solids.
DS—Dual Solids.
•—Pneumatics at extra cost.

TRANSMISSION

Location

A—Amidships.
J—Unit with jackshaft.
U—Unit with engine.

Auxiliary Location

No—Not furnished.
O2—2 speed axle unit optional at extra cost.
Op—Optional at extra cost.
A—Amidships.
R—Rear of amidships main transmission.
U—Unit with engine.

WHEELS DRIVEN

2C—Center pair of rear wheels.
2R—Rear pair of rear wheels.
4F—Front and center pair of rear wheels.
4R—Four rear wheels.
6—Six wheels.

Line Number	MAKE AND MODEL	GENERAL (See Keynote)				TIRE SIZE		MAJOR UNITS										FRAME																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																								
		Tonnage Rating	Chassis Price	Standard Wheelbase	Max. W. B. Furnished	Gross Vehicle Weight	Chassis Wt. (Stripped)	Front	Rear	ENGINE		TRANSMISSION		REAR AXLE		Type																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																										
										Make and Model	No. of Cylinders Bore and Stroke	Make and Model	Location and Forward Speeds	Location and Forward Speeds	Make and Model		Gear and Type	Drive and Torque	GEAR RATIOS																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																							

Line Number	ENGINE DETAILS										FUEL SYST.	ELEC-TRICAL	FRONT AXLE	BRAKES		BODY MOUNT-ING DATA		SPRINGS		Auxiliary Type											
	Piston Displacement	Compression Ratio	Torque lb. ft.	N.A.C.C. Rated H.P.	Max. Brake H.P. at R.P.M. Given	Valve Arrangement	Cams Shaft Drive	Piston Material	MAIN BEARINGS					Governor Make	Carburetors Make	Fuel Feed	Ignition System Make	Generator, Starter Make	Clutch Type and Make		Radiator Make	Universal Make	Make and Model	Steering Gear Make	SERVICE		Hand Location, Type	Cab to Rear of Frame	Cab to Rear Axle	Width of Frame	Front
									Number and Diameter	Length	Oiling System Type	Make, Location Type, Operation	Lining Area												Drum Material						
1468	4.4	322	43	120	2200	H	C	4-7-3/4	10 1/2	CC	Ha	Zen	VDR	DR	P.B.L	Lo	Spl	Tim	27451	Ros	O41A	720	A	CD	172	102	33 1/2	42x3	56x4		
2707	4.4	500	60	175	2200	H	H	4-7-3/4	14 1/2	CC	Ha	Zen	MDR	DR	dp.Lo	Lo	Spl	Tim	27451	Ros	O41A	720	A	CD	172	102	33 1/2	42x3	56x4		
4248	5.0	150	27	35	2600	H	C	4-7-3/4	14 1/2	CC	Ha	Zen	MDR	DR	dp.Lo	Lo	Spl	Tim	27461	Ros	O41A	720	A	CD	172	102	33 1/2	42x3	56x4		
5339	4.7	225	38	73	2200	H	C	4-7-3/4	13 1/2	PC	Mo	Str	MAL	AL	D.B.B	Yo	Spl	Tim		Ros	L41H	380	G	TX	129 1/2	Opt	31 1/2	40x2 1/2	50x3		
6339	4.7	225	38	73	2200	H	C	4-7-3/4	13 1/2	PC	Mo	Str	MAL	AL	D.Fu	Yo	Spl	Tim		Ros	L41HV	452	G	TX	106	Opt	31 1/2	40x2 1/2	50x3		
7339	4.7	225	38	73	2200	H	C	4-7-3/4	13 1/2	PC	Mo	Str	MAL	AL	D.Fu	Yo	Spl	Shu		Ros	L41HV	658	G	TX	106	Opt	31 1/2	41x2 1/2	62 1/2 x 3		
8360	4.7	238	40	90	2200	H	C	4-7-3/4	15	PC	Mo	Str	MAL	AL	D.Fu	Yo	Spl	Shu		Ros	L41HV	768	H	TX	106	Opt	31 1/2	41x2 1/2	62 1/2 x 3		
9529	4.7	355	51	115	2200	H	C	4-7-3/4	15	PC	Mo	Str	MAL	AL	D.Fu	Yo	Spl	Shu		Ros	L41HV	893	H	TD	93 1/2	Opt	31 1/2	41x2 1/2	62 1/2 x 3		
10428	4.4	280	45	93	2200	H	C	4-7-3/4	14 1/2	PC	Mo	Str	MAL	AL	D.Fu	Yo	Spl	Shu		Ros	L41HV	893	H	TD	93 1/2	Opt	31 1/2	41x2 1/2	62 1/2 x 3		
11478	4.4	315	51	103	2200	H	C	4-7-3/4	15	PC	Mo	Str	MAL	AL	D.Fu	Yo	Spl	Shu		Ros	L41HV	893	H	TD	93 1/2	Opt	31 1/2	41x2 1/2	62 1/2 x 3		
12529	4.4	355	51	115	2200	H	C	4-7-3/4	15	PC	Mo	Str	MAL	AL	D.Fu	Yo	Spl	Shu		Ros	L41HV	893	H	TD	93 1/2	Opt	31 1/2	41x2 1/2	62 1/2 x 3		
13314	5.5	213	33	75	2400	H	C	4-7-3/4	12 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	31000	Ros	LO41DV	450	c	21	88 1/2	60 1/2	34 1/2	42 1/2 x 3	54x3	
14358	5.5	240	38	84	2500	H	C	4-7-3/4	12 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	33000	Ros	LO41DV	450	c	21	88 1/2	60 1/2	34 1/2	42 1/2 x 3	54x3	
15358	5.5	240	38	84	2500	H	C	4-7-3/4	12 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	35000	Ros	LO41DV	519	c	21	88 1/2	60 1/2	34 1/2	42 1/2 x 3	54x3	
16358	5.5	240	38	84	2500	H	C	4-7-3/4	12 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	35000	Ros	LO41DV	519	c	21	88 1/2	60 1/2	34 1/2	42 1/2 x 3	54x3	
17404	5.5	240	38	84	2500	H	C	4-7-3/4	12 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	35000	Ros	LO41DV	519	c	21	88 1/2	60 1/2	34 1/2	42 1/2 x 3	54x3	
18404	5.5	240	38	84	2500	H	C	4-7-3/4	12 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	35000	Ros	LO41DV	519	c	21	88 1/2	60 1/2	34 1/2	42 1/2 x 3	54x3	
19404	5.5	240	38	84	2500	H	C	4-7-3/4	12 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	26450	Ros	LO41DV	543	c	FD	168 1/2	102 1/2	34 1/2	42 1/2 x 3	53x3	
20453	5.5	309	48	101	2400	H	C	4-7-3/4	14 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	27450	Ros	LO41DV	660	c	FD	115 1/2	71 1/2	34 1/2	41 1/2 x 3	54 1/2 x 4	
21453	5.5	309	48	101	2400	H	C	4-7-3/4	14 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	27450	Ros	LO41DV	660	c	FD	115 1/2	71 1/2	34 1/2	41 1/2 x 3	54 1/2 x 4	
22453	5.5	309	48	101	2400	H	C	4-7-3/4	14 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	27450	Ros	LO41DV	660	c	FD	115 1/2	71 1/2	34 1/2	41 1/2 x 3	54 1/2 x 4	
23453	5.5	309	48	101	2400	H	C	4-7-3/4	14 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	27450	Ros	LO41DV	660	c	FD	115 1/2	71 1/2	34 1/2	41 1/2 x 3	54 1/2 x 4	
24453	5.5	309	48	101	2400	H	C	4-7-3/4	14 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	27450	Ros	LO41DV	660	c	FD	115 1/2	71 1/2	34 1/2	41 1/2 x 3	54 1/2 x 4	
25358	5.5	240	38	84	2500	H	C	4-7-3/4	12 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	35000	Ros	LO41DV	519	c	21	88 1/2	60 1/2	34 1/2	42 1/2 x 3	54x3	
26358	5.5	240	38	84	2500	H	C	4-7-3/4	12 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	35000	Ros	LO41DV	519	c	21	88 1/2	60 1/2	34 1/2	42 1/2 x 3	54x3	
27404	5.5	240	38	84	2500	H	C	4-7-3/4	12 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	26450	Ros	LO41DV	543	c	FD	116	71 1/2	34 1/2	41 1/2 x 3	53x3	
28404	5.5	240	38	84	2500	H	C	4-7-3/4	12 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	26450	Ros	LO41DV	543	c	FD	116	71 1/2	34 1/2	41 1/2 x 3	53x3	
29404	5.5	240	38	84	2500	H	C	4-7-3/4	12 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	26450	Ros	LO41DV	543	c	FD	116	71 1/2	34 1/2	41 1/2 x 3	53x3	
30453	5.5	309	48	101	2400	H	C	4-7-3/4	14 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	26450	Ros	LO41DV	543	c	FD	116	71 1/2	34 1/2	41 1/2 x 3	53x3	
31453	5.5	309	48	101	2400	H	C	4-7-3/4	14 1/2	FP	OW	Str	MAL	AL	DR	dp.Lo	GO	Spl	Tim	27450	Ros	LO41DV	543	c	FD	116	71 1/2	34 1/2	41 1/2 x 3	53x3	
32453	5.5	165	29	73	3000	H	C	4-7-3/4	10 1/2	FP	Wa	Zen	MAL	AL	P.L	Ch	Blo	Tim	30000H	Ros	L41H	330	a	TX	Opt	32	42 1/2 x 2 1/2	58x2 1/2			
33453	5.5	165	29	73	3000	H	C	4-7-3/4	10 1/2	FP	Wa	Zen	MAL	AL	P.L	Ch	Blo	Tim	30000H	Ros	L41H	330	a	TX	Opt	32	42 1/2 x 2 1/2	58x2 1/2			
34282	5.5	188	33	85	3200	H	C	4-7-3/4	10 1/2	FP	Wa	Zen	MAL	AL	P.L	Ch	Blo	Tim	30000H	Ros	L41H	330	a	TX	Opt	32	42 1/2 x 2 1/2	58x2 1/2			
35358	5.5	254	39	110	2800	H	C	4-7-3/4	12 1/2	FP	Wa	Zen	MAL	AL	P.L	Ch	Blo	Tim	30000H	Ros	L41H	330	a	TX	Opt	32	42 1/2 x 2 1/2	58x2 1/2			
36358	5.5	254	39	110	2800	H	C	4-7-3/4	12 1/2	FP	Wa	Zen	MAL	AL	P.L	Ch	Blo	Tim	30000H	Ros	L41H	330	a	TX	Opt	32	42 1/2 x 2 1/2	58x2 1/2			
37462	5.5	324	45	125	2800	H	C	4-7-3/4	12 1/2	FP	Wa	Zen	MAL	AL	P.L	Ch	Blo	Tim	30000H	Ros	L41H	330	a	TX	Opt	32	42 1/2 x 2 1/2	58x2 1/2			
38215	5.0	142	27	72	3000	H	C	4-7-3/4	6 1/2	CC	No	Zen	MAL	AL	P.L	Ch	Blo	Tim	31000H	Ros	L41H	330	a	TX	Opt	32	42 1/2 x 2 1/2	58x2 1/2			
39215	5.0	142	27	72	3000	H	C	4-7-3/4	6 1/2	CC	No	Zen	MAL																		

Line Number	MAKE AND MODEL	GENERAL (See Keynote)				TIRE SIZE		MAJOR UNITS				FRAME							
		Tonnage Rating	Chassis Price	Standard Wheelbase	Max. W. B. Furnished	Gross Vehicle Weight	Chassis Wt. (Stripped)	Front	Rear	ENGINE		TRANSMISSION		REAR AXLE		Type			
										Make and Model	No. of Cylinders Bore and Stroke	Make and Model	Location and Forward Speeds	Make and Model	Gear and Type		Drive and Torque	Gear Ratios In High In Low	Side Rail Dimensions
1	Dodge Bros. KCL (Concluded)	Com'l	395	119	119	1815	B5.25/17	B5.25/17	Own	6-3 1/2 x 4 1/2	Own	U3	No	Own	SF	H4.11 11.55	x2 x x x	X	
2	H20	1 1/2	502	131	157	6075	2667	B7.00/20	B7.00/20	Own	6-3 1/2 x 4 1/2	Own	U4	No	Own	SF	H4.87 31.26	x2 x x x	1201.5
3	K20	1 1/2	525	131	...	5500	2559	B6.00/20	P32x6	Own	6-3 1/2 x 4 1/2	Own	U4	No	Own	SF	H4.87 31.26	x2 x x x	2201.5
4	K20X	1 1/2	525	131	...	6075	2612	B6.00/20	P32x6	Own	6-3 1/2 x 4 1/2	Own	U4	No	Own	SF	H4.87 31.26	x2 x x x	2217.5
5	K22	1 1/2	490	140	...	7800	2710	B7.00/20	B7.00/20	Own	6-3 1/2 x 4 1/2	Own	U5	No	Own	SF	H5.66 40.08	x2 x x x	2242.5
6	H30	1 1/2	490	131	157	8400	2612	B6.00/20	P32x6	Own	6-3 1/2 x 4 1/2	Own	U4	No	Own	SF	H5.66 40.08	x2 x x x	2242.5
7	K30	1 1/2	525	131	...	8400	2667	B6.00/20	P32x6	Own	6-3 1/2 x 4 1/2	Own	U4	No	Own	SF	H4.87 31.26	x2 x x x	2242.5
8	KD-30	1 1/2	525	136	161	8400	2612	B6.00/20	P32x6	Own	6-3 1/2 x 4 1/2	Own	U4	No	Own	SF	H4.87 31.26	x2 x x x	2242.5
9	K32	1 1/2	525	136	161	10500	2885	B6.00/20	P32x6	Own	6-3 1/2 x 4 1/2	Own	U4	No	Own	SF	H5.66 36.27	x2 x x x	2242.5
10	KD32	1 1/2	525	136	161	10500	2885	B6.00/20	P32x6	Own	6-3 1/2 x 4 1/2	Own	U4	No	Own	SF	H5.66 36.27	x2 x x x	2242.5
11	K32	1 1/2	525	136	161	10500	2885	B6.00/20	P32x6	Own	6-3 1/2 x 4 1/2	Own	U4	No	Own	SF	H5.66 36.27	x2 x x x	2242.5
12	H33	1 1/2	795	136	165	11000	3350	B7.00/20	DB7.00/20	Own	6-3 1/2 x 4 1/2	Own	U4	No	Own	SF	H6.37 40.88	x2 x x x	2242.5
13	K35	1 1/2	845	140	169	12500	3580	B6.50/20	DB6.50/20	Own	6-3 1/2 x 4 1/2	Own	U5	No	Own	SF	H6.33 44.78	x2 x x x	2242.5
14	KZ35	1 1/2	845	140	169	12500	3780	B7.00/20	DB7.00/20	Own	6-3 1/2 x 4 1/2	Own	U5	No	Own	SF	H6.33 44.78	x2 x x x	2242.5
15	H43	2-3	795	136	165	11000	3350	B7.00/20	DB7.00/20	Own	6-3 1/2 x 4 1/2	Own	U5	No	Own	SF	H6.37 40.88	x2 x x x	2242.5
16	K45	2-3	845	140	169	12500	3675	B7.00/20	DB7.00/20	Own	6-3 1/2 x 4 1/2	Own	U5	No	Own	SF	H6.33 44.78	x2 x x x	2242.5
17	KZ45	2-3	845	140	169	12500	3780	B7.00/20	DB7.00/20	Own	6-3 1/2 x 4 1/2	Own	U5	No	Own	SF	H6.33 44.78	x2 x x x	2242.5
18	F40	2-3	1995	150	190	16000	5173	B6.50/20	DB6.50/20	Own	6-3 1/2 x 4 1/2	Own	U4	No	Own	SF	H6.37 43.79	x2 x x x	2242.5
19	K40	2-3	150	190	19000	5344	P32x6	DP32x6	Own	6-3 1/2 x 4 1/2	Own	U4	No	Own	SF	H6.37 43.79	x2 x x x	2242.5	
20	(5) F-61	3-5 1/2	2575	170	195	20000	5789	P32x6	DP32x6	Own	6-3 1/2 x 4 1/2	Own	U4	No	Own	SF	H7.12 48.88	x2 x x x	2242.5
21	(5) G-61	3-5 1/2	2575	170	195	20000	5789	P32x6	DP32x6	Own	6-3 1/2 x 4 1/2	Own	U4	No	Own	SF	H7.12 48.88	x2 x x x	2242.5
22	G-80	4-8	5250	146	220	25000	7640	B9.75/20	DB9.75/20	Own	8-3 1/2 x 5	Own	U5	No	Own	SF	H7.21 62.70	x2 x x x	2242.5
23	Duplex	S3	160	Op	15000	6000	B8.25/20	DB8.25/20	Bud K325	6-3 1/2 x 4 1/2	BL 2352	U5	No	Tim 65200	WF	H6.75 36.27	x2 x x x	2242.5	
24	SAC	5-8	172	Op	18000	7400	B9.75/20	DB9.75/20	Bud K428	6-4 1/2 x 4 1/2	BL 3353	U5	No	Tim 75733	WF	R.8.50 81.07	x2 x x x	2242.5	
25	K5	5-7	172	Op	21000	8000	B10.50/20	DB10.50/20	Bud L525	6-4 1/2 x 4 1/2	BL 5351	U5	No	Tim 76733	w2F	R.8.50 81.07	x2 x x x	2242.5	
26	M	5-7	168	Op	28000	10000	P34x7	DS36x7	Bud GL6	6-4 1/2 x 4 1/2	BL 70	A7	No	Tim 68700	WF	R.8.50 81.07	x2 x x x	2242.5	
27	Esco	2-3 1/2	2860	165	205	15000	5900	B7.50/20	DB7.50/20	Con E603	6-4 1/2 x 4 1/2	CL 105R	U5	No	Tim 6642	BF	H5.66 36.27	x2 x x x	2242.5
28	Fageol	1 1/2	1350	148	172	12000	4000	B6.00/20	DB6.00/20	Wau ZK	6-3 1/2 x 4 1/2	WG T9	U4	No	Tim 53200H	BF	H5.66 36.27	x2 x x x	2242.5
29	106BK	1 1/2	1700	161	195	11200	4800	B6.50/20	DB6.50/20	Wau 6BK	6-3 1/2 x 4 1/2	WG T9	U4	No	Tim 53200	BF	H5.66 36.27	x2 x x x	2242.5
30	106RA	1 1/2	1825	161	195	12700	4900	B6.50/20	DB6.50/20	Wau 6BK	6-3 1/2 x 4 1/2	WG T9	U4	No	Tim 54200	BF	H5.66 36.27	x2 x x x	2242.5
31	135HP	2-3	2250	161	195	13400	5400	B7.50/20	DB7.50/20	Wau 6-90	6-3 1/2 x 4 1/2	BL 234	U4	No	Tim 54200H	BF	H5.66 36.27	x2 x x x	2242.5
32	135RA	2-3	2400	161	195	15000	5600	B7.50/20	DB7.50/20	Wau 6-90	6-3 1/2 x 4 1/2	BL 234	U4	No	Tim 56200H	BF	H5.66 36.27	x2 x x x	2242.5
33	135SC	2-3	2150	161	210	14700	5100	B7.50/20	DB7.50/20	Wau 6-90	6-3 1/2 x 4 1/2	BL 234	U4	No	Tim 54200H	BF	H5.66 36.27	x2 x x x	2242.5
34	200HP	2-3	3000	178	196	16300	6500	B8.25/20	DB8.25/20	Wau 6-110	6-4 1/2 x 4 1/2	BL 524	U4	No	Tim 54200	BF	H5.66 36.27	x2 x x x	2242.5
35	250HP	2 1/2	3000	178	196	16300	6500	B8.25/20	DB8.25/20	Wau 6-110	6-4 1/2 x 4 1/2	BL 524	U4	No	Tim 56200H	BF	H5.66 36.27	x2 x x x	2242.5
36	250MS	2 1/2	2700	178	196	16300	6175	B8.25/20	DB8.25/20	Wau 6MS	6-3 1/2 x 4 1/2	BL 334	U4	No	Tim 56200H	BF	H5.66 36.27	x2 x x x	2242.5
37	250MK	2 1/2	2750	178	196	16300	6200	B8.25/20	DB8.25/20	Wau 6MK	6-4 1/2 x 4 1/2	BL 334	U4	No	Tim 56200H	BF	H5.66 36.27	x2 x x x	2242.5
38	250RA	2 1/2	3150	178	196	19500	6700	B8.25/20	DB8.25/20	Wau 6-110	6-4 1/2 x 4 1/2	BL 524	U4	No	Tim 56200H	BF	H5.66 36.27	x2 x x x	2242.5
39	250SC	2 1/2	2925	178	230	17500	6200	B8.25/20	DB8.25/20	Wau 6-110	6-4 1/2 x 4 1/2	BL 524	U4	No	Tim 56200H	BF	H5.66 36.27	x2 x x x	2242.5
40	300HP	3-5	3500	178	196	20700	7200	B9.00/20	DB9.00/20	Wau 6-110	6-4 1/2 x 4 1/2	BL 524	U4	No	Tim 56200H	BF	H5.66 36.27	x2 x x x	2242.5
41	300RA	3-5	3775	178	196	23300	7700	B9.00/20	DB9.00/20	Wau 6-110	6-4 1/2 x 4 1/2	BL 524	U4	No	Tim 56200H	WF	H5.66 36.27	x2 x x x	2242.5
42	370HP	5-6	5000	182	200	25300	9700	B9.75/20	DB9.75/20	Wau 6-125	6-4 1/2 x 4 1/2	BL 734	U4	No	Tim 65725H	WF	R.5.12 7.12	x2 x x x	2242.5
43	370SR	5-6	4850	182	200	25300	9500	B9.75/20	DB9.75/20	Wau 6SRK	6-4 1/2 x 4 1/2	BL 734	U4	No	Tim 65725H	WF	R.5.12 7.12	x2 x x x	2242.5
44	370RA	5-6	5250	182	200	31000	9950	B9.75/20	DB9.75/20	Wau 6-125	6-4 1/2 x 4 1/2	BL 734	U4	No	Tim 66720DH	WF	R.5.12 7.12	x2 x x x	2242.5
45	470HP	6-7	5500	182	200	33500	10100	B9.75/20	DB9.75/20	Wau 6-125	6-4 1/2 x 4 1/2	BL 734	U4	No	Tim 66720DH	WF	R.5.12 7.12	x2 x x x	2242.5
46	Federal	DM	975	120	120	8000	3050	B6.00/20	P32x6	Con W10	4-3 1/2 x 4 1/2	WG T9	U4	No	Tim 6374	SF	H5.67 38.20	x2 x x x	2242.5
47	15X	1 1/2	745	137	174	10000	3500	B6.00/20	P32x6	Her JXA	6-3 1/2 x 4 1/2	WG T9	U4	No	Tim 6374	SF	H5.67 38.20	x2 x x x	2242.5
48	18X	1 1/2	640	137	174	10000	3500	B6.00/20	P32x6	Her JXA	6-3 1/2 x 4 1/2	WG T9	U4	No	Tim 6374	SF	H5.67 38.20	x2 x x x	2242.5
49	20	1 1/2	845	137	187	11000	3800	P6.00/20	DB6.00/20	Her JXA	6								

[illegible]

THE

Line Number	ENGINE DETAILS				MAIN BEARINGS	Governor Make	Carburetors Make	Fuel Feed	Ignition System Make	Generator, Starter Make	Clutch Type and Make	Radiator Make	Universal Make	Make and Model	BRAKES		BODY MOUNT-ING DATA		SPRINGS											
	Piston Displacement	Compression Ratio	Torque lb. ft.	N.A.C.C. Rated H.P.											Valve Arrangement	Piston Material	Number and Diameter	Length	Oiling System Type	Steering Gear Make	Make, Location Type, Operation	Lining Area	Drum Material	Hand Location, Type	Cab to Rear of Frame	Cab to Rear Axle	Width of Frame	Front	Rear	Auxiliary Type
1428	4.1	26.8	45.9	100-2200	H	L	7-2 1/2	PC	Mo	No	MAL	AL	D.B.L	Pe	Blo	Tim 27450	Ros	W41A	576	a	FD	128 1/2	73 1/2	36	46x3	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2
2672	17.1	44.0	57.0	125-1800	H	L	7-3 1/2	PC	En	No	MAL	AL	D.B.L	Pe	Blo	Tim 27450	Ros	W41A	576	a	FD	128 1/2	73 1/2	36	46x3	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2
3611	4.1	38.2	54.1	127-2300	H	L	7-3 1/2	PC	En	No	MAL	AL	D.B.L	Pe	Blo	Tim 27450	Ros	W41A	576	a	FD	128 1/2	73 1/2	36	46x3	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2
4255	4.1	38.2	54.1	127-2300	H	L	7-3 1/2	PC	En	No	MAL	AL	D.B.L	Pe	Blo	Tim 27450	Ros	W41A	576	a	FD	128 1/2	73 1/2	36	46x3	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2
5555	4.1	38.2	54.1	127-2300	H	L	7-3 1/2	PC	En	No	MAL	AL	D.B.L	Pe	Blo	Tim 27450	Ros	W41A	576	a	FD	128 1/2	73 1/2	36	46x3	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2
6358	5.2	25.4	38.1	90-2200	F	F	7-2 1/2	PC	Wa	No	MAL	AL	D.B.L	Ch	Spi	Tim 3000H	Ros	L4IH	268	a	TD	Opt	72	31	40x2 1/2	54x3	54x3	54x3	54x3	54x3
7462	5.1	32.4	45.4	125-2600	F	F	7-3 1/2	PC	Wa	No	MAL	AL	D.B.L	Ch	Spi	Tim 3000H	Ros	L4IH	394	a	TD	Opt	86	34	40x3	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2
8255	4.6	18.2	27.3	90-3200	F	F	7-2 1/2	PC	Wa	No	MAL	AL	D.B.L	Ch	Spi	Tim 3000H	Ros	L4IH	574	a	TD	Opt	86	34	40x3	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2
9358	4.6	18.2	27.3	90-3200	F	F	7-2 1/2	PC	Wa	No	MAL	AL	D.B.L	Ch	Spi	Tim 3000H	Ros	L4IH	574	a	TD	Opt	86	34	40x3	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2
10358	4.6	18.2	27.3	90-3200	F	F	7-2 1/2	PC	Wa	No	MAL	AL	D.B.L	Ch	Spi	Tim 3000H	Ros	L4IH	574	a	TD	Opt	86	34	40x3	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2
11462	4.6	18.2	27.3	90-3200	F	F	7-2 1/2	PC	Wa	No	MAL	AL	D.B.L	Ch	Spi	Tim 3000H	Ros	L4IH	574	a	TD	Opt	86	34	40x3	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2
12462	4.6	18.2	27.3	90-3200	F	F	7-2 1/2	PC	Wa	No	MAL	AL	D.B.L	Ch	Spi	Tim 3000H	Ros	L4IH	574	a	TD	Opt	86	34	40x3	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2	58x3 1/2
13298	5.3	20.0	33.7	80-2800	L	L	7-3 1/2	PC	Pe	Zen	MAL	DR	P.B.B	Yo	Blo	Cla F318	Ros	L4IH	318	p	CD	81	61	31 1/2	41x2 1/2	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3
14298	5.3	20.0	33.7	80-2800	L	L	7-3 1/2	PC	Pe	Zen	MAL	DR	P.B.B	Yo	Blo	Cla F318	Ros	L4IH	318	p	CD	81	61	31 1/2	41x2 1/2	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3
15298	5.3	20.0	33.7	80-2800	L	L	7-3 1/2	PC	Pe	Zen	MAL	DR	P.B.B	Yo	Blo	Cla F318	Ros	L4IH	318	p	CD	81	61	31 1/2	41x2 1/2	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3
16298	5.3	20.0	33.7	80-2800	L	L	7-3 1/2	PC	Pe	Zen	MAL	DR	P.B.B	Yo	Blo	Cla F318	Ros	L4IH	318	p	CD	81	61	31 1/2	41x2 1/2	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3
17369	4.8	24.0	34.9	99-2900	L	L	7-3 1/2	PC	Pe	Zen	MAL	DR	P.B.B	Yo	Blo	Cla F318	Ros	L4IH	318	p	CD	81	61	31 1/2	41x2 1/2	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3
18369	4.8	24.0	34.9	99-2900	L	L	7-3 1/2	PC	Pe	Zen	MAL	DR	P.B.B	Yo	Blo	Cla F318	Ros	L4IH	318	p	CD	81	61	31 1/2	41x2 1/2	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3
19428	4.8	24.0	34.9	99-2900	L	L	7-3 1/2	PC	Pe	Zen	MAL	DR	P.B.B	Yo	Blo	Cla F318	Ros	L4IH	318	p	CD	81	61	31 1/2	41x2 1/2	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3
20428	4.8	24.0	34.9	99-2900	L	L	7-3 1/2	PC	Pe	Zen	MAL	DR	P.B.B	Yo	Blo	Cla F318	Ros	L4IH	318	p	CD	81	61	31 1/2	41x2 1/2	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3
21428	4.8	24.0	34.9	99-2900	L	L	7-3 1/2	PC	Pe	Zen	MAL	DR	P.B.B	Yo	Blo	Cla F318	Ros	L4IH	318	p	CD	81	61	31 1/2	41x2 1/2	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3
22525	4.8	24.0	34.9	99-2900	L	L	7-3 1/2	PC	Pe	Zen	MAL	DR	P.B.B	Yo	Blo	Cla F318	Ros	L4IH	318	p	CD	81	61	31 1/2	41x2 1/2	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3
23525	4.8	24.0	34.9	99-2900	L	L	7-3 1/2	PC	Pe	Zen	MAL	DR	P.B.B	Yo	Blo	Cla F318	Ros	L4IH	318	p	CD	81	61	31 1/2	41x2 1/2	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3	54 1/2 x3
24228	4.7	14.2	27.3	59-2900	L	L	7-3 1/2	PC	Op	Str	MAL	AL	P.B.L	Yo	Spi	Tim 31020H	Ros	L4IH	356	G	TX	92	56	34	37x2 1/2	54x3	54x3	54x3	54x3	54x3
25282	4.7	14.2	27.3	59-2900	L	L	7-3 1/2	PC	Op	Str	MAL	AL	P.B.L	Yo	Spi	Tim 31020H	Ros	L4IH	356	G	TX	92	56	34	37x2 1/2	54x3	54x3	54x3	54x3	54x3
26282	4.7	14.2	27.3	59-2900	L	L	7-3 1/2	PC	Op	Str	MAL	AL	P.B.L	Yo	Spi	Tim 31020H	Ros	L4IH	356	G	TX	92	56	34	37x2 1/2	54x3	54x3	54x3	54x3	54x3
27339	4.7	14.2	27.3	59-2900	L	L	7-3 1/2	PC	Op	Str	MAL	AL	P.B.L	Yo	Spi	Tim 31020H	Ros	L4IH	356	G	TX	92	56	34	37x2 1/2	54x3	54x3	54x3	54x3	54x3
28339	4.7	14.2	27.3	59-2900	L	L	7-3 1/2	PC	Op	Str	MAL	AL	P.B.L	Yo	Spi	Tim 31020H	Ros	L4IH	356	G	TX	92	56	34	37x2 1/2	54x3	54x3	54x3	54x3	54x3
29428	4.7	14.2	27.3	59-2900	L	L	7-3 1/2	PC	Op	Str	MAL	AL	P.B.L	Yo	Spi	Tim 31020H	Ros	L4IH	356	G	TX	92	56	34	37x2 1/2	54x3	54x3	54x3	54x3	54x3
30428	4.7	14.2	27.3	59-2900	L	L	7-3 1/2	PC	Op	Str	MAL	AL	P.B.L	Yo	Spi	Tim 31020H	Ros	L4IH	356	G	TX	92	56	34	37x2 1/2	54x3	54x3	54x3	54x3	54x3
31428	4.7	14.2	27.3	59-2900	L	L	7-3 1/2	PC	Op	Str	MAL	AL	P.B.L	Yo	Spi	Tim 31020H	Ros	L4IH	356	G	TX	92	56	34	37x2 1/2	54x3	54x3	54x3	54x3	54x3
32501	4.9	33.0	48.6	110-2200	L	L	7-3 1/2	PC	Ha	Str	MAL	AL	P.B.L	Yo	Spi	Shu 1552B	Ros	L4IH	398	G	CD	142	83	34 1/2	40x2 1/2	54x3	54x3	54x3	54x3	54x3
33529	4.9	33.0	48.6	110-2200	L	L	7-3 1/2	PC	Ha	Str	MAL	AL	P.B.L	Yo	Spi	Shu 1552B	Ros	L4IH	398	G	CD	142	83	34 1/2	40x2 1/2	54x3	54x3	54x3	54x3	54x3
34762	17.4	42.0	57.0	125-1800	H	L	7-3 1/2	CC	FN	No	MAL	AL	P.B.L	Yo	Spi	Shu 1552B	Ros	L4IH	398	G	CD	142	83	34 1/2	40x2 1/2	54x3	54x3	54x3	54x3	54x3
35213	5.0	13.3	26.3	66-3400	L	L	7-3 1/2	CC	FN	No	MAL	AL	P.B.L	Yo	Spi	Shu 1552B	Ros	L4IH	398	G	CD	142	83	34 1/2	40x2 1/2	54x3	54x3	54x3	54x3	54x3
36186	4.6	12.4	21.1	39-2400	L	L	7-3 1/2	CC	FN	No	MAL	AL	P.B.L	Yo	Spi	Shu 1552B	Ros	L4IH	398	G	CD	142	83	34 1/2	40x2 1/2	54x3	54x3	54x3	54x3	54x3
37186	4.6	12.4	21.1	3																										

Line Number	MAKE AND MODEL	GENERAL (See Keynote)					TIRE SIZE		MAJOR UNITS							FRAME				
		Tonnage Rating	Chassis Price	Standard Wheelbase	Max. W. B. Furnished	Gross Vehicle Weight	Chassis Wt. (Stripped)	Front	Rear	ENGINE		TRANSMISSION		REAR AXLE			Gear Type			
										Make and Model	No. of Cylinders Bore and Stroke	Make and Model	Location and Forward Speeds	Aux. Location and Speeds	Make and Model	Gear and Type		Drive and Torque	GEAR RATIOS In High In Low	Side Rail Dimensions
1	Moreland (Con.) E226	6	3405	184	Op	22000	7150	B9.75/20	DB9.75/20	Her WXC3	6-4x4x4	BL 524	U4 No	Tim 65720H	WF	R 7.25	44.5	9x3x3x3x3	T	1383.4
2	H26	6	4815	196	Op	26000	8900	B9.75/20	DB9.75/20	Her RXB	6-4x4x4	BL 524	U4 No	Tim 66720W	WF	R 8.20	59.8	9x3x3x3x3	T	2501.4
3	Omort	20	1950	131	210	10000	3725	B6.00/20	DB6.00/20	Her JXA	6-3x4x4	WG T9	U4 No	Tim 53200H	BF	R 6.20	39.9	6x2x2x2x2	T	1322.4
4		25	2650	130	130	16000	6100	P32x6	DP32x6	Her WXB	6-3x4x4	Fu MGU	U4 No	Wls 6787L	2F	R 6.41	41.6	6x3x3x3x3	I	4298.4
5		30	3250	134	148	18000	6600	P34x7	DP34x7	Her WXB	6-3x4x4	Fu MGOG	A4 U2	Wls 8817L	2F	R 7.93	65.0	6x3x3x3x3	I	5298.4
6		35	3850	150	150	21000	7600	P36x8	DP36x8	Her WXC	6-4x4x4	Fu MGOG	A4 U2	Wls 1567H	2F	R 9.11	74.7	6x3x3x3x3	I	6339.4
7	Pierce-Arrow 138385	2-2 1/2	157298	157	200	13000	5750	B8.25/20	DB8.25/20	Ow 8	8-3x4x5	Co RU4SL	U4 No	Tim 56200	SF	R 5.28	32.6	7x4x3x3x3	C	6339.4
8		3-3 1/2	1950	160	200	15000	5725	B8.25/20	DB8.25/20	Her WXB	6-3x4x4	Cl 108B	U4 Op	Tim 56200	SF	R 6.16	40.2	7x4x3x3x3	C	8298.4
9		3-4	2350	150	200	17000	5725	B9.00/20	DB9.00/20	Her WXC2	6-4x4x4	Co RU4SL	U4 Op	Tim 58200	SF	R 6.83	42.2	7x4x3x3x3	C	9361.4
10		3-4 1/2	3000	150	220	18000	6660	B9.00/20	DB9.00/20	Her WXC2	6-4x4x4	Co RU4SL	U4 Op	Tim 57200	WF	R 6.8	42	8x4x3x3x3	C	10361.4
11		3-4 1/2	3600	150	220	19000	7850	B9.00/20	DB9.00/20	Her YXC3	6-4x4x4	Co TNU	U4 Op	Tim 65720	WF	R 5.4	28.6	8x4x3x3x3	C	11479.4
12		5-6	4150	150	200	24000	9250	B10.50/20	DB10.50/20	Her YXC3	6-4x4x4	Co TNU	U4 Op	Tim 66720	WF	R 7.6	40.2	9x4x3x3x3	C	12479.4
13		6-7	5400	160	200	28000	11600	B10.50/24	DB10.50/24	Her GXA	6-4x4x4	Ow 8	A4 Op	Ow 8	WF	R 10.525	9x4x3x3x3	C	13611.4	
14	Reo (A)	1500 lb.	530	130	130	5500	2805	B6.50/18	B6.50/18	Ow 8	6-3x4x5	Ow 8	U3	Ow 8	BF	H 4.9	16.1	7x2x2x2x2	C	14230.5
15		1 1/2	595	140	164	10500	3260	B6.00/20	DB6.00/20	Ow 8	6-3x4x5	Ow 8	A4 Op	Ow 8	BF	H 5.28	34.8	7x2x2x2x2	C	15230.5
16		2	845	142	166	12500	3865	B6.50/20	DB6.50/20	Ow 8	6-3x4x5	Ow 8	A4 Op	Ow 8	BF	H 5.28	34.8	7x2x2x2x2	C	16268.4
17		2 1/2	1245	142	184	15000	4475	B7.00/20	DB7.00/20	Ow 8	6-3x4x5	Ow 8	A4 Op	Ow 8	BF	H 5.28	34.8	7x2x2x2x2	C	17268.4
18		3	1795	170	205	17500	5125	B7.50/20	DB7.50/20	Ow 8	6-3x4x5	Ow 8	A4 Op	Ow 8	BF	H 5.28	34.8	7x2x2x2x2	C	18309.4
19		3 1/2	2295	170	205	20000	6280	B9.00/20	DB9.00/20	Ow 8	6-3x4x5	Ow 8	A4 Op	Ow 8	BF	H 5.28	34.8	7x2x2x2x2	C	19358.4
20	Schacht	10H	1295	166	199	11500	4850	B6.50/20	DB6.50/20	Con 20C	6-3x4x4	BL 35	U4 No	Tim	BF	H 5.83	31.2	6x2x2x2x2	P	20448.5
21		15HA	1735	160	199	13000	5200	B8.25/20	DB8.25/20	Con 20C	6-3x4x4	BL 35	U4 No	Tim	BF	H 5.83	31.2	6x2x2x2x2	P	21448.5
22		20HA	2185	160	199	15000	5450	B8.25/20	DB8.25/20	Her WXC	6-4x4x4	Ow 8	U5 No	Tim	BF	H 6.06	38.5	6x3x3x3x3	P	22448.5
23		25HA	2750	160	199	17000	5750	B9.00/20	DB9.00/20	Her WXC	6-4x4x4	Fu 5A-38	U5 No	Tim	BF	H 6.02	39.2	7x3x3x3x3	P	23349.4
24		30HA	3050	146	227	23000	6600	B9.75/20	DB9.75/20	Her WXC	6-4x4x4	Fu 5A-38	U5 No	Tim	BF	H 6.02	39.2	7x3x3x3x3	P	24349.4
25		35HA	3295	146	227	23000	6800	B9.75/20	DB9.75/20	Her WXC	6-4x4x4	Fu 5A-38	U5 No	Wls	2F	R 7.14	46.4	7x3x3x3x3	P	25349.4
26		40H	3725	146	227	24000	7400	B9.75/20	DB9.75/20	Her WXC2	6-4x4x4	Fu 5A-38	U5 No	Ow 8	2F	R 8.00	52.0	8x4x3x3x3	P	26360.4
27		45H	4295	156	239	25500	7600	B9.75/20	DB9.75/20	Her YXC	6-4x4x4	Fu 5A-53	U5 No	Ow 8	2F	R 7.07	49.8	8x4x3x3x3	P	27448.5
28		50H	4695	156	239	25500	7750	B10.50/20	DB10.50/20	Her YXC	6-4x4x4	Fu 5A-53	U5 No	Wls	2F	R 7.07	49.8	8x4x3x3x3	P	28448.5
29		55H	4895	156	239	25500	7850	B10.50/24	DB10.50/24	Her YXC	6-4x4x4	Fu 5A-53	U5 No	Wls	2F	R 7.07	49.8	8x4x3x3x3	P	29448.5
30		60H	5150	156	239	25500	8250	B10.50/24	DB10.50/24	Her YXC	6-4x4x4	Fu 5A-53	U5 No	Ow 8	2F	R 7.8	56.8	7x3x3x3x3	P	30428.4
31		65H	5450	156	239	25500	8450	B10.50/24	DB10.50/24	Her YXC	6-4x4x4	Fu 5A-53	U5 No	Ow 8	2F	R 7.8	56.8	7x3x3x3x3	P	31478.4
32		70H	5750	156	239	25500	8650	B10.50/24	DB10.50/24	Her YXC	6-4x4x4	Fu 5A-53	U5 No	Wls	2F	R 7.8	56.8	7x3x3x3x3	P	32529.4
33	Sterling	FB40	1135	142	162	11000	3450	B6.50/20	DB6.50/20	Con 25A	6-3x4x4	WG T9	U4 No	Ow 8	BF	H 5.66	36.2	6x2x2x2x2	C	33214.5
34		FB50	1240	142	162	11500	3650	B7.00/20	DB7.00/20	Wau MK	6-4x4x4	WG T9	U4 No	Ow 8	BF	H 5.66	36.2	6x2x2x2x2	C	34214.5
35		FB60	1402	142	162	14000	4150	B7.00/20	DB7.00/20	Wau TL	6-3x4x4	WG T9	U4 No	Ow 8	BF	H 5.66	36.2	6x2x2x2x2	C	35255.5
36		FB70	1605	142	162	17000	4650	B7.50/20	DB7.50/20	Wau ML	6-4x4x4	Ow 8	U5 No	Ow 8	BF	H 5.66	36.2	6x2x2x2x2	C	36358.4
37		FB80	1805	142	162	21000	5150	B8.25/20	DB8.25/20	Wau ML	6-4x4x4	Ow 8	U5 No	Ow 8	BF	H 5.66	36.2	6x2x2x2x2	C	37358.4
38		FB80 Spec	2005	142	162	21000	5650	B8.25/20	DB8.25/20	Wau ML	6-4x4x4	Ow 8	U5 No	Ow 8	BF	H 5.66	36.2	6x2x2x2x2	C	38358.4
39		FC90	2205	142	162	22000	6150	B9.00/20	DB9.00/20	Wau MK	6-4x4x4	Ow 8	U5 No	Ow 8	CD	R 8.66	61.7	10x3x4x4x4	L	39381.4
40		FC90	2405	142	162	24000	6650	B9.00/20	DB9.00/20	Wau MK	6-4x4x4	Ow 8	U5 No	Ow 8	CD	R 8.0	57.0	10x3x4x4x4	L	40381.4
41		FC97S	2605	142	162	26000	7150	B9.00/20	DB9.00/20	Wau 6SRLL	6-4x4x4	Ow 8	U5 No	Ow 8	w/2F	R 9.3	61.2	12x3x4x4x4	L	41462.4
42		FC100	2805	142	162	28000	7650	B9.00/20	DB9.00/20	Wau 6SRLL	6-4x4x4	Ow 8	U5 No	Ow 8	CD	R 9.3	61.2	12x3x4x4x4	L	42381.4
43		FC115	3005	142	162	30000	8150	B9.00/20	DB9.00/20	Wau 6SRLL	6-4x4x4	Ow 8	U5 No	Ow 8	w/2F	R 8.20	56.4	12x3x4x4x4	L	43462.4
44		FC107	3205	142	162	32000	8650	B9.00/20	DB9.00/20	Wau 6SRLL	6-4x4x4	Ow 8	U5 No	Ow 8	CD	R 8.20	56.4	12x3x4x4x4	L	44462.4
45		FC107	3405	142	162	34000	9150	B9.00/20	DB9.00/20	Wau 6SRLL	6-4x4x4	Ow 8	U5 No	Ow 8	w/2F	R 10.0	66.6	15x3x4x4x4	L	45462.4
46		FC135	3605	142	162	36000	9650	B9.00/20	DB9.00/20	Wau SRL	6-4x4x4	Ow 8	U5 No	Ow 8	CD	R 9.3	62.2	15x3x4x4x4	L	46462.4
47		FC140	3805	142	162	38000	10150	B9.00/20	DB9.00/20	Wau 6-125	6-4x4x4	Ow 8	U5 No	Ow 8	CD	R 9.3	62.2	15x3x4x4x4	L	47462.4
48		FC140	4005	142	162	40000	10650	B9.00/20	DB9.00/20	Wau 6-125	6-4x4x4	Ow 8	U5 No	Ow 4						

Type	ENGINE DETAILS										FUEL SYST.	ELEC-TRICAL	FRONT AXLE	BRAKES		BODY MOUNT-ING DATA		SPRINGS														
	MAIN BEARINGS													SERVICE		Type		Type														
	Line Number	Piston Displacement	Compression Ratio	Torque lb. ft.	N.A.C.C. Rated H.P.	Max. Brake H.P. at R.P.M. Given	Valve Arrangement	Camshaft Drive	Piston Material	Number and Diameter	Length	Oiling System	Governor Make	Carburetors Make	Fuel Feed	Ignition System Make	Generator, Starter Make	Clutch Type and Make	Radiator Make	Universal Make	Steering Gear Make	Make, Location Type Operation	Lining Area	Drum Material	Hand Location, Type	Cab to Rear of Frame	Cab to Rear Axle	Width of Frame	Front	Rear	Auxiliary Type	
1383	4.4	26.2	43.3	92	2400	L	G	C	7-2	13	PC	No	Zen	M	AL	AL	P.B.L	Lo	Cle	Tim 35120H	Ros	L4IH	555	a	TD	156	101	34	41x2	54x3		
2501	4.9	33.0	48.6	110	2200	L	G	C	7-2	12	PC	No	Zen	M	AL	AL	P.B.L	Lo	Cle	Tim 26450TW	Ros	W4IA	620	a	TD	168	111	34	42x2	56x3		
3228	4.4	14.3	27.3	60	2400	L	G	C	7-2	10	PC	No	Zen	M	AL	AL	P.B.L	Lo	Cle	Tim 30000H	Ros	L4IH	249	p	TD	81	51	34	36x2	45x2		
4298	4.7	19.0	35.7	66	2400	L	G	C	7-2	13	PC	No	Zen	M	AL	AL	P.B.L	Lo	Cle	Shu 5429	Ros	L4IH	406	G	TD	84	53	31	40x2	54x3		
5063	4.7	22.5	38.4	73	2000	L	G	C	7-2	13	PC	No	Zen	M	AL	AL	P.B.L	Lo	Cle	Shu 5532	Ros	L4IH	406	G	TD	88	57	31	40x2	54x3		
7385	5.0	27.4	39.2	125	2800	L	G	C	7-2	14	CC	Ha	Str	M	DR	DR	P.Lo	Lo	Cle	Tim 14706	Ha	B4IM	399	D	TD	116	62	34	38x2	56x3		
8298	4.7	19.0	33.7	70	2600	L	G	C	7-2	13	PC	Ha	Zen	M	DR	DR	P.Lo	Lo	Cle	Tim 14706	Ha	B4IM	399	D	TD	113	59	34	38x2	56x3		
9361	4.7	23.0	40.3	77	2400	L	G	C	7-2	13	PC	Ha	Zen	M	DR	DR	P.Lo	Lo	Cle	Tim 14706	Ha	B4IM	399	D	CD	143	89	34	38x2	56x3		
10361	4.7	23.0	40.3	77	2400	L	G	C	7-2	13	PC	Ha	Zen	M	DR	DR	P.Lo	Lo	Cle	Tim 15735	Ha	B4IA	473	D	CD	119	65	34	41x2	56x3		
11479	4.7	31.8	51.3	104	2200	L	G	C	7-2	14	PC	Ha	Zen	M	DR	DR	P.Lo	Lo	Cle	Tim 15733	Ha	B4IA	473	D	CD	118	163	34	41x2	56x3		
12479	4.6	31.8	51.3	104	2200	L	G	C	7-2	14	PC	Ha	Zen	M	DR	DR	P.Lo	Lo	Cle	Tim 26050	Ha	B4IA	473	D	CD	168	72	34	41x2	56x3		
13611	4.5	41.0	54.1	130	2000	L	G	C	7-3	16	CC	Ha	Zen	M	DR	DR	P.Lo	Lo	Spi	Tim 26050	Ha	B4IA	720	T	TD	127	72	34	41x2	56x3		
14230	5.3	15.2	23.4	68	2800	L	G	C	7-2	12	CC	No	Str	M	DR	DR	P.Ow	Ow	Cle	Own	Ros	L4IH	280	P	TD	21	102	34	38x2	57x2		
15230	5.3	15.2	23.4	68	2800	L	G	C	7-2	12	CC	No	Str	M	DR	DR	P.Ow	Ow	Cle	Own	Ros	L4IH	280	P	TD	21	102	34	38x2	57x2		
16268	4.9	17.5	27.3	75	2800	L	G	C	7-2	12	CC	No	Str	M	DR	DR	P.Ow	Ow	Cle	Own	Ros	L4IH	289	a	TD	21	105	60	34	40x2	52x2	
17268	4.9	17.5	27.3	75	2800	L	G	C	7-2	12	CC	No	Str	M	DR	DR	P.Ow	Ow	Cle	Own	Ros	L4IH	289	a	TD	21	105	60	34	40x2	52x2	
18309	4.9	20.1	31.3	104	2200	L	G	C	7-2	14	CC	No	Str	M	DR	DR	P.Ow	Ow	Cle	Own	Ros	L4IH	344	a	FD	124	68	34	44x3	54x3		
19348	4.9	23.0	36.4	110	2800	L	G	C	7-2	15	CC	Mo	Sch	M	DR	DR	P.Lo	Lo	Cle	Own	Ros	L4IH	390	G	TD	143	83	34	44x3	56x3		
20248	5.1	15.0	27.3	65	2600	L	G	C	7-2	10	FP	No	Str	M	DR	DR	P.Ow	Ow	Spi	Tim	Ros	L4IH	452	G	TX	129	57	31	40x2	50x3		
21248	5.1	15.0	27.3	65	2600	L	G	C	7-2	10	FP	No	Str	M	DR	DR	P.Ow	Ow	Spi	Tim	Ros	L4IH	452	G	TX	129	57	31	40x2	50x3		
22339	4.7	22.5	38.4	73	2200	L	G	C	7-2	13	PC	Mo	Str	M	AL	AL	D.BB	Yo	Spi	Tim	Ros	L4IH	452	G	TX	129	57	31	40x2	50x3		
23339	4.7	22.5	38.4	73	2200	L	G	C	7-2	13	PC	Mo	Str	M	AL	AL	D.Fu	Yo	Spi	Tim	Ros	L4IH	452	G	TX	129	57	31	40x2	50x3		
24339	4.7	22.5	38.4	73	2200	L	G	C	7-2	13	PC	Mo	Str	M	AL	AL	D.Fu	Yo	Spi	Shu	Ros	L4IH	452	G	TX	129	57	31	40x2	50x3		
25339	4.7	22.5	38.4	73	2200	L	G	C	7-2	13	PC	Mo	Str	M	AL	AL	D.Fu	Yo	Spi	Shu	Ros	L4IH	452	G	TX	129	57	31	40x2	50x3		
26360	4.7	23.8	40.3	80	2200	L	G	C	7-3	15	PC	Mo	Str	M	AL	AL	D.Fu	Yo	Spi	Shu	Ros	L4IH	452	G	TX	129	57	31	40x2	50x3		
27428	4.4	28.0	45.9	93	2200	L	G	C	7-3	15	PC	Mo	Str	M	AL	AL	D.Fu	Yo	Spi	Shu	Ros	L4IH	452	G	TX	129	57	31	40x2	50x3		
28428	4.4	28.0	45.9	93	2200	L	G	C	7-3	15	PC	Mo	Str	M	AL	AL	D.Fu	Yo	Spi	Shu	Ros	L4IH	452	G	TX	129	57	31	40x2	50x3		
29529	4.9	35.5	51.2	115	2200	L	G	C	7-3	15	PC	Mo	Str	M	AL	AL	D.Fu	Yo	Spi	Shu	Own	W4IA	847	G	TD	118	102	31	40x2	60x3		
30428	4.4	28.0	45.9	93	2200	L	G	C	7-3	15	PC	Mo	Str	M	AL	AL	D.Fu	Yo	Spi	Shu	Ros	L4IH	893	H	TD	92	107	31	40x2	50x3		
31478	4.4	28.0	45.9	93	2200	L	G	C	7-3	15	PC	Mo	Str	M	AL	AL	D.Fu	Yo	Spi	Shu	Ros	L4IH	893	H	TD	92	107	31	40x2	50x3		
32478	4.4	28.0	45.9	93	2200	L	G	C	7-3	15	PC	Mo	Str	M	AL	AL	D.Fu	Yo	Spi	Shu	Ros	L4IH	893	H	TD	92	107	31	40x2	50x3		
33529	4.4	35.5	51.2	115	2200	L	G	C	7-3	15	PC	Mo	Str	M	AL	AL	D.Fu	Yo	Spi	Shu	Ros	L4IH	893	H	TD	92	107	31	40x2	50x3		
34529	5.0	13.7	28.0	72	3300	L	C	A	4-2	6	CC	No	Zen	M	DR	DR	P.Lo	Lo	Pe	Tim 30000H	Ros	L4IH	269	P	TX	96	57	34	38x2	50x2		
35529	5.0	13.7	28.0	72	3300	L	C	A	4-2	6	CC	No	Zen	M	DR	DR	P.Lo	Lo	Pe	Tim 31000H	Ros	L4IH	282	P	TX	96	57	34	38x2	50x2		
36529	4.4	23.0	38.4	80	2500	L	G	C	7-2	12	CC	Ha	Zen	M	DR	DR	P.Ow	Ow	Spi	Tim 31000H	Ros	L4IH	330	a	TX	144	91	34	42x2	54x3		
37529	4.4	23.0	38.4	80	2500	L	G	C	7-2	12	CC	Ha	Zen	M	DR	DR	P.Ow	Ow	Spi	Tim 33000H	Ha	L4IH	396	a	TX	144	91	34	42x2	54x3		
38529	4.4	23.0	38.4	80	2500	L	G	C	7-2	12	CC	Ha	Zen	M	DR	DR	P.Ow	Ow	Spi	Tim 33000H	Ha	L4IH	396	a	TX	144	91	34	42x2	54x3		
39529	4.4	23.0	38.4	80	2500	L	G	C	7-2	12	CC	Ha	Zen	M	DR	DR	P.Ow	Ow	Spi	Tim 33000H	Ha	L4IH	396	a	TX	144	91	34	42x2	54x3		
40529	4.4	23.0	38.4	80	2500	L	G	C	7-2	12	CC	Ha	Zen	M	DR	DR	P.Ow	Ow	Spi	Tim 35000H	Ros	L4IH	664	a	TX	172	108	34	48x3	54x3		
41529	4.4	23.0	38.4	80	2500	L	G	C	7-2	12	CC	Ha	Zen	M	DR	DR	P.Ow	Ow	Spi	Tim 35000H	Ros	L4IH	664	a	TX	172	108	34	48x3	54x3		
42529	4.4	23.0	38.4	80	2500	L	G	C	7-2	12	CC	Ha	Zen	M	DR	DR	P.Ow	Ow	Spi	Tim 35000H	Ros	L4IH	664									

Line Number	MAKE AND MODEL	GENERAL (See Keynote)			TIRE SIZE		MAJOR UNITS										FRAME				
		Wheels Driven—6-Wheelers	Tonnage Rating	Chassis Price	Standard Wheelbase	Max. W. B. Furnished	Gross Vehicle Weight	Chassis Wt. (Stripped)	Front	Rear	ENGINE		TRANSMISSION		REAR AXLE				Side Rail Dimensions	Type	
											Make and Model	No. of Cylinders Bore and Stroke	Make and Model	Location and Forward Speeds	Make and Model	Gear and Type	Drive and Torque	Gear Ratios In High In Low			
Four-Wheel-Drive																					
1	Coleman.....E52	2 1/2	3800	120	144	12800	7200	B9.00/24	B9.00/24	Bud K393	6-4 1/2 x 4 1/2	Fu RU 16	U 4 A 2	Wis CR15	2F	H Opt	Opt	10x2 1/4 x 1 1/4	B		
2E53	3 1/2	5300	130	180	18900	8000	B9.75/24	B9.75/24	Bud K428	6-4 1/2 x 4 1/2	Fu MRU16	U 4 A 2	Wis CR26	2F	H Opt	Opt	12x2 1/4 x 1 1/4	B		
3E54	4 1/2	5600	130	180	20400	8800	B10.50/24	B10.50/24	Bud L468	6-4 1/2 x 5 1/2	Fu MRU16	U 4 A 2	Wis CR30	2F	H Opt	Opt	12x2 1/4 x 1 1/4	B		
4E55	5 1/2	6150	130	180	23000	9600	B11.25/24	B11.25/24	Bud L525	6-4 1/2 x 5 1/2	Fu MRU16	U 4 A 2	Wis CR30	2F	H Opt	Opt	12x2 1/4 x 1 1/4	B		
5E55S	5 1/2	7200	144	180	24500	10600	B11.25/24	B11.25/24	Bud L525	6-4 1/2 x 5 1/2	Fu MRU16	U 4 A 2	Wis CR122	2F	H Opt	Opt	12x2 1/4 x 1 1/4	B		
6E56	6 1/2	7800	144	180	29800	11600	B10.50/24	DB10.50/24	Bud GF6	6-4 1/2 x 6	Fu MHU	U 4 A 2	Wis CR122	2F	H Opt	Opt	12x2 1/4 x 1 1/4	B		
7E57	7 1/2	9700	144	180	32000	12400	B11.25/24	DB11.25/24	Ste LT6	6-5 1/2 x 6	BL 714	U 4 A 2	Wis CR122	2F	H Opt	Opt	16x3 1/4 x 1 1/4	B		
8	Corblitt (3).....10FB6	1 1/2-2 1/2	2300	Op	Op	Op	4420	B6.50/20	DB6.50/20	Con 25A	6-3 1/2 x 4	BL 214	U 4 A 2	Tim 53200H	SF	H 6.20	Opt	8 1/2 x 3 1/4	T		
910FB6	2 1/2-3 1/2	3200	Op	Op	Op	5060	B7.00/20	DB7.00/20	Con 20C	6-3 1/2 x 4	BL 224	U 4 A 2	Tim 54200H	SF	H 7.80	Opt	8 1/2 x 3 1/4	T		
1012FB6	2 1/2-3 1/2	4000	Op	Op	Op	5630	B7.50/20	DB7.50/20	Con E602	6-4 1/2 x 4 1/2	FuL 5A38	U 5 A 2	Tim 56200H	SF	H 7.40	Opt	9x3 1/4	T		
1112FD6	2 1/2-3 1/2	4300	Op	Op	Op	5730	B7.50/20	DB7.50/20	Con E602	6-4 1/2 x 4 1/2	FuL 5A38	U 5 A 2	Wis 4916L	2F	H 7.36	Opt	9x3 1/4	T		
1215FD6	3 1/2-4 1/2	5700	Op	Op	Op	8100	B8.25/20	DB8.25/20	Con 21R	6-4 1/2 x 4 1/2	FuL 5A53	U 5 A 2	Wis 70000L	2F	H 8.00	Opt	9x3 1/4	T		
1318FD6	3 1/2-4 1/2	6300	Op	Op	Op	9200	B9.00/20	DB9.00/20	Con 22R	6-4 1/2 x 4 1/2	FuL 5A53	U 5 A 2	Wis 1237H	2F	H 8.00	Opt	9x3 1/4	T		
14	FWD.....H4	1 1/2-2 1/2	3325	120	160	11000	5300	P34x7	P34x7	Wis BU	4-4x5	Cot A	A 4 Op	Own H	BF	H 7.86	38.0	5 1/2 x 2 1/4	C		
15H5	2 1/2-3 1/2	3385	133	180	13000	5900	P9.00/20	P9.00/20	Wau MS	6-3 1/2 x 4 1/2	BL 51	A 4 Op	Own U	BF	H 8.92	47.7	5 1/2 x 2 1/4	C		
16H6	3 1/2-4 1/2	4135	138	170	16000	6900	P9.75/20	P9.75/20	Wau MK	6-3 1/2 x 4 1/2	BL 55	A 7 R2	Own U	BF	H 9.35	54.7	5 1/2 x 2 1/4	C		
17B3	4 1/2-5 1/2	4200	124	156	15500	6460	S36x8	S36x8	Own A	4-4 1/2 x 5 1/2	Cot DAF	A 3 Op	Own B	BF	H 8.9	35.6	5 1/2 x 2 1/4	C		
18CU-6	3 1/2-4 1/2	4985	147	179	19500	8000	B10.50/20	B10.50/20	Wau SR5	6-4 1/2 x 5 1/2	Own U	A 5 Op	Own U	BF	H 7.35	37.3	7x3 1/4	C		
19CU6A	3 1/2-4 1/2	4685	147	179	19000	7800	B10.50/20	B10.50/20	Wau SR5	6-4 1/2 x 5 1/2	BL 615	A 5 Op	Own U	BF	H 7.62	55.2	7x3 1/4	C		
20SSU	4-5	5135	147	179	22000	8300	B11.25/20	B11.25/20	Wau SRL	6-4 1/2 x 5 1/2	Own U	A 5 Op	Own U	BF	H 7.35	37.3	7x3 1/4	C		
21SSUA	4-5	4835	147	179	21500	8100	B11.25/20	B11.25/20	Wau SRL	6-4 1/2 x 5 1/2	BL 706	A 5 Op	Own M	BF	H 7.35	37.3	7x3 1/4	C		
22TH3	5-6 1/2	7400	165	195	29500	11200	B12.75/20	B12.75/20	Wau SRK	6-4 1/2 x 5 1/2	BL 714	U 4 A 2	Wis 131W	2F	H 10.0	207	8x3 1/4	C		
23MF6	5-6 1/2	5785	147	179	24500	9100	B10.50/20	DB10.50/20	Wau SRL	6-4 1/2 x 5 1/2	Own U	A 5 Op	Own M	BF	H 7.35	37.3	7x3 1/4	C		
24	(F-T-Wb-Dr.) LBU	5-6 1/2	4800	171	Op	23500	9000	B9.00/20	DB9.00/20	Wau SR5	6-4 1/2 x 5 1/2	BL 55	U 4 Op	Own U	D	H 7.35	39.5	7x3 1/4	C		
25M7	7 1/2-10	8500	165	195	37000	12400	P40x10	DP40x10	Wau SR5	6-5 1/2 x 6	BL 714	U 4 A 2	Wis 131W	2F	H 8.36	173	10x3 1/4	C		
26	(T) 60-T	20-25	6300	134	Op	60000	10000	B10.50/20	DB10.50/20	Wau 125	6-4 1/2 x 5 1/2	Own U	A 5 Op	Own M	BF	H 7.35	37.3	7x3 1/4	C		
27	(T) 72-T	25-30	7000	120	Op	72000	10450	B9.75/20	DB9.75/20	Wau 125	6-4 1/2 x 5 1/2	BL 724	U 4 A 2	Wis 1237	2F	H 6.7	47.4	8x3 1/4	C		
28	Indiana.....12XA	1 1/2	2650	141	Op	10000	4350	B6.50/20	DB6.50/20	Her JXC	6-3 1/2 x 4 1/2	BL	U 4 A 2	Tim 53200H	SF	H 5.14	54.7	5 1/2 x 2 1/4	C		
2913XA	2 1/2	3950	141	Op	14000	5900	B7.50/20	DB7.50/20	Her WXB	6-3 1/2 x 4 1/2	BL	U 4 A 2	Wis	SF	H 5.40	50.0	5 1/2 x 2 1/4	C		
3016XA	3 1/2	4850	156	Op	16000	7500	B8.25/20	DB8.25/20	Her WXC2	6-4 1/2 x 4 1/2	BL	U 4 A 2	Wis	SF	H 6.06	89.0	8x3 1/4	C		
3118XA	3 1/2	5550	160	Op	21000	9000	B9.00/20	DB9.00/20	Her YXC	6-4 1/2 x 4 1/2	BL	U 4 A 2	Wis	SF	H 7.53	110	8 1/2 x 3 1/4	C		
3218XA	3 1/2	5400	160	224	21000	8700	B9.00/20	DB9.00/20	Her YXC3	6-4 1/2 x 4 1/2	BL	U 4 A 2	Wis	SF	H 6.14	38.6	8 1/2 x 3 1/4	C		
3320XA	4 1/2	7200	188	Op	24000	10600	B9.75/20	DB9.75/20	Her HXB	6-5x6	BL	U 4 A 2	Wis	SF	H 8.00	128	9x3 1/4	C		
3422XA	4 1/2	10000	200	Op	31000	14000	B10.50/20	DB10.50/20	Her HXC	6-5 1/2 x 6	BL	U 4 A 2	Wis	SF	H 9.11	86.0	9 1/2 x 3 1/4	C		
35	LeMoon (c.o.e.) H100	5	2350	135	155	11000	9300	B9.75/20	DB9.75/20	Wau 6RB	6-5x5 1/2	BL 7341	U 4 A 2	Op Own-Tim	WF	H 8.2	51.4	9x3 1/4	C		
36	Mar-Herr.....A10	1 1/2	3250	135	155	11000	9300	B9.75/20	DB9.75/20	Her JXA	6-5x5 1/2	WG T9	U 4 A 2	Op Own-Tim	WF	H 8.60	82.0	7 1/2 x 2 1/4	C		
37A11	2 1/2	3250	135	155	11000	9300	B9.75/20	DB9.75/20	Her JXA	6-5x5 1/2	WG T9	U 4 A 2	Op Own-Tim	WF	H 8.60	82.0	7 1/2 x 2 1/4	C		
38A12	3 1/2	4300	155	167	17000	7000	B8.25/20	DB8.25/20	Her WXC	6-4 1/2 x 4 1/2	Fu 5A380	U 5 A 2	Op Own-Tim	WF	H 7.0	116	9 1/2 x 3 1/4	C		
39A13	4 1/2	4800	155	167	17000	7500	B9.00/20	DB9.00/20	Her WXC3	6-4 1/2 x 4 1/2	Fu 5A380	U 5 A 2	Op Own-Tim	WF	H 7.0	116	9 1/2 x 3 1/4	C		
40A14	5 1/2	5700	155	167	17000	8150	B9.00/20	DB9.00/20	Her WXC3	6-4 1/2 x 4 1/2	Fu 5A380	U 5 A 2	Op Own-Tim	WF	H 7.0	116	9 1/2 x 3 1/4	C		
41TH300	4 1/2	6150	163	193	19300	8985	B9.75/20	DB9.75/20	Her YXC	6-4 1/2 x 4 1/2	Fu 5A380	U 5 A 2	Op Own-Wis	2F	H 8.0	143	9 1/2 x 3 1/4	C		
42TH310	5-5 1/2	7150	163	193	19300	9620	B9.75/20	DB9.75/20	Her YXC3	6-4 1/2 x 4 1/2	Fu 5A380	U 5 A 2	Op Own-Wis	2F	H 8.0	143	9 1/2 x 3 1/4	C		
43TH310A	6	8050	163	193	19300	10120	B9.75/22	DB9.75/22	Her RXC	6-4 1/2 x 5 1/2	Fu 5A530	U 5 A 2	Op Own-Wis	2F	H 7.79	140	9 1/2 x 3 1/4	C		
44(13) TH317	8-9	9350	168	216	21600	10950	B10.50/20	DB10.50/20	Her HXB	6-5x6	BL 724	U 4 A 2	Op Own-Wis	2F	H 7.79	140	9 1/2 x 3 1/4	C		
45(13) TH320	8-9	11500	198	228	22800	12800	B10.50/24	DB10.50/24	Her HXC	6-5 1/2 x 6	BL 724	U 4 A 2	Op Own-Wis	2F	H 8.05	166	10x3 1/4	C		
46	Oshkosh.....JSW	1 1/2-2 1/2	2650	123	Op	10860	4760	B7.00/20	DB7.00/20	Her WXC	6-4 1/2 x 4 1/2	BL 224	U 4 A 2	Op Own JSB	SF	H 6.19	40.1	6 1/2 x 3 1/4	T		
47JSB	1 1/2-2 1/2	2490	140	Op	10750	6700	B9.00/20	B9.00/20	Her JXC	6-3 1/2 x 4 1/2	BL 51-4	U 4 A 2	Op Own JSB	SF	H 6.19	40.1	6 1/2 x 3 1/4	T		
48LB	2 1/2-3 1/2	4375	146	165	13900	6700	B9.00/20	B9.00/20	Her JXC	6-3 1/2 x 4 1/2	BL 51-4	U 4 A 2	Op Own JSB	SF	H 6.19	40.1	6 1/2 x 3 1/4	T		
49LC	3 1/2-4 1/2	4575																		

Line Number	ENGINE DETAILS										FUEL SYST.	ELEC. TRICAL	CLUTCH TYPE AND MAKE	RADIATOR MAKE	UNIVERSALS MAKE	FRONT AXLE	STEERING GEAR MAKE	BRAKES		BODY MOUNTING DATA		SPRINGS		Auxiliary Type																																																																																																																																																																																																																																																																																																																																																																																																																																																																														
	Piston Displacement	Compression Ratio	Torque lb. ft.	N.A.C.C. Rated H.P.	Max. Brake H.P. at R.P.M. Given	Valve Arrangement	Camshaft Drive	Piston Material	MAIN BEARINGS									Oiling System Type	Governor Make	Carburetors Make	Fuel Feed	Ignition System Make	Generator, Starter Make		Service	Lining Area	Drum Material	Hand Location, Type	Cab to Rear of Frame	Cab to Rear Axle	Width of Frame	Front	Rear																																																																																																																																																																																																																																																																																																																																																																																																																																																																					
									Number and Diameter	Length	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model																	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model	Make and Model

Line Number	MAKE AND MODEL	Wheels Driven—6—Wheeler	GENERAL See Keynote					TIRE SIZE		MAJOR UNITS					FRAME						
			Tonnage Rating	Chassis Price	Standard Wheelbase	Max. W. B. Furnished	Gross Vehicle Weight	Chassis Wt. Stripped	Front	Rear	ENGINE		TRANSMISSION		REAR AXLE						
											Make and Model	No. of Cylinders Bore and Stroke	Make and Model	Location and Forward Speeds	Make and Model	Gear and Type	Drive and Torque				
																		In High	In Low	Side Rail Dimensions	Type
1	Ken.	89SBT	2C	7	2380	188	224	25500	7350	P32x6	DP32x6	Her JXC	6-3 1/2 x 4 1/2	BL 234	U 4 Op	Tim SBT151	SF	A 7.4	45.5	8x3x 1/4	TL
2	127SBT	2C	8	3450	188	224	26000	8000	B8.25/20	DB8.25/20	Her WXC2	6-4 1/2 x 4 1/2	BL 334	U 4 Op	Tim SBT151	SF	A 7.4	45.5	8x3x 1/4	TL	
3	146SBT	2C	9	4250	188	224	33000	9000	B9.00/20	DB9.00/20	Bud K393	6-4 1/2 x 4 1/2	BL 334	U 4 Op	Tim SBT251	SF	A 7.8	48.	8x3x 1/4	TL	
4	186SDT	2C	10	6450	205	235	38000	10500	B9.00/20	DB9.00/20	Her YXC2	6-4 1/2 x 4 1/2	BL 1554	U 4 A 3	Tim SBT310W	2F	H 7.33	104.	9x3x 1/4	T	
5	241SDT	2C	10	6850	205	235	40500	11000	B9.00/20	DB9.00/20	Her RXB	6-4 1/2 x 5 1/2	BL 714	U 4 A 3	Tim SBT310W	2F	H 7.33	85.5	9x3x 1/4	T	
6	D-346	4R	10	10250	210	240	40500	14300	B9.75/20	DB9.75/20	Cum HA6	6-4 1/2 x 6	BL 714	U 4 A 3	Tim SW320W	WF	H 6.8	92.	8x3x 1/4	C	
7	346A	4R	10	8800	210	240	40500	13000	B9.75/20	DB9.75/20	Has 160	6-4 1/2 x 5 1/2	BL 714	U 4 A 3	Tim SW310W	WF	H 7.25	84.5	8x3x 1/4	C	
8	346B	4R	10	8550	210	240	40500	13000	B9.75/20	DB9.75/20	Bud GF-6	6-4 1/2 x 6	BL 714	U 4 A 3	Tim SW310W	WF	H 7.25	98.4	8x3x 1/4	C	
9	346C	4R	10	9500	210	240	40500	14000	B9.75/20	DB9.75/20	Has 175	6-5x6	BL 714	U 4 A 3	Tim SW310W	WF	H 7.25	98.4	8x3x 1/4	C	
10	386C	4R	10	10200	210	240	50100	14500	B9.75/20	DB9.75/20	Has 175	6-5x6	BL 714	U 4 A 3	Tim SW410W	WF	H 7.60	103.	8x3x 1/4	C	
11	Kleiber . . .	81	5	1900	180	190	20000	6500	P32x6	DP32x6	Her JXB	6-3 1/2 x 4 1/2	BL 2241	U 4 No	Tim SBT 75	BF	R 5.14	32.	7 1/2 x 3 1/4 x 1/4	C	
12	121	5	7	2800	190	200	26000	8500	B8.25/20	DB8.25/20	Con 18R	6-4x4 1/2	BL 3241	U 4 No	Tim SBT 151	BF	R 6.17	33.4	7 1/2 x 3 1/4 x 1/4	C	
13	141	9	9	3950	200	210	33000	9500	B9.00/20	DB9.00/20	Con 21R	6-4 1/2 x 4 1/2	BL 5241	U 4 No	Tim SBT251	BF	R 6.84	41.	7 1/2 x 3 1/4 x 1/4	C	
14	La Fran-R. Q6	4R	9-12	11625	216	260	40000	14900	B10.50/20	DB10.50/20	Ow 312B	12-4x5	BL 714	U 4 No	Tim SWD410	WF	Opt Opt	12x3 1/4 x 1/4	L	14754.5	
15	Le Moon (9) 701	4R	5-6	4475	187	199	8500	8500	B8.25/20	DB8.25/20	Lye AEC	8-3 1/2 x 4 1/2	Fu VUOG	U 5 No	T 63703-97H	WF	R 6.20	43.8	7x4x 1/4	B	
16	(9) 801	4R	6-7	5100	187	199	9720	9720	B9.00/20	DB9.00/20	Lye AEC	8-3 1/2 x 4 1/2	Fu VUOG	U 5 No	T 65703-97H	WF	H 6.75	47.7	7x4x 1/4	B	
17	802	4R	6-7	5350	187	199	9800	9800	B9.00/20	DB9.00/20	Wau 6SRL	6-4 1/2 x 5 1/2	Fu VUOG	U 5 No	T 65703-97H	WF	H 6.75	47.7	7x4x 1/4	B	
18	900	4R	7-8	6775	191	203	12000	12000	B9.75/20	DB9.75/20	Wau 6SRL	6-4 1/2 x 5 1/2	BL 607	A 7 No	Tim SW310W	WF	H 9.25	86.9	9x4x 1/4	B	
19	1000	4R	8-10	7950	196	208	12600	12600	B9.75/24	DB9.75/24	Wau 6AB	6-4 1/2 x 5 1/2	BL 714	U 4 A 3	Tim SW310W	WF	H 9.25	128.	9x4x 1/4	B	
20	1200	4R	10-12	8500	196	208	14000	14000	B9.75/24	DB9.75/24	Wau 6RB	6-5x5 1/2	BL 714	U 4 A 3	Tim SW410W	WF	H 9.25	128.	9x4x 1/4	B	
21	1200D	4R	10-12	9750	196	208	14000	14000	B9.75/24	DB9.75/24	Cum Dle.H6	6-4 1/2 x 6	BL 714	U 4 A 3	Tim SW410W	WF	H 7.6	47.6	9x4x 1/4	B	
22	Mack	BX	4R	8-15	7950	178	207	12000	B8.25/22	DB8.25/22	Ow 312B	6-4 1/2 x 5 1/2	Ow 312B	U 4 No	Ow 312B	2F	A 6.58	46.0	9x3x 1/4	C	
23	BQ	4R	8-15	8350	217	257	15000	15000	B9.75/22	DB9.75/22	Ow 312B	6-4 1/2 x 5 1/2	Ow 312B	U 4 No	Ow 312B	2F	A 6.54	46.0	9x3x 1/4	C	
24	AC	4R	8-15	8500	217	257	14550	14550	P40x8	DP40x8	Ow 312B	6-4 1/2 x 5 1/2	Ow 312B	U 4 No	Ow 312B	2F	R 9.26	59.4	8x3x 1/4	C	
25	AK	4R	8-15	9000	217	257	15900	15900	B9.75/22	DB9.75/22	Ow 312B	6-4 1/2 x 5 1/2	Ow 312B	U 4 No	Ow 312B	2F	A 7.46	47.8	8x3x 1/4	C	
26	AP	4R	8-15	10500	217	257	14850	14850	P40x8	DP40x8	Ow 312B	6-4 1/2 x 5 1/2	Ow 312B	U 4 No	Ow 312B	2F	R 9.26	59.4	8x3x 1/4	C	
27	AP	4R	8-15	11000	217	257	16400	16400	B9.75/22	DB9.75/22	Ow 312B	6-4 1/2 x 5 1/2	Ow 312B	U 4 No	Ow 312B	2F	A 7.46	47.8	8x3x 1/4	C	
28	Mar-Her TH310A-6	10	10000	193	229	14070	14070	14070	B9.75/22	DB9.75/22	Her RXC	6-4 1/2 x 5 1/2	Fu A530	U 5 A 2	Ow 312B	2F	R 9.11	163.	8x3x 1/4	P	
29	(3) TH15	6	12-13	12500	198	234	15420	15420	B9.75/22	DB9.75/22	Her HXB	6-5x6	BL 724	U 4 A 3	Ow 312B	2F	R 9.11	163.	8x3x 1/4	P	
30	(13) TH320	6	15-18	14500	225	255	18450	18450	B10.50/24	DB10.50/24	Her HXC	6-5 1/2 x 6	BL 724	U 4 A 3	Ow 312B	2F	R 9.11	188.	10x3x 1/4	P	
31	Moreland RA15	2C	3	1550	153	Op	15000	5300	B6.50/20	DB6.50/20	Her JXC	6-3 1/2 x 4 1/2	BL 224	U 4 No	Tim SBT75	SF	R 5.66	35.0	7 1/2 x 2 1/4 x 1/4	T	
32	RA20	2C	5	1981	149	Op	20000	6100	P32x6	DP32x6	Her JXC	6-3 1/2 x 4 1/2	BL 224	U 4 No	Tim SBT151	SF	R 6.17	38.2	7 1/2 x 2 1/4 x 1/4	T	
33	BD21M	4C	5	3534	184	Op	21000	8300	B7.50/20	DB7.50/20	Her WXC3	6-4 1/2 x 4 1/2	BL 334	U 4 No	Tim 64800	WF	R 6.40	39.6	9x3x 1/4	T	
34	ED25M	4C	7	4067	184	Op	25000	8900	B8.25/20	DB8.25/20	Her WXC3	6-4 1/2 x 4 1/2	BL 334	U 4 No	Tim 65000	WF	R 7.50	46.0	9x3x 1/4	T	
35	HD34M	4C	10	5869	220	Op	34000	11000	B9.00/20	DB9.00/20	Her RXB	6-4 1/2 x 5 1/2	BL 524	U 4 No	Tim 65720	W	R 8.5	62.0	9x3x 1/4	T	
36	TD34	4C	10	7607	221	Op	34000	13250	B9.75/20	DB9.75/20	Con 16H	6-4 1/2 x 5 1/2	BL 724	U 4 No	Tim 68720W	W	R 8.75	62.0	11x3x 1/4	T	
37	Sterling FBT152	2R	8 1/2	4580	174	204	30400	9500	B9.00/20	DB9.00/20	Wau 6-110	6-4x4 1/2	Ow 312B	U 5 No	Ow 312B	2F	R 7.8	55.5	10x3x 1/4	L	
38	FDT152	2R	8 1/2	4705	174	204	30400	9700	B9.00/20	DB9.00/20	Wau 6-110	6-4x4 1/2	Ow 312B	U 5 No	Ow 312B	2F	R 9.0	52.7	10x3x 1/4	L	
39	FDS180	4R	8-10	8605	158	Op	36000	12850	P40x8	DP40x8	Wau AB	6-4 1/2 x 5 1/2	Ow 312B	U 4 A 3	Tim 310	2F	R 9.1	113.	15x3x 1/4	L	
40	FDS200	4R	10-12	9130	159	Op	40000	13550	P40x8	DP40x8	Wau RB	6-5x5 1/2	Ow 312B	U 4 A 3	Tim 410	2F	R 9.1	113.	15x3x 1/4	L	
41	FCS210	4R	15-18	10175	Op	Op	42000	14750	P40x8	DP40x8	Wau RB	6-5x5 1/2	Ow 312B	U 4 A 3	Ow 312B	2F	R 9.1	113.	15x3x 1/4	L	
42	FDT200	2R	12-12 1/2	7670	178	208	40000	12050	P40x8	DP40x8	Wau 6-125	6-4 1/2 x 5 1/2	Ow 312B	U 4 Op	Ow 312B	2F	R 8.85	58.8	12x3x 1/4	L	
43	FDT250	2R	16-16 1/2	8855	186	216	50000	13550	P42x9	DP42x9	Wau RB	6-5x5 1/2	Ow 312B	U 4 Op	Ow 312B	2F	R 8.85	55.5	15x3x 1/4	L	
44	FCT180	2R	10-10 1/2	7265	178	208	36000	11200	P36x8	DP36x8	Wau SRL	6-4 1/2 x 5 1/2	Ow 312B	U 4 Op	Ow 312B	2F	R 8.2	54.5	12x3x 1/4	L	
45	FCT200	2R	12-12 1/2	7685	178	208	40000	11800	P40x8	DP40x8	Wau 6-125	6-4 1/2 x 5 1/2	Ow 312B	U 4 Op	Ow 312B	2F	R 9.3	61.8	12x3x 1/4	L	
46	Ward 440TC	15	11000	240	246	44000	14000	14000	B9.75/22	DB9.75/22	Cu. Die. HA	6-4 1/2 x 6	BL 735	A 5 No	Tim SBT420W	WF	R 6.42	40.4	14x3x 1/4	T	
47	LaFr. 440TR	15	9350	240	246	44000	13700	13700	B9.75/22	DB9.75/22	Wau RB	6-5x5 1/2	BL 735	A 5 No	Tim SBT420W	WF	R 6.42	40.4	14x3x 1/4	T	
48	340TM	7 1/2	4700	304	230	28000	9200	9200	B8.25/20	DB8.25/20	Wau MK	6-4 1/2 x 4 1/2	BL 5352	U 5 No	Tim SBT251H	SF	T Opt Opt	12x3x 1/4	T	48381.4	
49	400T	12	7100	205	241	40000	13000	13000	B9.75/20	DB9.75/20	Wau 6-125	6-4 1/2 x 5 1/2	BL 5352	U 5 No	Tim SW320W	WF	R 8.5	65.5	14x3x 1/4	T	
50	Wht. 630SW251	4R	5-6	193	205	193	10000	10000	B8.25/20	DB8.25/20	Ow A7	6-4 1/2 x 5 1/2	Ow 4B	U 4 No	Tim SW251	WF	R 6.75	44.2	8x3x 1/4	C	
51	642SW320	4R	7-9	198	210	198	12670	12670	B9.00/20	DB9.00/20	Ow 1AB	6-4 1/2 x 5 1/2	Ow 7B	U 4 No	Tim SW310W	WF	R 8.5	55.6	8x3x 1/4	C	
52	643SW420	4R	9-11	198	215	198	14400	14400	P40x8	DP40x8	Ow 1AB	6-4 1/2 x 5 1/2	Ow 7B	U 4 No	Tim SW410W	WF	R 10.2	69.1	18x3x 1/4	C	

Hercules Develops Two Small Fours for Commercial Use

HERCULES Motors Corp. of Canton, Ohio, has added the following fours to its present line of heavy duty four- and six-cylinder engines:

Model ZXA, 2 1/2 in. bore, 3 in. stroke, 58.8 displacement; Model ZXB, 2 5/8 in. bore, 3 in. stroke, 64.9 displacement.

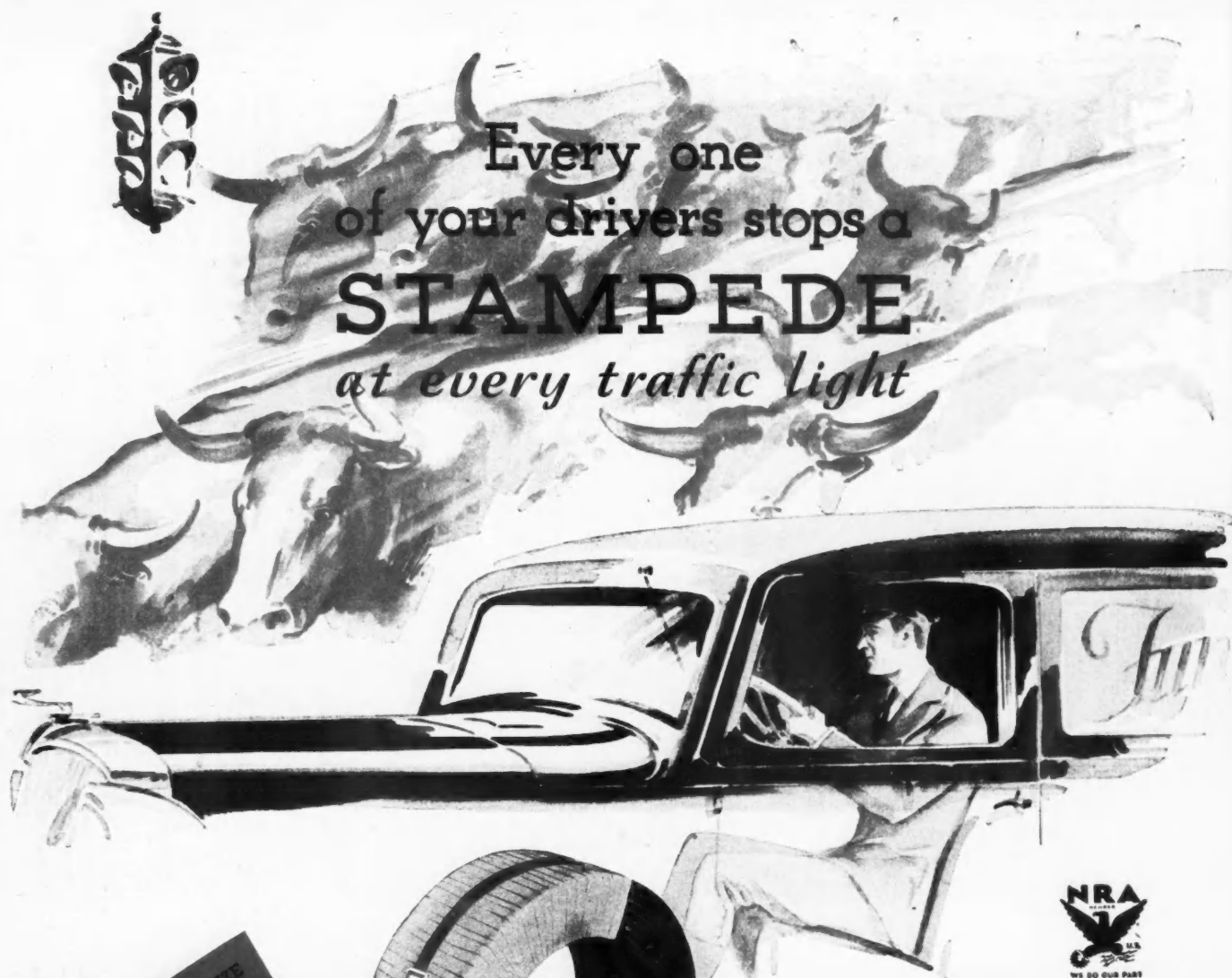
These two L-head models are identical in general design and the majority of the parts are interchangeable; the only difference being in the bore sizes and the parts affected thereby. The maximum torque of the ZXA is 37 lbs. developed at from 1500 to 2400 r.p.m. and on the ZXB is 40 lbs. developed from 1500 to 2400 r.p.m. Both models peak at 4000 r.p.m., the ZXA developing 22 h.p., the ZXB 24 1/2 h. p. at this speed.

To meet present operating conditions calling for sustained high speed, special consideration has been given to valve cooling. Standard practice includes thermo-syphon cooling, but water pumps

The small fours added to the Hercules line are designed to furnish 37 and 40-ft. lbs. torque

are available if specified. Either down-draft or up-draft manifolds are optional.

Type	Line Number	ENGINE DETAILS										SYST. FUEL	ELEC-TRICAL	FRONT AXLE	BRAKES		BODY MOUNT-ING DATA		SPRINGS															
		Piston Displacement	Compression Ratio	Torque lb. ft.	N.A.C.C. Rated H.P.	Max. Brake H.P. at R.P.M. Given	Valve Arrangement	Camshaft Drive	MAIN BEARINGS		Governor Make				Carburetors Make	Fuel Feed	Ignition System Make	Generator, Starter Make	Clutch Type and Make	Radiator Make	Universals Make	Steering Gear Make	SERVICE		Cab to Rear of Frame	Cab to Rear Axle	Width of Frame	Front	Rear	Auxiliary Type				
									Piston Material	Number and Diameter		Length	Make, Location Type, Operation	Lining Area									Drum Material	Hand Location, Type										
TL	1	282	4.7	176	33.7	73-2700	L	G	A	7-2 1/4	10 1/2	FP	No	Zen	M	DR	DR	P.B.L	Pe	Spi	Tim 31000H	Ros	L61HV	536a	TX	168	102	31 1/2	38x2 1/2	52x4	N			
TL	2	361	4.4	235	40.8	83-2400	L	L	A	7-3	11 1/2	FP	No	Zen	M	DR	DR	P.B.L	Pe	Spi	Tim 33000H	Ros	L61HV	536a	FD	168	102	31 1/2	38x2 1/2	52x4	N			
TL	3	393	4.9	260	42.1	103-2600	L	L	A	7-3	11 1/2	FP	No	Zen	M	DR	DR	P.B.L	Pe	Spi	Tim 33000H	Ros	L61HV	654a	FD	168	102	31 1/2	38x2 1/2	52x4	N			
TT	4	453	4.7	280	30.0	48.6	98	2200	L	L	G	A	7-3	14	CC	Ha	Zen	M	DR	DR	P.B.L	Pe	Spi	Tim 35000N	Ros	W4rA	815a	FD	192	120	33 1/2	42x3	56x4	N
TT	5	501	4.9	330	5.6	110-2200	L	L	G	A	7-3	12 1/2	CC	Ha	Zen	M	DR	DR	P.B.L	Pe	Spi	Tim 36020N	Ros	W4rA	815a	FD	192	120	33 1/2	42x3	56x4	N		
CC	6	672	4.2	420	57	125-1800	H	H	C	A	7-3 1/4	13 1/2	FP	No	Cu	No	DR	DR	P.B.L	Pe	Spi	Tim 36020N	Ros	W4rA	815a	FD	192	120	33 1/2	42x3	56x4	N		
CC	7	468	4.4	322	3.3	125-2400	H	H	C	A	7-3 1/4	10 1/2	FP	No	Bu	No	DR	DR	P.B.L	Pe	Spi	Tim 36020N	Ros	W4rA	815a	FD	192	120	33 1/2	42x3	56x4	N		
CC	8	638	4.3	410	54.1	126-1850	H	H	C	A	7-3 1/4	10 1/2	FP	No	Bu	No	DR	DR	P.B.L	Pe	Spi	Tim 36020N	Ros	W4rA	815a	FD	192	120	33 1/2	42x3	56x4	N		
CC	9	707	4.4	506	60.0	170-2000	H	H	C	A	7-3 1/4	11 1/2	FP	No	Bu	No	DR	DR	P.B.L	Pe	Spi	Tim 36020N	Ros	W4rA	815a	FD	192	120	33 1/2	42x3	56x4	N		
CC	10	707	4.4	506	60.0	170-2000	H	H	C	A	7-3 1/4	11 1/2	FP	No	Bu	No	DR	DR	P.B.L	Pe	Spi	Tim 36020N	Ros	W4rA	815a	FD	192	120	33 1/2	42x3	56x4	N		
CC	11	263	4.4	164	31.5	70-3000	L	L	G	C	7-2 1/2	10 1/2	PC	No	Str	M	DR	DR	P.B.L	Ow	Spi	Tim 30000H	Ros	L61HV	412G	TD	168	104	34	38x2 1/2	47x3	N		
CC	12	339	4.2	212	38	90-2700	L	L	G	C	7-2 1/2	13 1/2	PC	No	No	Str	M	DR	DR	D.B.L	Ow	Spi	Tim 33020H	Ros	L61HV	559G	TD	170	108	34	38x2 1/2	56x4	N	
CC	13	427	4.2	270	45.9	118-2500	H	H	C	C	7-2 1/2	13 1/2	PC	No	No	Str	M	DR	DR	D.B.L	Ow	Spi	Tim 33020H	Ros	L61HV	654G	TD	180	118	34	38x2 1/2	56x4	N	
CC	14	754	5.1	510	76.7	240-2900	L	L	G	C	7-3 1/4	10 1/2	PC	No	No	Str	M	DR	DR	dp.Lo	Ow	Blo	Tim 27450W	Ros	W61A	782D	CD	111	216	34	44x3	58x4	N	
CC	15	420	5.2	300	44.4	130-2800	L	L	G	C	5-2 1/2	12 1/2	PC	Ha	Str	M	DR	DR	D.Fu	Ch	Spi	Tim 35000H	Ros	W61A	613a	CD	162	108	34	39x2 1/2	46x3 1/2	N		
CC	16	420	5.2	300	44.4	130-2800	L	L	G	C	5-2 1/2	12 1/2	PC	Ha	Str	M	DR	DR	D.Fu	Ch	Spi	Tim 35000H	Ros	W61A	711a	CD	162	108	34	39x2 1/2	46x3 1/2	N		
CC	17	462	4.5	300	45.9	98-2000	L	L	G	A	7-3	13 1/2	PC	Wa	Str	M	My	DR	D.B.L	Ch	Spi	Tim 26045W	Ros	W61A	966a	CD	162	108	34	48x3 1/2	53x4	N		
CC	18	462	4.5	300	45.9	98-2000	L	L	G	A	7-3	13 1/2	PC	Wa	Str	M	My	DR	D.B.L	Ch	Spi	Tim 26045W	Ros	W61A	966a	CD	162	108	34	48x3 1/2	53x4	N		
CC	19	549	4.5	332	48.6	100-2000	L	L	G	A	4-3 1/4	11 1/2	PC	Wa	Str	M	My	LN	D.B.L	Ch	Spi	Tim 27045W	Ros	W61A	966a	CD	162	108	34	48x3 1/2	53x4	N		
CC	20	677	4.6	460	60.0	127-2000	L	L	G	A	4-3 1/4	11 1/2	PC	Wa	Str	M	My	LN	D.B.L	Ch	Spi	Tim 27045W	Ros	W61A	966a	CD	162	108	34	48x3 1/2	53x4	N		
CC	21	672	4.4	420	57	125-1800	H	H	C	A	7-3 1/4	10 1/2	FP	No	Bu	No	DR	DR	dp.BL	Ch	Spi	Own BX	Own	O61A	974a	FX	192	109	38 1/2	54 1/2 x 3	48x3 1/2	N		
CC	22	468	5.2	310	33	117-2400	L	L	G	C	4-3 1/2	10 1/2	FP	Ow	Str	M	RB	NE	P.Ow	Ow	Spi	Own BX	Own	O61A	902a	FX	192	111	37 1/2	50x3 1/2	48x3 1/2	N		
CC	23	611	5.7	398	54.2	128-2300	L	L	G	C	4-3 1/2	10 1/2	FP	Ow	Str	M	RB	NE	P.Ow	Ow	Spi	Own BX	Own	O61A	902a	FX	192	111	37 1/2	50x3 1/2	48x3 1/2	N		
CC	24	611	5.7	398	54.2	128-2300	L	L	G	C	4-3 1/2	10 1/2	FP	Ow	Str	M	RB	NE	P.Ow	Ow	Spi	Own BX	Own	O61A	902a	FX	192	111	37 1/2	50x3 1/2	48x3 1/2	N		
CC	25	611	5.7	398	54.2	128-2300	L	L	G	C	4-3 1/2	10 1/2	FP	Ow	Str	M	RB	NE	P.Ow	Ow	Spi	Own BX	Own	O61A	902a	FX	192	111	37 1/2	50x3 1/2	48x3 1/2	N		
CC	26	706	4.4	427	60.0	138-1900	L	L	G	C	4-3 1/4	11 1/2	PS	Ow	Str	M	RB	NE	P.Ow	Ow	Spi	Own BX	Own	O61A	930a	FX	180	109	38 1/2	48x3 1/2	52x4	N		
CC	27	706	4.4	427	60.0	138-1900	L	L	G	C	4-3 1/4	11 1/2	PS	Ow	Str	M	RB	NE	P.Ow	Ow	Spi	Own BX	Own	O61A	930a	FX	180	109	38 1/2	48x3 1/2	52x4	N		
CC	28	529	4.9	350	51.3	114-2200	L	L	G	C	7-3	14 1/2	PC	Ow	Str	M	RB	NE	P.Ow	Ow	Spi	Own BX	Own	O61A	1044a	FX	180	109	38 1/2	48x3 1/2	52x4	N		
CC	29	707	4.5	460	60.0	148-2000	L	L	G	C	7-3 1/4	17 1/2	PC	Ha	Zen	M	DR	DR	dp.BL	Lo	Cle	Tim 26450TW	Ros	W4/61A	760a	TD	152	102	34	44x3	46x4	N		
CC	30	779	5.5	508	66.2	164-2000	L	L	G	A	7-3 1/4	17 1/2	PC	Ha	Zen	M	DR	DR	dp.BL	Lo	Cle	Tim 26450TW	Ros	W4/61A	760a	TD	152	102	34	44x3	46x4	N		
CC	31	282	5.0	176	33.8	73-2800	L	L	G	C	7-2 1/2	10 1/2	PC	No	Zen	M	AL	AL	P.B.L	Lo	Cle	Tim 30000H	Ros	L61H	412a	TD	168	71	34	40x2 1/2	44x3	N		
CC	32	282	5.0	176	33.8	73-2800	L	L	G	C	7-2 1/2	10 1/2	PC	No	Zen	M	AL	AL	P.B.L	Lo	Cle	Tim 30000H	Ros	L61H	570a	TD	168	67	34	40x2 1/2	44x3	N		
CC	33	383	4.4	262	43.3	92-2400	L	L	G	C	7-2 1/2	13 1/2	PC	No	Zen	M	AL	AL	P.B.L	Lo	Cle	Tim 31000H	Ros	L61H	570a	TD	192	101	34	40x2 1/2	52x4	N		
CC	34	383	4.4	262	43.3	92-2400	L	L	G	C	7-2 1/2	13 1/2	PC	No	Zen	M	AL	AL	P.B.L	Lo	Cle	Tim 31000H	Ros	L61H	661a	TD	192	101	34	41 1/2 x 2 1/2	48x3 1/2	N		
CC	35	501	4.9	330	45.9	110-2200	L	L	G	C	7-3	12 1/2	PC	No	Zen	M	AL	AL	P.B.L	Lo	Cle	Tim 36020H	Ros	W61A	966a	TD	Opt	Opt	38	44x3	48x3 1/2	N		
CC	36	611	4.5	384	54.1	127-2000	F	G	A	7-2 1/4	12 1/2	CC	Ha	Zen	M	DR	DR	D.Ow	Mo	Spi	Tim 35000N	Ros	L61HV	596a	CX	192	91	34	42x2 1/2	57x4	N			
CC	37	358	5.0	254	38.5	110-2800	F	G	A	7-2 1/4	12 1/2	CC	Ha	Zen	M	DR	DR	D.Ow	Mo	Spi	Tim 35000N	Ros	L61HV	596a	CX	192	91	34	42x2 1/2	57x4	N			
CC	38	358	5.0	254	38.5	110-2800	F	G	A	7-2 1/4	12 1/2	CC	Ha	Zen	M	DR	DR	D.Ow	Mo	Spi	Tim 35000N	Ros	L61HV	596a	CX	192	91	34	42x2 1/2	57x4	N			
CC	39	549	4.5	330	45.9	99-2000	L	L	G	C	4-3 1/4	11 1/2	CC	Ha	Zen	M	DR	DR	D.Ow	Mo	Spi	Tim 26450N	Ros	W41A	576a	CX	Opt	88	34	48x3	58x4	N		
CC	40	677	4.4	440	60.0	125-2000	L	L	G	A	7-3 1/4	11 1/2	CC	Ha	Zen	M	DR	DR	D.Ow	Mo	Spi	Tim 27450N	Ros	W41A	792a	CX	Opt	89	34	48x3	58x4	N		
CC	41	677	4.4	440	60.0	125-2000	L	L	G	A	7-3 1/4	11 1/2	CC	Ha	Zen	M	DR	DR	D.Ow	Mo	Spi	Tim 27450N	Ros	W41A	792a	CD	Opt	93	34	48x3	60x3 1/2	N		
CC	42	462	5.5	324	45.9	125-2400	F	G	A	7-3	13 1/2	CC	Ha	Zen	M	DR	DR	D.Ow	Mo	Spi	Tim 26450TW	Ros	O41A	792a	CX	192	93	34	48x3	58x4	N			
CC	43	677	4.4	440	60.0	125-2000	L	L	G	A	7-3 1/4	11 1/2	CC	Ha	Zen	M	DR																	



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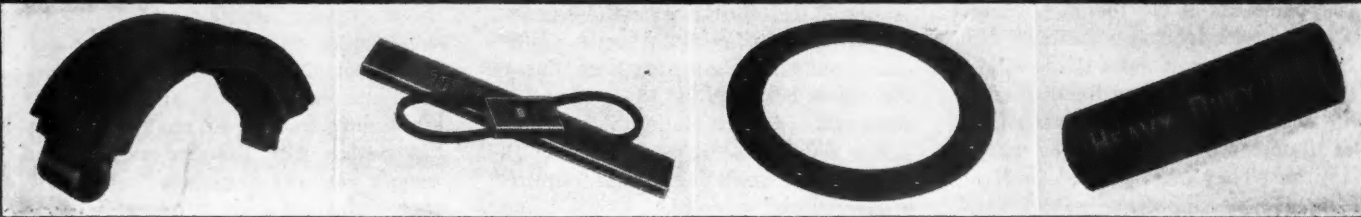
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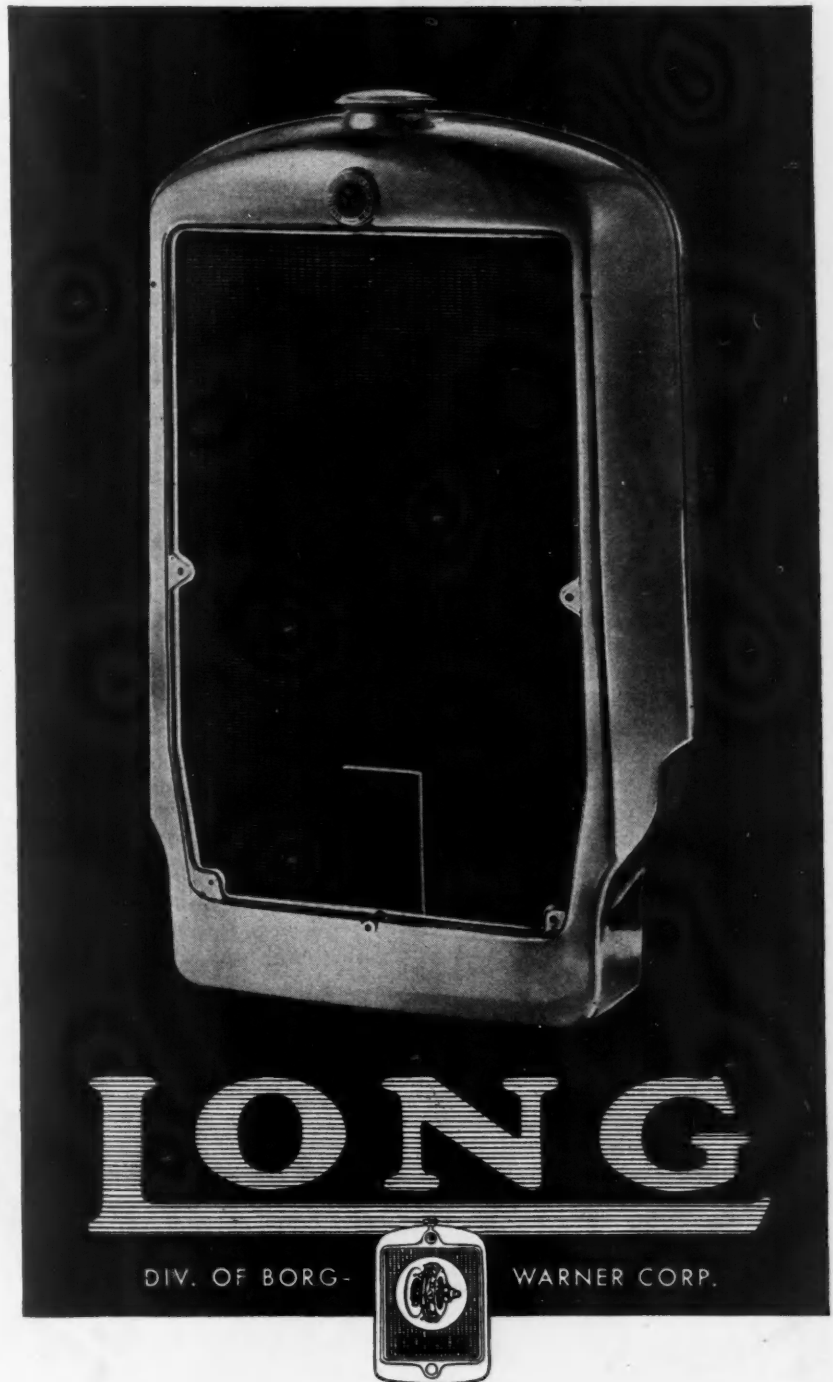


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CHICAGO, ILL. MFG. CO.

Ford Bolsters Bid

(CONTINUED FROM PAGE 36)

tem which has proven so highly successful in giving improved fuel economy for the passenger car.

The truck engine also utilizes new cast-iron cylinder heads with which horsepower output is obtained equal to that of the former powerplant equipped with aluminum heads and single carburetor. The engine develops in excess of 80 hp., approximately 5 hp. more than formerly. A new combustion chamber shape for the cylinder heads gives a greater power output from a lower compression pressure. The compression ratio of the new iron heads is 5.32 to 1.

To assure long bearing life the connecting rod insert bearings are of a new high-load bronze, capable of withstanding high oil temperatures. The crankshaft is of the new cast alloy-steel.

Among other changes in the truck for 1934 are a new seat cushion with "mattress" type springs.

The front bumper has been lengthened and will be chrome-plated.

News

(CONTINUED FROM PAGE 38)

troit Axle Co. executive, has been named vice-president of the Thornton Tandem Co., President Richard F. Barnum reports. Mr. Gilbert joins Thornton in connection with the expansion of production and marketing of its dual-ratio positive four-wheel drive unit for trucks.

Raymond Selling Fruehaufs

G. W. Chamberlin, vice-president in charge of sales for the Fruehauf Trailer Co., announces appointment of Henry W. Raymond to the sales staff, with headquarters at the factory. Mr. Raymond formerly was general sales manager of the Lapeer Trailer Corp.

Evans and Strawbridge Join

Boyd V. Evans former chief engineer of Detroit Motor Bus Co., and W. L. Strawbridge, have organized the Transportation Materials Co., with offices at 1916 Fairmount Ave., Philadelphia, Pa., and specialize in a complete line of cleaners and equipment for the transportation field, including trucks.

Fisken and Timper Advance

E. W. Timper has been made manager of sales promotion for Chevrolet Motor Co., succeeding C. P. Fisken, now advertising manager.

Furlong Directs Sales

Donald G. Furlong has been appointed director of sales for the Thornton Tandem Co. Formerly Furlong was with the Ford Motor Co. in charge of commercial car and truck sales for the Dearborn branch. He is now inaugurating a distribution policy.

Henry With Gulf Fleet

James S. Henry, automotive engineer, is now with the Gulf Refining Co., in the Houston, Tex., office studying fleet operation. He was formerly with the Humble Oil & Refining Co., Houston.

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Lansing, Michigan

Julian Deane Resigns

Julian (Larry) Deane, sales promotion manager of United American Bosch Corp., has resigned to become associated with McCann-Erickson, Inc., New York advertising agency.

Franklin F. Chandler

Franklin F. Chandler, 57, vice-president of the Ross Gear & Tool Company, died at his home in Lafayette, Ind., following a brief illness.

Henry Bert Edwards

Henry Bert Edwards, 75, died of pneumonia in Detroit. After serving as branch manager of the Studebaker truck division in Chicago and New York he returned to Detroit as wholesale manager.

John E. Peters

John E. Peters, 60, died in Detroit after a year's illness. For 20 years he had been salesman and then sales manager for Federal Motor Truck Co.

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